

PRELIMINARY DESIGN OF A FREE VORTEX  
AXIAL FLOW TURBINE

Victor Endo

THEY ARE THE ONLY  
ONE FURNISHING THEM.

PRELIMINARY DESIGN OF A FREE VORTEX AXIAL FLOW TURBINE

by

VICTOR ENDO

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Lieutenant, United States Navy

B.S.E.E. University of New Mexico (1971)

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VICTOR ENDO

Submitted to the Department of Ocean Engineering, on May 12, 1977 in partial fulfillment of the requirements for the degrees of Ocean Engineer and Master of Science in Naval Architecture and Marine Engineering.

## ABSTRACT

The thesis covers the mechanical, thermodynamics, and aerodynamics of designing a free vortex axial flow turbine with the aid of a digital computer. It uses the Improved Ainley-Mathieson Performance Estimation Method to determine the aerodynamic pressure losses so off-design and design point calculations can be conducted to produce performance curves. It also gives physical and thermodynamic properties of the turbine.

Thesis Supervisor: Alexander Douglas Carmichael

Title: Professor of Power Engineering

THESE ARE THE  
REMARKS OF THE

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# TABLE OF CONTENTS

4

	PAGE
TITLE PAGE	1
ABSTRACT	2
ACKNOWLEDGEMENTS	3
TABLE OF CONTENTS	4
LIST OF FIGURES	5
INTRODUCTION	7
CHAPTER 1 CYCLE CALCULATIONS	9
CHAPTER 2 MECHANICAL DESIGN OF AN AXIAL FLOW TURBINE	19
CHAPTER 3 PERFORMANCE ESTIMATION	42
CHAPTER 4 OPERATION OF COMPUTER PROGRAM	57
REFERENCES	62
APPENDIX I SYMBOLS AND NOTATION USED IN THIS TEXT	64
APPENDIX II USER'S GUIDE TO COMPUTER PROGRAM WITH COMPARISON DATA AND SAMPLE INPUT AND OUTPUT OF A TURBINE DESIGN	68
APPENDIX III COMPUTER PROGRAM (1) and (2)	94



## LIST OF FIGURES

## PAGE

1-1.	Axial Flow Turbine Stage . . . . .	10
1-2.	Velocity Diagram . . . . .	10
1-3.	T-S Diagram of a Reaction Stage. . . . .	10
2-1.	Conventional Blade Profile . . . . .	20
2-2.	Variation in Profile Loss with Incidence Angle . . . . .	20
2-3.	Relation Between Gas and Blade Angles. . . . .	20
2-4.	Angle variation with Mach Number . . . . .	20
2-5.	Rotating Turbine Blade . . . . .	21
2-6.	Variation of Taper Factor with Blade Profile Area. . . . .	22
2-7.	Optimum Spacing of Turbine Blades. . . . .	24
2-8.	Non Dimensional Area of Turbine Blades verses Blade Turning Angle. . . . .	25
2-9.	Blade Section Modulus. . . . .	25
2-10.	Bending Moments Acting on a Turbine Blade. . . . .	26
2-11.	Effect of Taper on Blade Natural Bending Frequency . . . . .	27
2-12.	Variation of Rotation Coefficient with Blade Height Ratio. . . . .	27
2-13.	Cambell Diagram. . . . .	29
2-14.	Goodman Diagram. . . . .	30
2-15.	Linearized Goodman Diagram . . . . .	30
2-16.	Amplification Factor . . . . .	31
2-17.	Firtree Blade Attachments. . . . .	34





2-18.	Turbine Disc Stress Analysis . . . . .	36
2-19.	Turbine Disc Load Distribution . . . . .	39
2-20.	Axial Spacing Between Blades . . . . .	41
2-21.	Stagger Angle of Turbine Blades. . . . .	41
3-1.	Analysis of Losses in Flow Though a Row of Turbine Blades . . . . .	43
3-2.	Gas and Blade Angles Used in Subroutine PLOSS. . . . .	43
3-3.	Profile Loss Coefficients for Conventional Section Blades at Zero Incidence . . . . .	44
3-4.	Off-Design Performance of Cascades of Turbine Blades .	47
3-5.	Variation of Relative Profile Loss with Relative Incidence. . . . .	48
3-6.	Effects of Trailing Edge Thickness on Blade Loss Coefficients . . . . .	49
3-7.	Performance Curves for Turbines. . . . .	52
4-1.	Simplified Block Diagram of Computer Program . . . . .	58
4-2.	Flow Diagram of Subroutines FIG and BK which Interpolate curves by a 4pt Lagrange Polynomials . . .	60
4-3.	Flow Diagram Showing How Turbine Flow Areas are Determined . . . . .	61
II-1.	Comparison of Turbine Wheels . . . . .	81
II-2.	Comparison of Total and Static Efficiencies verses Tip Clearance for Computer Designed Turbine and Airesearch Designed Turbine. . . . .	82
II-3.	Performance Curves from Data Calculated from Computer Program . . . . .	34



The purpose of this thesis is to develop a computer program which can be used in the Preliminary Design of a Free Vortex Axial Flow turbine. The design plan consists of three main parts. The first will be cycle calculations to determine stage characteristics at each station such as angles, density, temperature, flow areas, etc. The properties from this phase will be used in parts 2 and 3. The second part is to determine mechanical characteristics such as size and weights of turbine parts and stress and vibrational characteristics. The final phase is to determine turbine characteristics at design and off design parameters such as speed, pressure ratio, etc. This last phase is optional since compressible flow tables and gas properties are required and many turbine designers may be more interested in the size and weights than off design characteristics in the preliminary design phase. These three parts are covered in chapters one through three, chapter four will cover the operation of the computer program developed in this text. Appendix one is a list of symbols used in the text, appendix two is a user's guide to the computer program and has an example turbine design which is compared with a design proposed by reference 20. Appendix three is the computer program.

The computer program is set up so it has many options of input variables so that as the designer becomes more knowledgeable



in his design study he can specify more characteristics and have more control over the design output. There are checks in the program to limit poor design properties such as excessive swirl in output, high tip to hub ratios, negative reactions, etc. The computer program can be used to design a constant hub, mean, or tip turbine. The design does not consider the effects of cooling in turbines, since time was not available to incorporate this into the computer program. This limitation mostly affects aircraft gas turbine and aircraft derivative marine gas turbines since these engines operate at high turbine inlet temperatures and cooling is required to allow for a high mean time between failures.

There has been no effort made to prove the relations used in the design calculations. These can be found in basic physics, thermodynamics, and fluid mechanics books.

The notation as to signs of angles in the velocity diagrams will be the same as used in chapter 4 of Gas Turbine Handbook which is reference (3). An illustration is enclosed in chapter 1 of the sign convention and notation used in this text. A deviation from this notation will be used in section (3.2).



1.1 The basic design of an axial flow turbine starts with the cycle calculations to determine the thermodynamic properties throughout the turbine. The basis of these calculations are from known inlet conditions and assumed design parameters. The computer program is set up so that a minimum number of these parameters must be specified and leaves the option to the designer to more fully specify the design parameters, if he so desires. This chapter intends to show how these basic calculations are carried out and the methods employed to determine the physical properties such as flow area from these design parameters. Figures (1-1), (1-2), and (1-3) show the sign convention and notation that will be used in this chapter. The following are some of the design parameters needed for the cycle calculations; inlet temperature ( $^{\circ}\text{R}$ )  $T_{0i}$ , turbine inlet pressure (psi)  $P_{0i}$ , omega (rpm)  $\Omega$ , specific heat (btu/lbm  $^{\circ}\text{R}$ )  $C_p$ , mass flow (lbm./sec)  $\dot{m}$ , ratio of specific heat  $\gamma$ , tip radius (ft)  $r_t$ , tip to hub ratio ( $r_t/r_h$ ), desired efficiency  $\eta_t$ , loading coefficient  $\psi$ , etc. Appendix two (2) will specify which parameters must be known and which ones are optional.

1.2 Determination of overall turbine parameters from inlet conditions and assumed design parameters.

$$(a) \quad (1.2-1)$$

$$\Delta h_{0t} = \frac{\text{Power (hp)} \cdot 550}{\dot{m} \eta_t} \quad (1.2-2)$$

$$T_{0e} = T_{0i} - \frac{\Delta h_{0t}}{C_p} (^{\circ}\text{R})$$





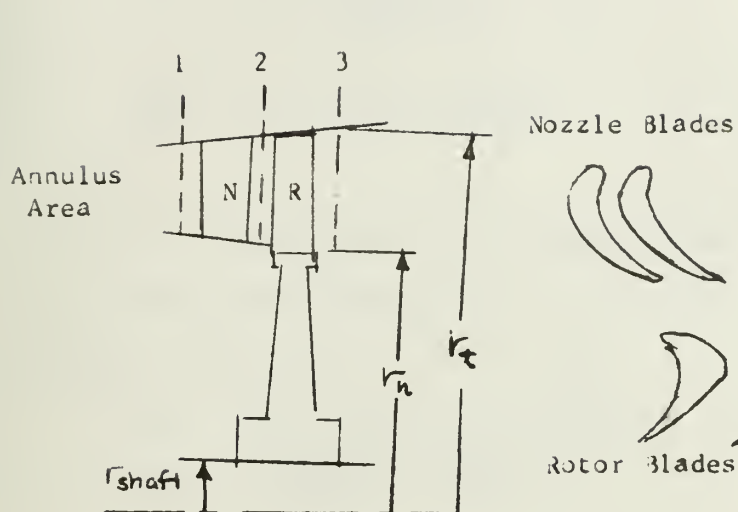


Fig. 1-1 Axial Flow Turbine Stage

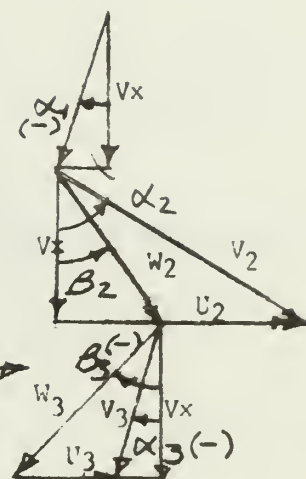


Fig. 1-2 Velocity Diagram

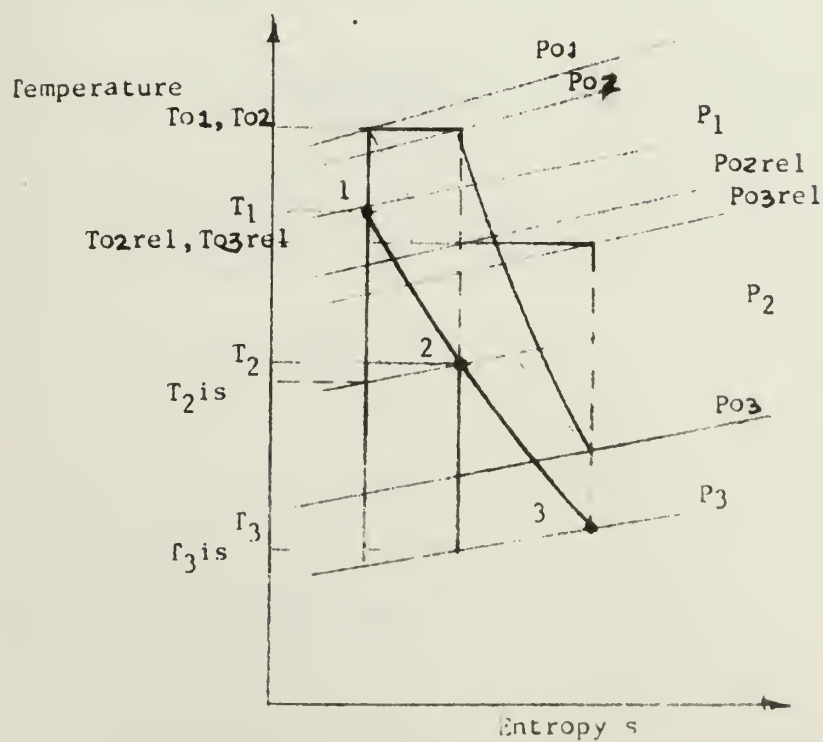


Fig. 1-3 T-S diagram for a Reaction stage



$$\eta_p = \frac{\ln(T_{oe}/T_{oi})}{\ln \left[ \frac{(T_{oe}/T_{oi}-1)}{\eta_t} + 1 \right]} \quad (1.2-3)$$

$$P_{oe} = P_{oi} (T_{oe}/T_{oi})^{\frac{\gamma}{\eta_p(\gamma-1)}} \quad (1.2-4)$$

(b) If the pressure ratio were specified instead of output power the following calculations will be performed.

Polytropic Efficiency

$$\eta_p = \frac{\ln(1 - \eta_t(1 - (P_{oe}/P_{oi})^{\frac{\gamma-1}{\gamma}}))}{\ln[(P_{oe}/P_{oi})^{\frac{\gamma}{\gamma-1}}]} \quad (1.2-5)$$

$$T_{oe} = T_{oi} (P_{oe}/P_{oi})^{\frac{\gamma}{\eta_p(\gamma-1)}} \quad (1.2-6)$$

$$\Delta h_o = C_p(T_{oi} - T_{oe}) \quad (1.2-7)$$

$$\text{Power out} = \frac{\dot{m} \Delta h_o \eta_t J}{550} \quad (1.2-8)$$

(c)

$$\omega = \frac{\Omega (2\pi)}{60} \quad (\text{rad/sec}) \quad (1.2-9)$$

$$r_h = r_t / (r_t / r_h) \quad (1.2-10)$$

$$r_m = (r_h + r_t) / 2 \quad (1.2-11)$$

$$U_h = r_h \omega (\text{ft/sec}) \quad (1.2-12)$$



$$\Delta h_{o\text{stage}} = \frac{U_h V_x (\tan \alpha_2 - \tan \alpha_3)}{g_0 J} \quad (1.2-13)$$

$$\psi = \frac{g_0 J \Delta h_{o\text{stage}}}{U_h^2} \quad (1.2-14)$$

The value of  $\psi$  is taken at the hub of the last stage because this is the most critical in the design of a turbine in reference to loading. Normally the value of  $\psi$  is from (1.5 - 2.9). The lower value is for lightly loaded stages while the higher is for highly loaded stages. To have no exit swirl  $\psi$  must be less than or equal to 2.0.

$$\psi \leq 2.0 \Rightarrow (\text{no exit swirl})$$

$$\text{for } \alpha_h = 0 \quad R = 1 - \psi/2 \quad (1.2-15)$$

where

$$R = \frac{h_2 - h_3}{h_{01} - h_{03}} \quad \begin{array}{l} \text{ratio of change in static enthalpy} \\ \text{across the rotor to change in} \\ \text{stagnation enthalpy across the stage.} \end{array} \quad (1.2-15a)$$

With the value of  $\psi$  specified, the number of stages required to develop the power out desired can be determined.

$$\Delta h_{o\text{stage}} = \frac{\psi U_h^2}{g_0 J} \quad (1.2-16)$$

Determining number of stages

$$N = \text{integer } \frac{\Delta h_{ot}}{\Delta h_{o\text{stage}}}$$

If  $N$  were specified instead of  $\psi$ , the change in enthalpy per stage would be determined.

$$\Delta h_{o\text{stage}} = \frac{\Delta h_{ot}}{N}$$



This program uses a constant change in enthalpy for each stage.

If  $\psi \leq 2.0$  and  $\alpha_h = 0$  the radius of the hub can be determined from equation (1.2-17).

$$r_{he} = \left[ \frac{g_0 J \Delta h_{o_{stage}}}{\omega^2} \right]^{\frac{1}{2}} \quad (1.2-17)$$

Another parameter which will be used is  $\phi \equiv V_x / U_h$  (flow coefficient). Where  $V_x$  is the axial velocity, which is constant in a free vortex turbine. This parameter ranges from 0.4 - 1.3, the higher values are used for aircraft turbines which use high axial velocity for propulsion. In marine and industrial turbines a low value is desired because the energy of the exhaust gases are not recoverable in this form, unless it is used to drive another turbine.

Another parameter that may be specified is the critical Mach number at the exit.

$$M_{crit} = \frac{V_x}{\left[ \frac{2}{\gamma+1} g_0 R T_{oe} \right]^{\frac{1}{2}}} \quad (1.2-18)$$

where  $R$  is the gas constant

$$R \equiv C_p (\gamma - 1 / \gamma) J \quad (1.2-19)$$

The  $V_x$  is computed exactly

$$V_x = M_{crit} \left[ (2 / (\gamma + 1)) g_0 R T_{oe} \right]^{\frac{1}{2}} \quad (1.2-13a)$$





If the exit Mach number is specified the turbine exit area is varied until the required area is obtained to give the specified axial velocity  $V_x$ . If this number is not specified but the exit area is specified the value of  $V_x$  is varied until the required area is achieved.

From the continuity equation for gases

$$V_x = \frac{\dot{m}}{2\pi \int_{r_h}^{r_t} \rho(r) r dr} \quad (1.2-20)$$

where

$$\rho(r) = \frac{P(r)}{RT(r)} \quad (1.2-21)$$

$$T(r) = T_o - \frac{V(r)^2}{2C_p g_o J} \quad (1.2-22)$$

$$P(r) = P_o / (T_o / T(r))^{\frac{\gamma}{\gamma-1}} \quad (1.2-23)$$

$$V(r) = \frac{V_x}{\cos(\alpha(r))} \quad (1.2-24)$$

where  $r$  varies between the hub and tip. If the exit angle is not equal to zero

$$\alpha_{hub_e} = \tan^{-1} \left[ (1 - R_{he} - \psi/2) / \phi \right] \quad (1.2-25)$$

and for free vortex

$$rV_\theta = rV_x(\tan \alpha) = \text{constant} \quad (1.2-26)$$

then

$$\alpha(r_e) = \tan^{-1} \left[ \frac{r_{he} \tan(\alpha_{he})}{r_e} \right] \quad (1.2-27)$$

where  $r_e$  is taken at hub mean, and tip. This calculation is carried out in function program ANGLE. Thus the exit angle has been determined at the hub, mean, and tip radii.



The pressure, density, and temperature calculations are performed in subroutine TPl. The integral calculation is performed in subroutine DENR which uses Simpson's Rule to compute the axial velocity  $V_x$ . With the absolute angles  $\alpha(r_e)$  determined the relative angles  $\beta(r_e)$  can then be determined by equation (1.2-28). This is carried out in function program ANGLEB.

$$\beta(r_e) = \tan^{-1} \left[ \tan(\alpha(r_e)) - \frac{U(r_e)}{V_x} \right] \quad (1.2-28)$$

From equation (1.2-15a)

$$R(r_e) = 1 - \frac{\Delta h_{\text{stage } 30J}}{2(\omega r_e)^2} - \frac{V_x \tan(\alpha_e)}{\omega r_e} \quad (1.2-29)$$

which is computed in function program REACTI. With the exit conditions specified, the properties at station 1 and 2 can then be determined for each stage.

The following properties are now known at the exit of the turbine;  $V_x$ ,  $T_{0e}$ ,  $P_{0e}$ ,  $f(r_e)$ ,  $P(r_e)$ ,  $T(r_e)$ ,  $\alpha(r_e)$ ,  $\beta(r_e)$ ,  $R(r_e)$ , and  $r_e$ , where  $r_e = r_{\text{hub}}, r_{\text{mean}}, r_{\text{tip}}$ .

1.3 Determination of properties at station 1 for stage  $n$  where  $n = 1 \dots N$ .

$$T_{01} = T_{03n} + \frac{\Delta h_{\text{stage}}}{C_p} \quad (1.3-1)$$

$$P_{01n} = P_{03n} / (T_{03n} / T_{01n})^{\frac{\gamma}{\gamma-1}} \quad (1.3-2)$$

If this is the 1st stage  $\alpha_{1n} = 0.0$  where  $n = 1$

If not  $\alpha_{1n}(r_c) = \alpha_{3n}(r_c)$  where  $r_c$  is the radius which remains constant through out the turbine, ie(constant hub, tip, or mean).



$$\beta_{ln}(r_c) = \tan^{-1} \left[ \tan(\alpha(r_c)) - \frac{U(r_c)}{V_x} \right] \quad (1.3-3)^{16}$$

$$T_{ln}(r_c) = T_{o,ln} - \frac{V_{ln}^2(r_c)}{2g_o J C_p} \quad (1.3-4)$$

$$V_{ln}(r_c) = \frac{V_x}{\cos(\alpha_{ln}(r_c))} \quad (1.3-5)$$

$$P_{ln}(r_c) = P_{o,ln} / (T_{o,ln} / T_{ln})^{\frac{\gamma}{\gamma-1}} \quad (1.3-6)$$

$$\rho_{ln}(r_c) = \frac{P_{ln}(r_c)}{RT_{ln}(r_c)} \quad (1.3-7)$$

which is identical to equations in section for exit of the turbine.

A trial blade height (h) at station ln will be established.

$$h = \frac{\dot{m}}{2\pi V_x r_c \rho(r_c)} \quad (1.3-8)$$

Let  $\Delta h = h/10$ . Now depending on whether the turbine is constant hub, mean, or tip, radius are established at the other radii.

Constant hub;

$$r_{ln}(\text{hub}) = r_c \quad (1.3-9)$$

$$r_{ln}(\text{mean}) = r_c + h/2 \quad (1.3-10)$$

$$r_{ln}(\text{tip}) = r_c + h \quad (1.3-11)$$

Similarly for constant mean or constant tip.

$\alpha$  can then be determined at all radii for station l and temperature; pressure, and density can be calculated at all radii in the same manner as was carried out at the exit of the turbine. Then

$V_x$  prime can be calculated in the same manner as  $V_x$  was calculated.



$$V_x \text{ prime} = \frac{\dot{m}}{2\pi \int_{r_h}^{r_t} \rho(r) r dr} \quad (1.3-12)$$

When  $V_x \text{ prime} = V_x$  the proper radii has been calculated at station 1. The method employed continuously decreases delta h until convergence is accomplished. This is the same method which was used at the exit of the turbine and will be used in station 2 calculations. The properties of the gas is now determined as per exit of the turbine for station 1. If this isn't the first stage the properties at station 1 are the same as station 3 of the preceeding stage.

1.4 Determination of properties at station 2 for stage n where n = 1 . . . . N

$$T_{02n} = T_{01n} \quad (1.4-1)$$

$$\alpha_{2n}(r_c) = \tan^{-1} \left[ \frac{g_0 \Delta h_{os}}{V_x \omega r_c} + \tan(\alpha_{3n}(r_c)) \right] \quad (1.4-2)$$

$$V(r_c) = V_x / \cos(\alpha_{2n}(r_c)) \quad (1.4-3)$$

$$T_{2n} = T_{02n} - V(r_c)^2 / 2g_0 J C_p \quad (1.4-4)$$

$$P_{2n} \approx P_{1n} (T_{2n} / T_{1n})^{\frac{\gamma}{\gamma-1}} \quad (1.4-5)$$

which is an approximation from figure (1-3), where the actual pressure is determined from pressure loss data as will be covered in Chapter 3.

$$P_{02n} = P_{2n} (T_{02n} / T_{2n})^{\frac{\gamma}{\gamma-1}} \quad (1.4-6)$$

$$\rho_{2n} = \frac{P_{2n}(r)}{RT_{2n}(r)} \quad (1.4-7)$$





determine temperature, pressure, and density at station 2. The calculations are carried out in the same manner as station 1 calculations, to determine the gas properties and radii at station 2. The calculations are then carried out at station 1 of the next stage (ie;  $n = n-1$  until properties at all stations and positions have been computed for all stages.)

1.5 Having solved for gas properties at all stations and positions and having flow areas and gas angles the relative stagnation temperature, and the relative Mach numbers for stators and rotors can be calculated.

$$T_{02rel} = T_2 + \frac{w^2}{2goCpJ} \quad (1.5-1)$$

$$M_{nozzle\ inlet} = \left[ \left( \frac{T_{01n}}{T_{1n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-2)$$

$$M_{nozzle\ exit} = \left[ \left( \frac{T_{02n}}{T_{2n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-3)$$

$$M_{rotor\ inlet} = \left[ \left( \frac{T_{02nrel}}{T_{2n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-4)$$

$$M_{rotor\ exit} = \left[ \left( \frac{T_{03nrel}}{T_{3n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-5)$$

These calculations are carried out at each stage. The following parameters have been determined at all stations of the turbine; temperature, pressure, density, flow area, absolute and relative angles, Reaction, relative Mach numbers, and the axial velocity  $V_x$ . These properties will be used to determine the mechanical aspects of the Turbine design and be used in performance estimation.



## Mechanical Design of an Axial Flow Turbine

2.1 In this chapter the mechanics of the program and assumptions made to design nozzle and rotor blades, and the turbine disc will be discussed.

2.2 Having the gas properties and flow angles throughout the turbine the blade angles can be calculated for the turbine. The criteria used in establishing the blade angles is the loss data provided by figure (2-2) and (2-3) based upon Ainley Mathieson method of performance estimation. As can be seen for reaction blades in figure (2-2) the incidence angles can vary from -15 to + 15 without much of an increase in profile loss  $\gamma_p$ . In actual construction this large variation can allow for a decrease in twist in the blade from hub to tip normally required in free vortex blades. For the computer program, selection of blade angle, at the inlet of the blade is for zero incidence at design point.

Figure (2-3) shows how blade exit angle varies with gas angle there is also a correction for Mach number. For Mach numbers less than 0.5 the values given by figure (2-3) will be used, for values greater than Mach 1 the blade angles will be equal to gas angles while for values between Mach 0.5-1.0 a linear variation will be assumed as is shown in figure (2-4). The calculation of these blade angles are carried out in subroutine Blade, which also will be used to determine gas angles when blade angles are known for off-design calculations in chapter 3. There is an additional correction for blades



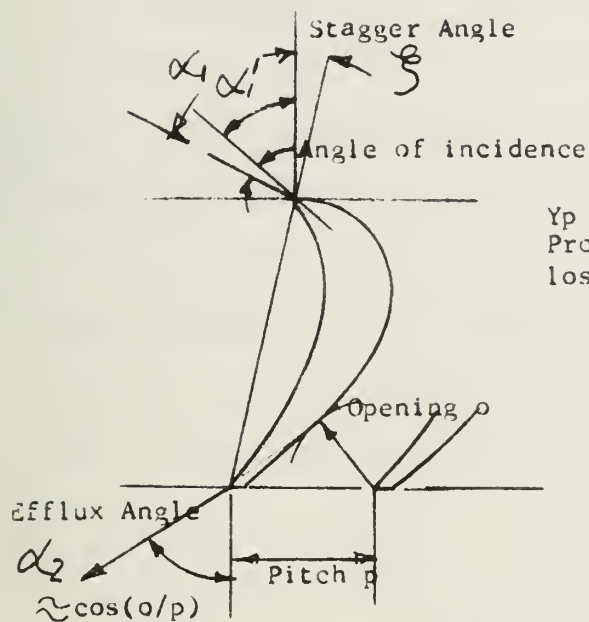


Figure 2-1. Conventional Blade Profile

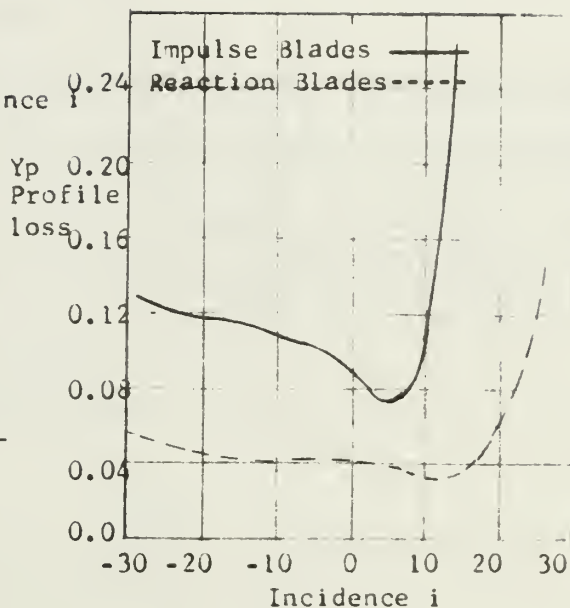


Figure 2-2. Variation in profile loss with incidence angle (From D.G.Ainley and G.C.R.Mathieson(1))

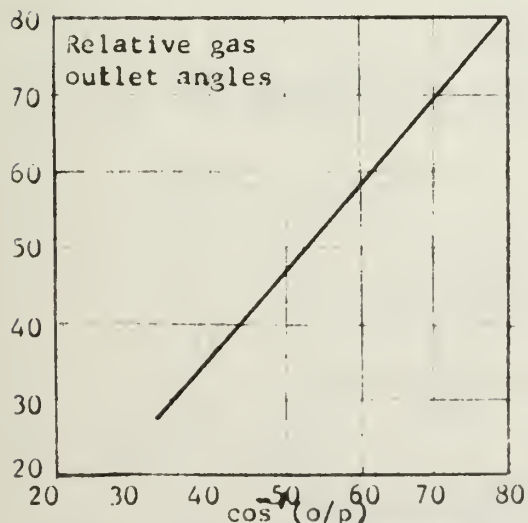


Figure 2-3. Relation between gas and blade angles (From D.G.Ainley and G.C.R.Mathieson(2))

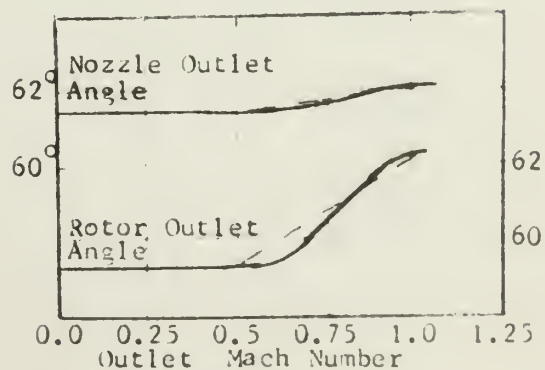


Figure 2-4. Angle variation with mach number (From D.G.Ainley and G.C.R.Mathieson(2))



with a curved back trailing edge in reference (2) but this will not be applied in the computer program.

2.3 The computer program can now determine the ideal pitch to chord ratio  $P/C$ , blade chord ( $C$ ), aspect ratio  $H/C$ , bending and centrifugal stresses, number of blades, 1st bending frequency, and weight of blades. As will be shown in this section all these parameters are interrelated and they all help determine each other.

### 2.3a Centrifugal Stress Calculations

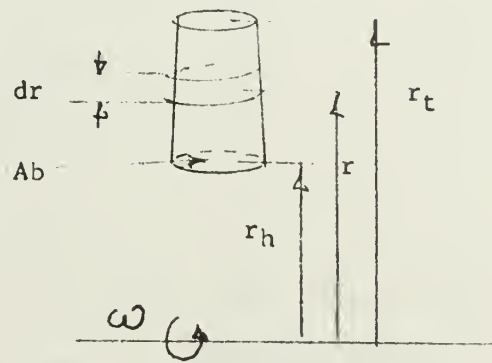


Figure 2-5. Rotating Turbine blade

$$dF = dmr \omega^2 = dmr (2\pi N/60)^2 \quad (2.3-1)$$

$$A(r) = A_b f(r) \quad (2.3-2)$$

$$dm = A_b f(r) dr \frac{\rho}{g_0} \quad (2.3-3)$$

$$dF = (2\pi N/60)^2 \frac{\rho}{g_0} A(r) r dr$$

If  $A(r) = \text{constant}$

$$F_c = (2\pi N/60)^2 \frac{A_b \rho}{g_0} \int_{r_{\text{hub}}}^{r_{\text{tip}}} r dr \quad (2.3-4)$$

$$\sigma_c = \frac{F_c}{A_b} = (2\pi N/60)^2 \frac{\rho}{g_0} \left[ \frac{r_{\text{tip}}^2 - r_{\text{hub}}^2}{2} \right]$$





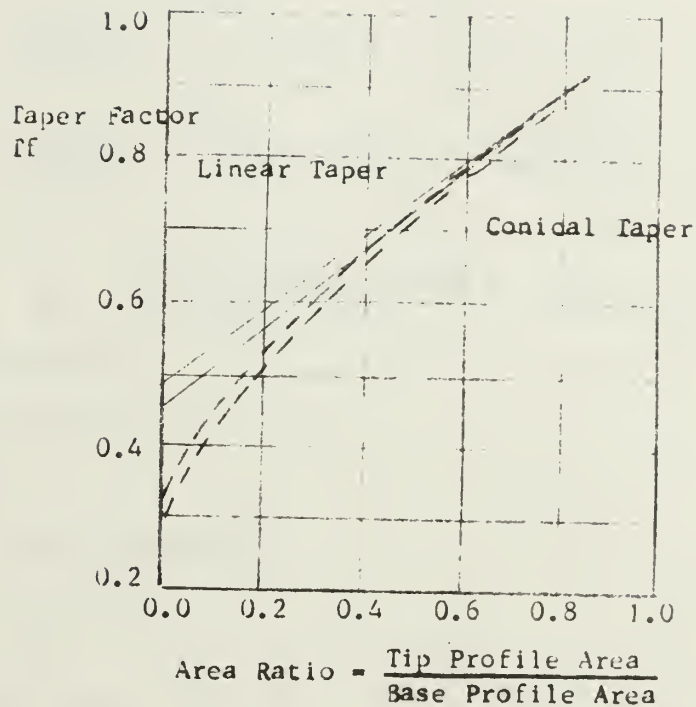


Figure 2-6. Variation of taper factor with blade profile area ratio (From Emmert, Current Design Practices for Gas Turbine Elements.(9))

$$c = 4.51 \text{ Tf } \rho \text{ Ab } \left( \frac{\Omega}{1000} \right)^2 \quad (2.3-5)$$

where;  $\sigma_c$  = average tensile stress(psi)

Tf = taper factor(from figure (2-6))

$\rho_b$  = specific mass of the blade

A = annular flow area of blade ring (sq. in.)

$\Omega$  = shaft speed (rpm)

For the computer program Tf will be based on linear variation with tip to hub area ratio (areara) as an input, normally this will be between 0.25 - 0.333



If a linear taper is assumed equation (2.3-1) can be integrated directly assuming;

$$A(r) = A_b(1 - \alpha r) \quad (2.3-6)$$

$$\begin{aligned} F_c &= \left( \frac{2\pi\Omega}{60} \right)^2 \frac{J_b}{g_o} A_b \int_{r_{hub}}^{r_{tip}} (1 - \alpha r) r dr \\ &= \left( \frac{2\pi\Omega}{60} \right)^2 \frac{J_b}{g_o} A_b \left[ \frac{(r_{tip}^2 - r_{hub}^2)}{2} + \frac{\alpha}{3} (r_{hub}^3 - r_{tip}^3) \right] \end{aligned} \quad (2.3-7)$$

$$\mathcal{E}_c = \frac{F}{A_b} = \left( \frac{2\pi\Omega}{60} \right)^2 \frac{J_b}{g_o} \left[ \frac{(r_{tip}^2 - r_{hub}^2)}{2} + \frac{\alpha}{3} (r_{hub}^3 - r_{tip}^3) \right] \quad (2.3-8)$$

and  $\alpha$  can be evaluated

$$A_{tip} = \alpha \text{ area ratio } (A_b)$$

$$\alpha = \frac{1 - \text{area ratio}}{r_{tip}} \quad (2.3-9)$$

2.3b Determination of ideal pitch to chord ratio. This value for P/C ratio is taken from figure (2-7) which is a correlation of Ainley's optimum spacing data. For stators the inlet gas angle to enter figure (2-7) is  $-\alpha_1$  and the relative efflux angle is  $\alpha_2$ . For rotors  $\beta_2$  is the inlet gas angle while  $-\beta_3$  is the relative efflux angle. This calculation is done in subroutine STOCRA and the input angles are taken at the mean radius.

2.3c Determination of non-dimensional section modulus (SM) and non-dimensional area at the base of the blade ( $\text{Area}/C^2$ ). An input to get these values is the turning angle at the base of the blade. ie (hub for rotor and tip for stator). Figure (2-8) taken from reference (7) illustrates how the non dimensional base area is obtained. The non-dimensional section modulus is obtained from figure (2-9) which is



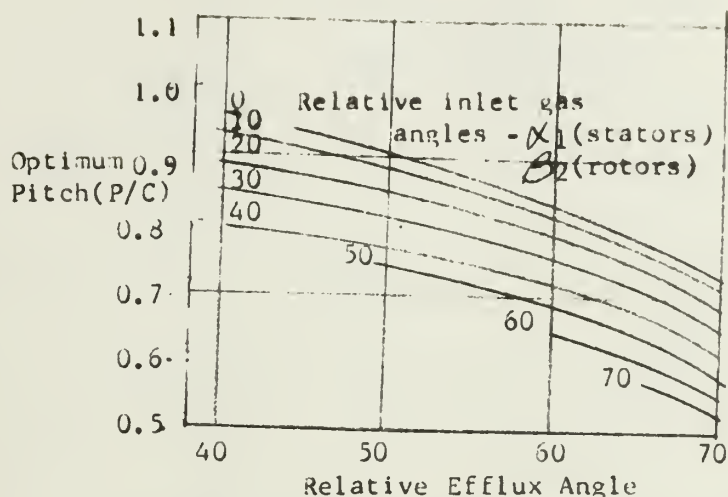


Figure 2-7. Optimum spacing of turbine blades. (From a correlation of Ainley's loss data (2))

an unpublished relationship provided by Ainley for use in reference (6).

A modification was made to this relationship based upon calculations performed by Professor A.D. Carmichael to make the section modulus larger to conform to modern turbine blades. The SM was multiplied by 2.35 for use in the computer program.

2.3d Determining bending stresses on turbine blades. As can be seen in figure (2-10) and equation (2.3-10) a detailed calculation would be needed to determine the gas bending stresses on a turbine blade but because angle  $\phi$  is small and  $M_w$  is by far the greater bending moment an approximation proposed in reference (6) is used in the preliminary design.

$$(\delta_{gb})_{\max} \approx \frac{m V_x (\tan \alpha_2 - \tan \alpha_3) h}{2 g o n L} \quad (\text{rotor}) \quad (2.3-11)$$

where the angles are taken at mean radius and the following are the definition of the parameters;



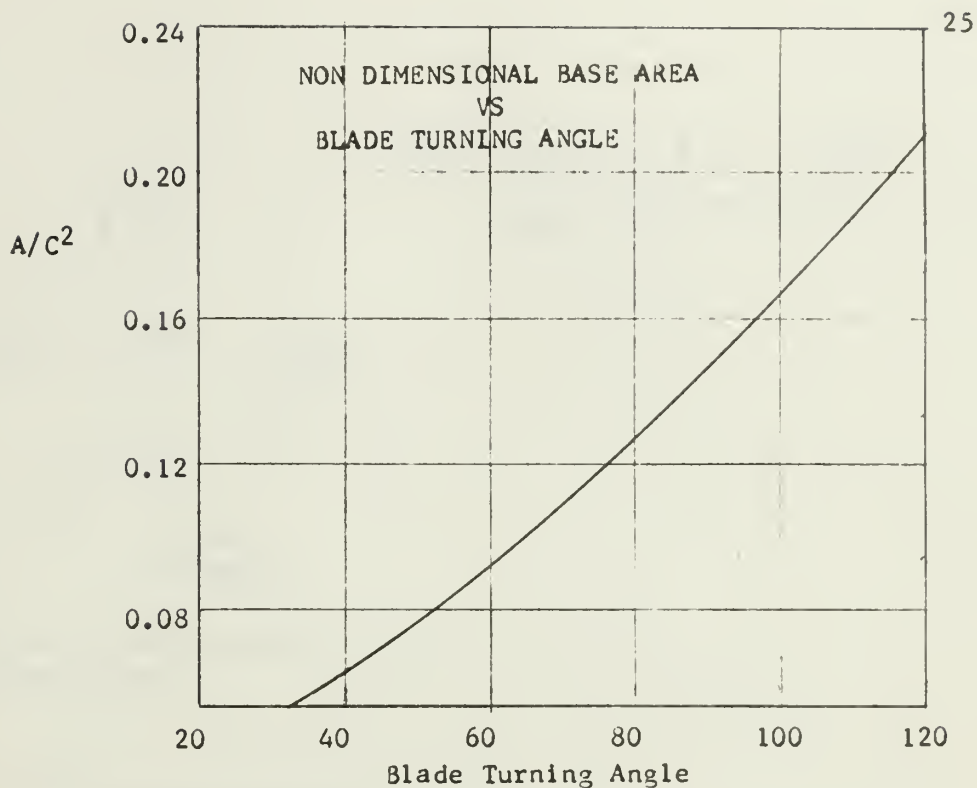


Figure 2-8. Approximate Areas of Turbine Blades vs Turning angle. (From paper by R.E.Dundas in Gas Turbine Handbook(7))

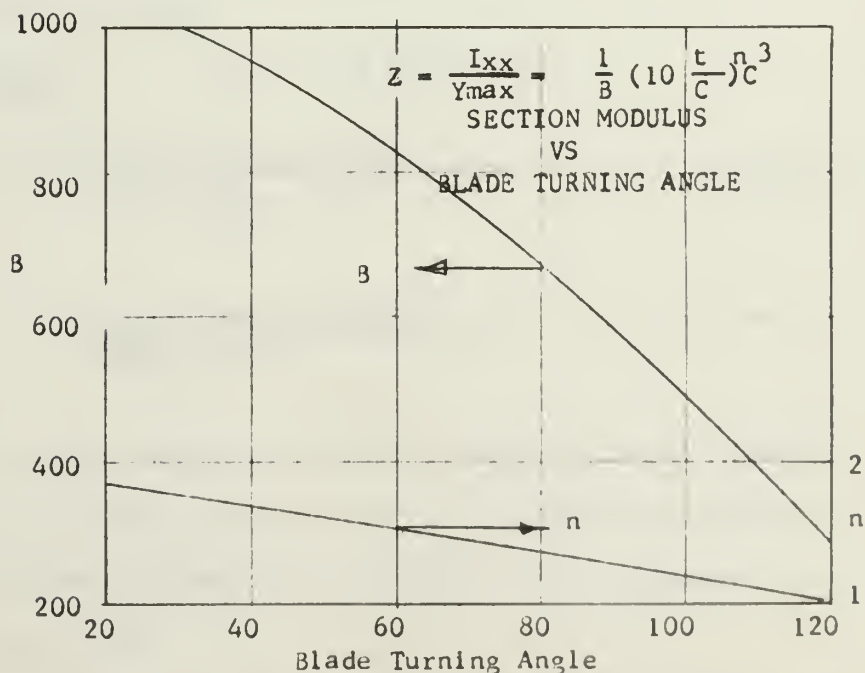
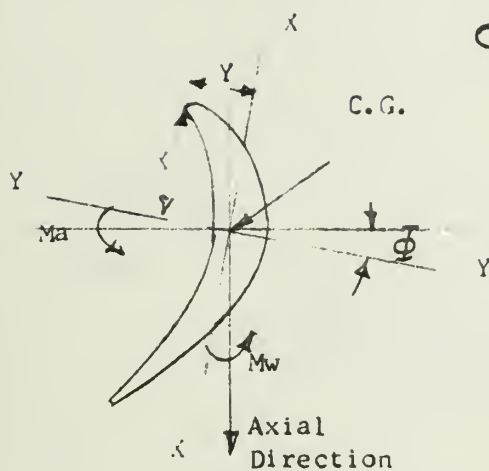


Figure 2-9. Section Modulus as a function of blade turning angle. (From unpublished paper by Ainley for use in reference (6))







$$\sigma_{gb} = \frac{X}{I_{yy}} (M_a \cos \phi - M_w \sin \phi) + \frac{Y}{I_{xx}} (M_w \cos \phi + M_a \sin \phi) \quad (2.3-10)$$

Figure 2-10. Bending moments acting on a turbine blade.

$Z$  = SM C Section Modulus

SM = non dimensional section modulus

$n$  = number of blades

$$= \frac{2\pi r_m}{\text{Pitch}} \quad \text{where } r_m = \text{mean radius}$$

$$(\sigma_{gb})_{\max} = \frac{\dot{m} V_x (\tan \alpha_2 - \tan \alpha_3) h}{2 g_o \frac{2\pi r_m}{P} \text{SM } C^3} \quad (2.3-11a)$$

$$= \frac{\dot{m} V_x (\tan \alpha_2 - \tan \alpha_3) h \frac{P}{C}}{4\pi g_o r_m \text{SM } C^2}$$

At this point the blade chord has not been determined so the bending stress cannot be calculated, but equation (2.3-11a) will be coupled with a natural frequency equation and a cyclic loading equation to calculate the blade chord.



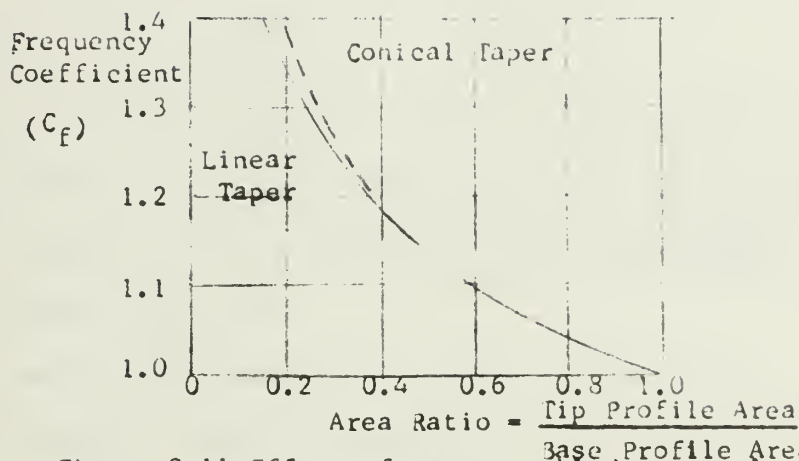


Figure 2-11 Effect of taper on blade natural frequency (From Emmert, Current Design Practices for Gas Turbine Power Elements (9))

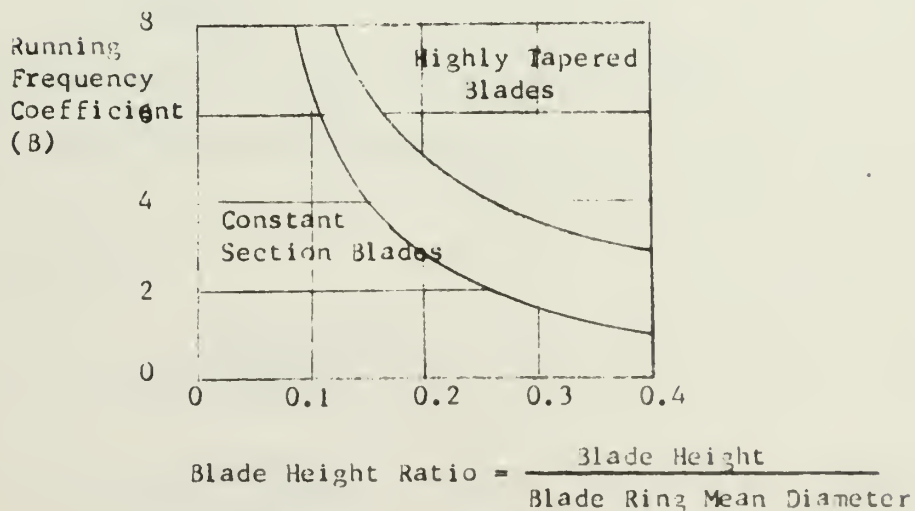


Figure 2-12. Variation of rotation coefficient with blade height ratio. (From Emmert, Current Design Practices for Gas Turbine Power Elements. (9))



2.3e Determination of natural bending frequency of blades from 23 reference (9). The following expression is used in the computer program.

$$f_s = 11.0 C_f \frac{k}{h^2} \sqrt{\frac{E}{\rho}} \quad (2.3-12a)$$

$f_s$  is standing frequency (cps)

$C_f$  frequency correction factor taken from figure (2-11)

$k$  minimum radius of gyration of base profile (in)

$h$  blade height (in)

$E$  modulus of elasticity (psi)  $(29 - 30) \times 10^6$  psi

$\rho$  specific weight of blade material (lb/in<sup>3</sup>)

$$f_s = 11.0 \frac{C_f}{h^2} \sqrt{\frac{IE}{A \rho}} \quad (2.3-12b)$$

$$= 11.0 \frac{C_f}{h^2} \sqrt{\frac{ZtE}{A \rho}} \quad \text{assuming } I \approx Zt$$

$A$  cross sectional area of base of blade (in<sup>2</sup>)

$t$  maximum thickness of blade (in)

$$= (t/C) C$$

$$A = (A/C^2) C^2$$

$$f_s = 11.0 \frac{C_f}{h^2} \sqrt{\frac{SM C^3 (t/C) E C}{(A/C^2)}}$$

$$= 11.0 \frac{C_f}{h^2} \left[ \frac{SM(t/C)E}{(A/C^2)} \right]^{\frac{1}{2}} \text{ chord} \quad (2.3-12c)$$

The running frequency  $f$  given by equation (2.3-13) is a function of the natural frequency and the speed of rotation of the blade. The



factor  $\beta$  is given by figure (2-12). A plot of this frequency as a function of operating speed is illustrated in figure (2-13) which is called a Cambell Diagram. Normally for good design practice the running frequency should avoid the 1st four harmonics of the operating speed in its operating range. For the design of the blade chord in the computer program the effect of the rotational speed as indicated in equation (2.3-13) will be neglected but a more stringent criteria using the harmonics of the operating speed to estimate the vibrational stress in the blade will be used.

$$f_r = \sqrt{f_s^2 + \beta(\Omega/60)^2} \quad (2.3-13)$$

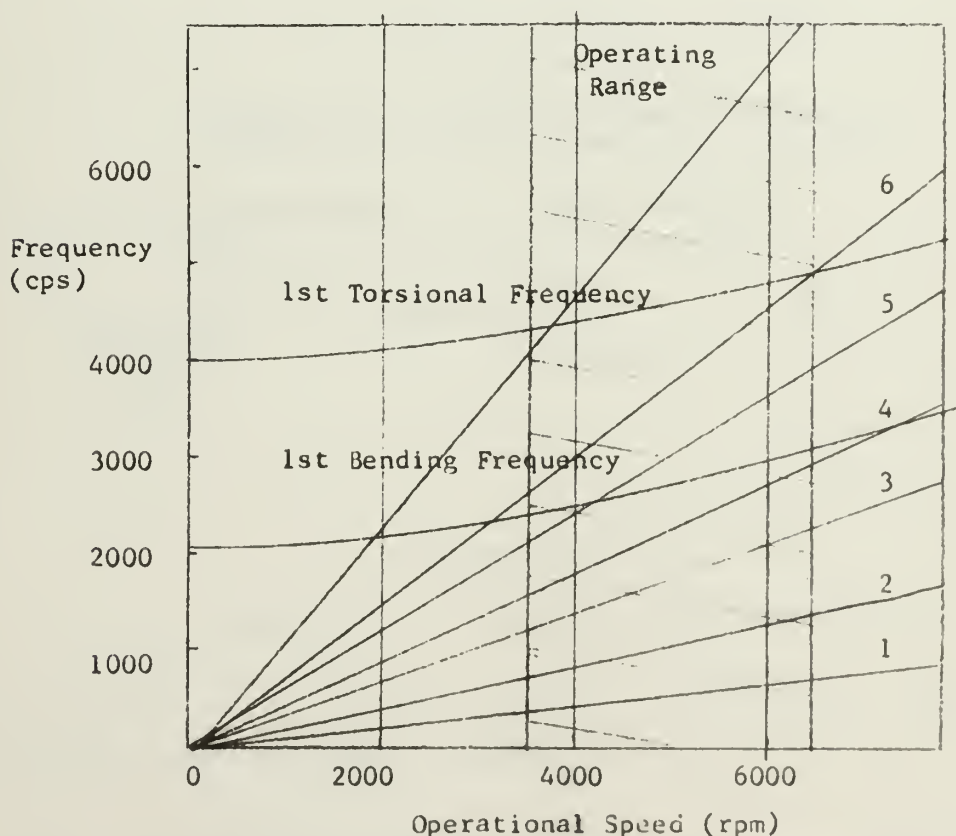


Figure 2-13. Cambell diagram, plot of frequency of vibrations as a function of operating speed.





2.3f Determination of turbine blade characteristics based on cyclic loading requirements by use of a Goodman Diagram of steady and vibrational stresses. In reference (7) a use of the Goodman diagram is used to determine the chord of a compressor blade. This can also be applied to turbine blades. A Goodman diagram is a plot of vibrational stresses as a function of steady stresses. The vibrational stress is taken at  $10^7$  cycles at the operating temperature.

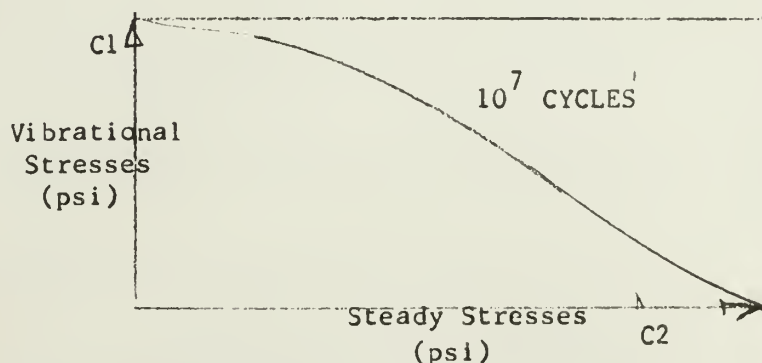


Figure 2-14. Goodman Diagram of Vibrational stresses as a function of steady stresses.

This plot can be approximated as a linear function.

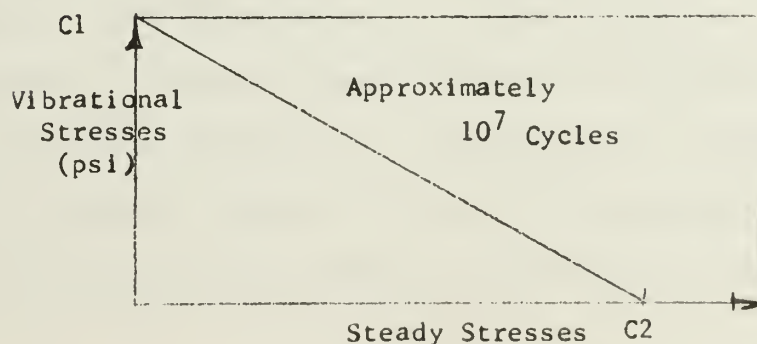


Figure 2-15. Linearized Goodman Diagram

In reference (7) the vibrational stress is proposed as a function of the bending stress as shown in equation (2.3-14)

$$\sigma_{\text{vib}} = 1.3 \text{ Amplification factor } \sigma_{\text{gb}} \quad (2.3-14)$$



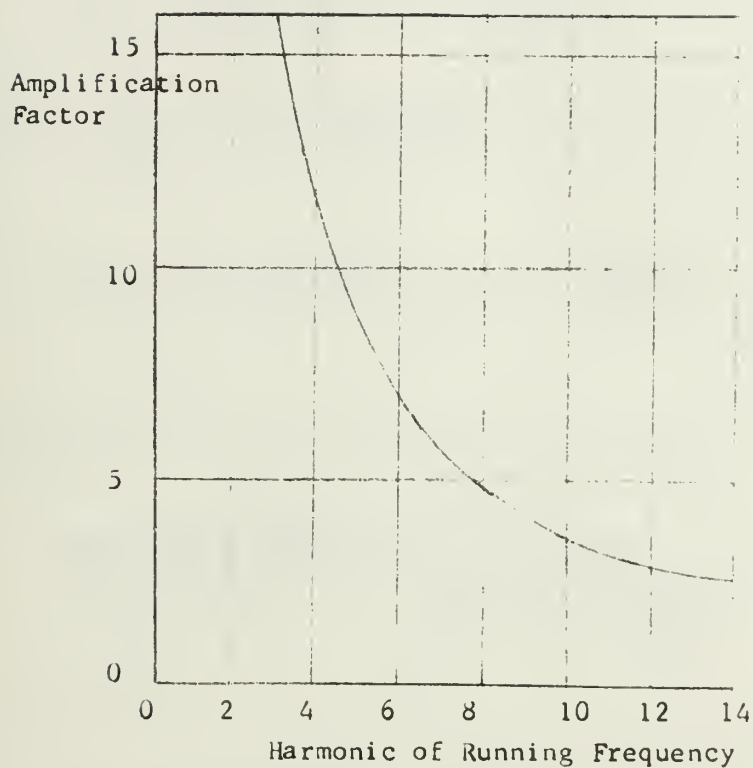


Figure 2-16. Amplification Factor. (From Paper of Trumpler & Owens (19))

where Amplification factor is shown in figure (2-16) as a function of the harmonic of operating speed. The 1.3 is a stress concentration factor specified in reference (7). From personal conversation with Professor A. Douglas Carmichael at M.I.T. the amplification factor can be approximated by  $41/n$  where  $n$  is the harmonic of the operating speed.

$$n = \frac{f}{\left( \frac{\Omega(\text{rpm})}{60} \right)} = \left( \frac{f}{\Omega} \right) 60 \quad (2.3-15)$$

From figure (2-15)

$$\delta_{\text{vib}} = C1 - \frac{C1}{C2} \delta_{\text{steady}} \quad (2.3-16)$$

where  $\delta_{\text{steady}} = \delta_{\text{centrifugal}} + \delta_{\text{gas bending}}$

and  $C1$  is the upper limit for vibrational stresses and  $C2$  is the



upper limit for steady stresses.

32

$$\delta_{gb} \frac{1.3 \Omega / 60}{f} + \frac{C1}{C2} = C1 - \frac{C1}{C2} (\delta_{cf})$$

$$\text{but } \frac{C1}{C2} \ll \frac{1.3 (\Omega / 60) 41}{f}$$

$$\text{so } \delta_{gb} \frac{(1.3 \times 41 \times \Omega / 60)}{f} \approx C1 - \frac{C1}{C2} (\delta_{cf}) \quad (2.3-17)$$

From equation (2.3-11a), (2.3-12c), and (2.3-17)

$$\frac{\dot{m} V_x (\tan \alpha_2 - \tan \alpha_3) \left( \frac{p}{C} \right) h (1.3 \times 41 \times (\Omega / 60))}{4 \dot{m}_{go} r_m SM C^3 \left[ \frac{11.0 \times C_f}{h^2} \left[ \frac{SM(t/C) E}{(A/C^2) g} \right] \right]^{1/2}} = C1 - \frac{C1}{C2} \delta_{cf} \quad (2.3-18)$$

$$\text{Stress}_3 = \frac{\dot{m} V_x (\tan \alpha_2 - \tan \alpha_3) h \left( \frac{p}{C} \right)}{4 \dot{m}_{go} r_m SM} \quad (2.3-19)$$

$$\text{Freq} = \frac{11.0 \times C_f}{h^2} \left[ \frac{SM(t/C)}{(A/C^2) g} \right]^{1/2} \quad (2.3-20)$$

$$\frac{1.3 \times 41 (\Omega / 60)}{C^3 \text{Freq}} (\text{Stress}_3) = C1 \left( 1 - \frac{\delta_{cf}}{C2} \right) \quad (2.3-13a)$$

$$C4 = 1.3 \times 41 (\Omega / 60) \text{Stress}_3 / \text{Freq} \quad (2.3-21)$$

$$\frac{C4}{C^3} = C1 \left( 1 - \frac{\delta_{cf}}{C2} \right)$$

$$\text{Chord} = C = \left[ \frac{C4}{C1 \left( 1 - \frac{\delta_{cf}}{C2} \right)} \right]^{1/3} \quad (2.3-22)$$



Now that the chord has been determined the bending stress and natural bending frequency can be determined from equation (2.3-11a) and (2.3-12c) respectively. Having solved for the chord at the base of the blade the chord can be determined at mean and tip assuming a linear taper.

$$Ab_m = \frac{Ab(1 + A_{reara})}{2} \quad (2.3-23)$$

Assuming constant  $(A/C^2)$

$$C_m = \frac{[Ab_m]^{1/2}}{[A/C^2]} \quad (2.3-24)$$

At the opposite end from the base

$$C_t = \frac{[A_{reara}(Ab)]^{1/2}}{[A/C^2]} \quad (2.3-25)$$

where  $C_t = C_{hub}$  (for stator)

$C_t = C_{tip}$  (for rotor)

Now that the chord has been determined at the midchord the aspect ratio and pitch can be determined by equation (2.3-26) and (2.3-27).

$$\text{Aspect ratio } (h/C) = h/C_m \quad (2.3-26)$$

$$\text{Pitch}_m = (P/C) C_m \quad (2.3-27)$$

With pitch determined the number of blades can be calculated,

$$\text{Number of blades } (N) = \text{Integer } \frac{2\pi r_m}{\text{Pitch}} \quad (2.3-28)$$

The computer program uses the criteria of having even number of blades for the stator and prime numbers for the rotor to prevent blade interaction of resonant frequencies. The determination of whether a number is prime is done in subroutine TESTP. Thus having solved for the number of blades the pitch can be recalculated and the weight





of the blades. Blades are integrated from the base to end to get volume and is multiply by its specific weight to get the total weight. In the computer program this specific weight is used for both the blades and disc. Also it is assumed they are solid blades with no cooling passages.

$$\begin{aligned}
 \text{Weight of blades} &= N_b \int_{r_h}^{r_t} A(r) dr & (2.3-29) \\
 &= N_b A_b \int_r^{r_t} (1 - \alpha r) dr \\
 &= N_b A_b \left[ (r_t - r_h) + \frac{\alpha}{2} (r_h^2 - r_t^2) \right]
 \end{aligned}$$

where  $\alpha$  is given by equation (2.3-9)

In this section pitch was determined by a chart of ideal pitch to reduce losses. In actual turbine design the loss data would be coupled with stress analysis of the turbine blade attachment to determine allowable pitch. The type most commonly used is the fir-tree method which is illustrated in figure (2-17). This method of selecting turbine blade chord length is just one of many which are used in industry.

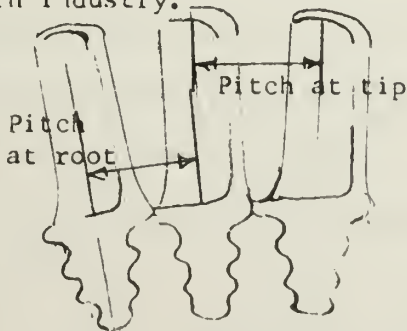


Figure 2-17. Firtree blade attachments.



## 2.4 Turbine Disc Design

The method for design of a disc in this section will be based upon a constant average stress analysis. The stress relations are those proposed in reference (15) and (18). This method will ignore stress concentrations and thermal stresses. In actual disc design these cannot be ignored since thermal stresses may be a major part of the disc stresses and stress concentrations may cause the disk to fail even though the average stress is well below the ultimate tensile stress of the material (UTS). For calculations in the computer program the average tensile stress used is that shown in equation (2.4-1) from reference (7).

$$\sigma_{avg} = \frac{0.75 \text{ UTS}}{\left[ \frac{N_b}{N_o} \right]^2} \quad (2.4-1)$$

$$\left[ \frac{N_b}{N_o} \right] = \text{Ratio of burst speed to design speed} \quad (2.4-2)$$

The value of this ratio is normally between 1.2 - 1.3. The value of ultimate tensile stress (UTS) is taken at the design operating temperature, higher values producing smaller disc dimensions. This value is an input to the computer program. The computer program has minimum values for some of the dimensions illustrated in figure (2-13) to prevent impossibilities to manufacture and to allow adequate thickness for attachment of blades.

The following equations are based on figure (2-13) taken from reference (15).

$$\sigma_{avg} = \frac{\rho \omega^2 r_o^2 + \frac{N}{2\pi} \sigma_b \frac{A_b}{W_o T_o}}{1 + \frac{Z_o r_o}{W_o T_o}} \quad (2.4-3)$$



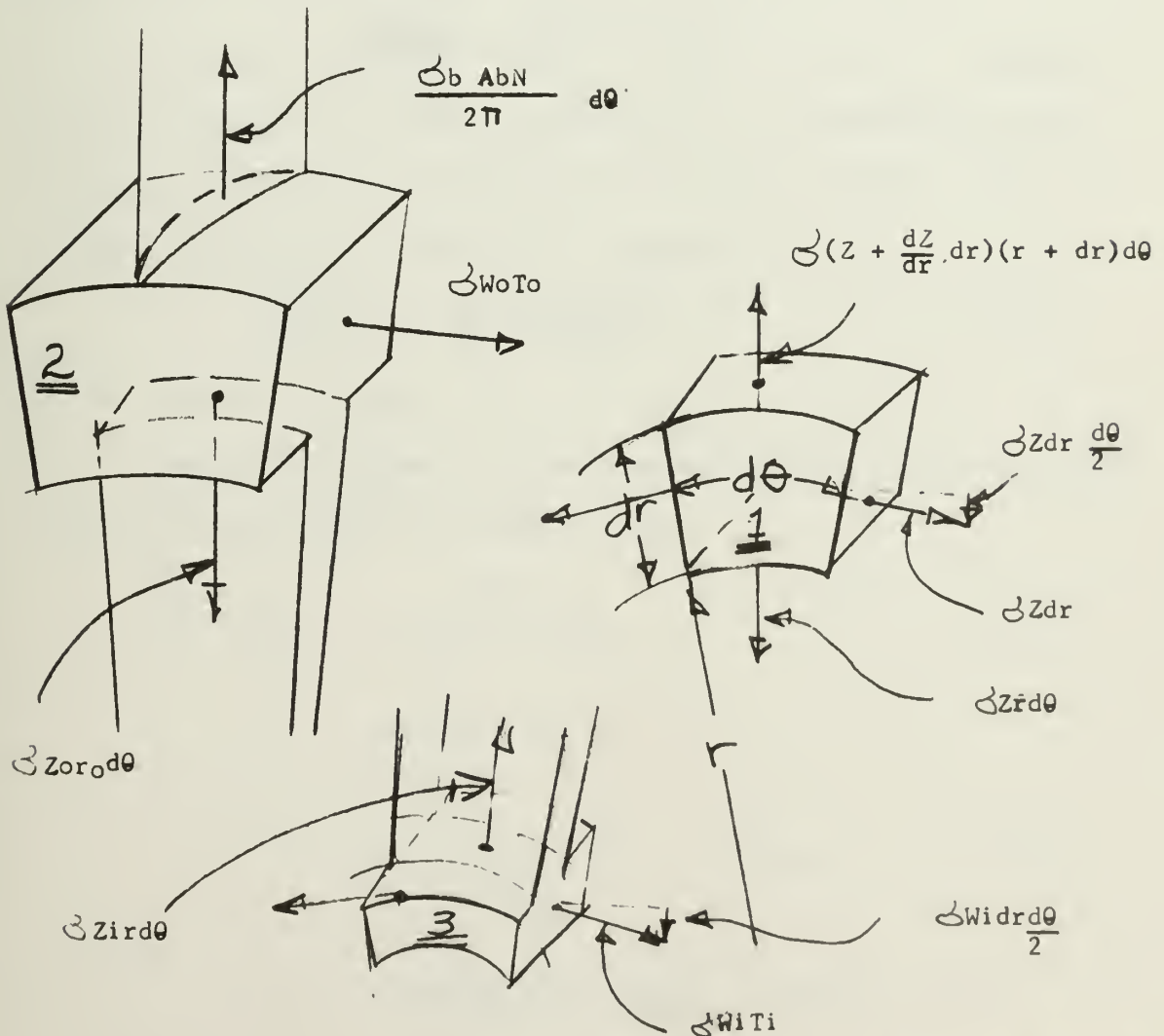
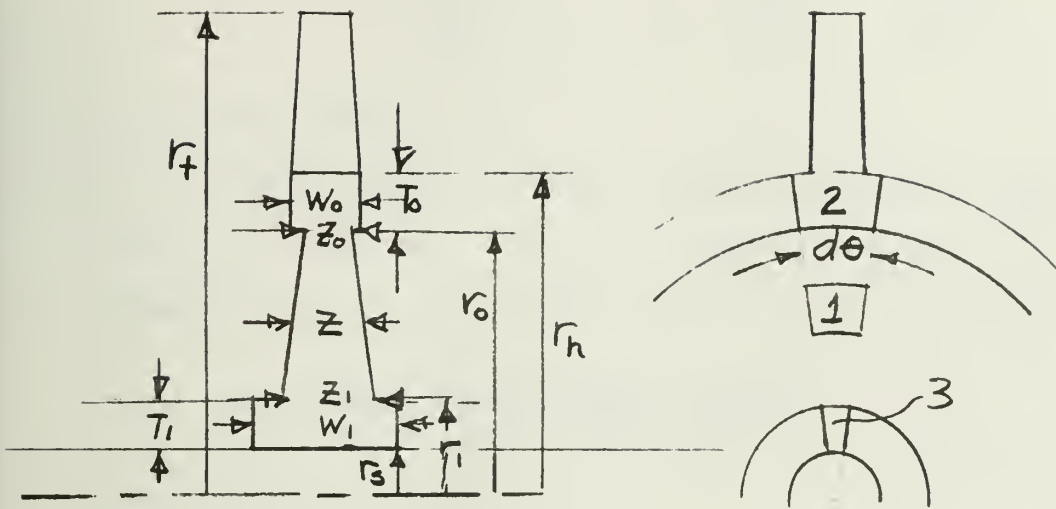


Figure 2-18. Turbine disc stress analysis (From Aircraft Turbines by J.L.Kerrebrock(15))



$$\delta_b = \frac{\rho \omega^2 r_t^2}{2g_o} \left\{ 1 - \left( \frac{r_h}{r_t} \right)^2 - \frac{2\alpha r_t}{3} \left[ 1 - \left( \frac{r_h}{r_t} \right)^3 \right] \right\} \quad (2.4-4) \quad 37$$

for a linear tapered blade.

For the computer program  $W_o$  will be assumed to equal .65 chord and  $T_o$  will equal half of  $W_o$  unless otherwise specified. With these parameters known  $Z_o$  can be solved for by equation (2.4-5). With  $Z_o$

$$Z_o = \left( \frac{W_o T_o}{r_o} \right) \left\{ \left[ \left( \frac{r_o}{r_t} \right)^2 + \frac{N \delta_b}{2\pi W_o T_o} \right] \frac{\rho \omega^2 r_t^2}{2g_o} - 1 \right\} \quad (2.4-5)$$

known  $Z(r)$  can be solved for.

$$Z(r) = Z_o e^{\frac{\rho \omega^2 r_t^2}{2g_o} \left[ \left( \frac{r_o}{r_t} \right)^2 - \left( \frac{r}{r_t} \right)^2 \right]} \quad (2.4-6)$$

with a minimum of 0.3 inches. The value of  $r_i$  is assumed to equal 2  $r_{shaft}$  where  $r_{shaft}$  is an input to the computer program. This defines  $T_i$  so  $W_i$  can be solve for in equation (2.4-7).

$$W_i = \left( \frac{Z_i r_i}{T_i} \right) \left[ 1 - \left( \frac{r_i}{r_t} \right)^2 \frac{\rho \omega^2 r_t^2}{g_o} \right] \quad (2.4-7)$$

with a minimum of 1.2  $W_o$ .

Having determined the dimensions of the disc the weight of the disc can be determined by equation (2.4-8)

$$\begin{aligned} \text{Disc weight} &= \rho \int_0^{2\pi} \int_{r_s}^{r_o} A(r) dr d\theta \quad (2.4-8) \\ &= \rho 2\pi \int A(r) dr \\ &= 2\pi \rho \left[ W_i \int_{r_{shaft}}^{r_i} r dr + \int_{r_i}^{r_o} Z(r) r dr + W_o \int_{r_o}^{r_h} r dr \right] \\ &= 2\pi \rho \left\{ \frac{W_i}{2} (r_i^2 - r_{shaft}^2) + \frac{W_o}{2} (r_h^2 - r_o^2) \right. \\ &\quad \left. + 2Z_o r_t^2 e^{\frac{\rho \omega^2 r_o^2}{2g_o}} \left[ e^{-\frac{\rho \omega^2 r_t^2}{2g_o} \left( \frac{r_i}{r_t} \right)^2} - e^{-\frac{\rho \omega^2 r_t^2}{2g_o} \left( \frac{r_o}{r_t} \right)^2} \right] \right\} \end{aligned}$$





Having determined the dimensions of the disc, the burst speed can be computed assuming the disc yields prior to bursting. For this analysis the disc is divided in half as illustrated in figure (2-19) and the stress is assumed constant over area  $2A$ . The force acting on this area is composed of the disc load and the blade load. It is assumed that the blade load is evenly distributed on the outer rim. In actual stress analysis of the disc this assumption is an over simplification of the stress, but for preliminary design purposes it will be adequate. A more detailed analysis is covered in reference (7). The force due to the disc is now calculated by taking the force in the direction perpendicular to the area  $2A$  caused by the disc load and summing it over the disc half and adding this to the force caused by the blade load. This force is divided by the area  $2A$  to get the average tensile stress  $\sigma_{ta}$ . This stress is set equal to the ultimate tensile strength of the material of the disc at its operating temperature. The value of  $N_b$  is then calculated by equation (2.4-16)



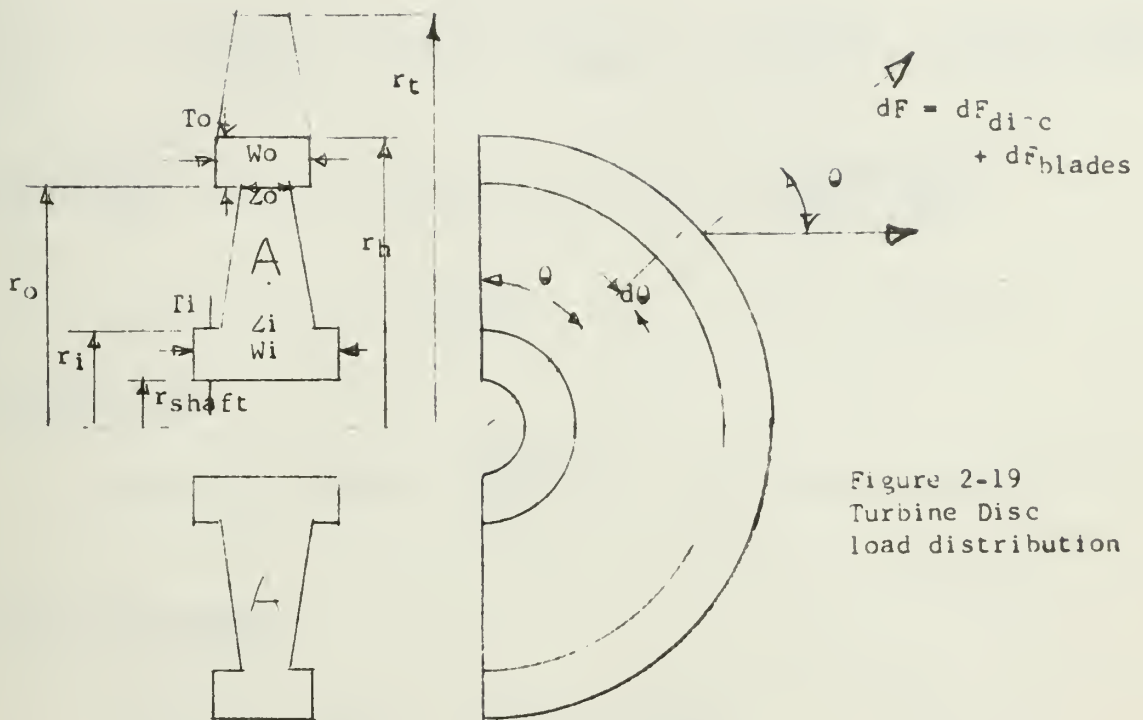


Figure 2-19  
Turbine Disc  
load distribution

Total area =  $2A$

$$A = W_i T_i + \int_{r_i}^{r_o} Z(r) dr + W_o T_o \quad (2.4-9)$$

For disc load

$$dm = \int_{\theta_0}^{\theta} dV \quad (2.4-10)$$

$$dF = (dm r \omega^2) \sin \theta = dm r \left( \frac{2\pi N}{60} \right)^2 \sin \theta \quad (2.4-11)$$

$$\begin{aligned} F &= \left( \frac{2\pi N}{60} \right)^2 \int_{\theta_0}^{\theta} \int_{r_{shaft}}^{r_h} Z(r) r^2 \sin \theta d\theta dr \\ &= \left( \frac{2\pi N}{60} \right)^2 \int_{\theta_0}^{\theta} 2 \int_{r_{shaft}}^{r_h} Z(r) r^2 dr \end{aligned} \quad (2.4-12)$$



$$F = \left( \frac{2\pi N b}{60} \right)^2 \frac{\rho}{2} \left[ W_i \int_{r_{\text{shaft}}}^{r_i} r^2 dr + \int_{r_i}^{r_o} Z(r) r^2 dr + W_o \int_{r_o}^{r_h} r^2 dr \right]$$

$$= \left( \frac{2\pi N b}{60} \right)^2 \frac{\rho}{2} \left[ \frac{W_i (r_i^3 - r_{\text{shaft}}^3)}{3} + Z_o \int_{r_i}^{r_o} e^{-a r} r^2 dr + \frac{W_o (r_h^3 - r_o^3)}{3} \right]$$

$Z_o e^{\frac{\rho \omega^2 r^2}{2g_o}} \int_{r_i}^{r_o} e^{-a r} r^2 dr$  is integrated numerically  $a = \frac{\rho \omega^2 r^2}{2g_o} \left[ \left( \frac{1}{r_*} \right)^2 \right]$

$$\sigma_{ta}(\text{due to disc}) = \frac{F_{\text{disc}}}{2A} \quad (2.4-13)$$

For blade load

$$\sigma_{ta}(\text{due to blades}) = \frac{N \sigma_b (A_b)}{2\pi A} \quad \sigma_b \text{ given by equation (2.4-4)}$$

(2.4-14)

Average disc stress

$$\sigma_{ta}(\text{Disc} + \text{blades}) = \frac{F_{\text{disc}} + F_{\text{blades}}}{2A} \quad (2.4-15)$$

Set  $\sigma_{ta} = \text{ULT}$  (ultimate tensile strength of material)

The burst speed  $N_b$  can now be calculated. This value will slightly higher than that in equation (2.4-2) due to approximations and arbitrary values selected for some of the disc dimensions.

$$N_b = \left( \frac{30}{\pi} \right) \left[ \frac{\frac{\rho}{2} \sigma_{\text{ult}} 2A}{\int_{r_{\text{shaft}}}^{r_h} Z(r) r^2 dr + \frac{N A_b r}{2\pi} \left[ 1 - \left( \frac{r_h}{r_t} \right)^2 - \frac{\alpha r_t}{3} \left[ 1 - \left( \frac{r_h}{r_t} \right)^3 \right] \right]} \right]^{\frac{1}{2}}$$

(2.4-16)



2.5 Having determined the blade and disc dimensions the axial length of the turbine can be calculated. To do this a criteria for spacing between blades must be selected. From reference (6) the values of spacing between blades is 0.2 to 0.5 of the axial chord length. The longer dimension will tend to reduce the flare which would prevent the gas flow to separate from the turbine wall. It would also decrease the chance of inducing vibrational stresses. The shorter dimension would decrease the size and weight of the turbine. For the computer program a value of  $\frac{1}{2}$  the average axial chord at mid-height was selected. The computer program will then calculate the axial length and weight of the shaft from station 1 of the 1st stage to station 3 of the last stage. A value of  $18^\circ$  was selected as the stagger angle in the computer program.

Figure 2-20 Axial Spacing between blades

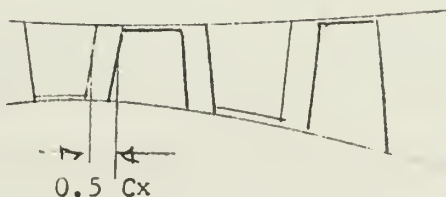
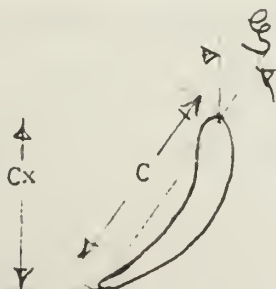


Figure 2-21 Stagger Angle of Turbine blades







## Performance Estimation

3.1 The design and off-design characteristics are very important in a complete analysis of a gas turbine since most turbines do not operate only at their design point. Also cycle calculations without actual loss data can not accurately determine the efficiency of a turbine, so this analysis can give a more accurate assessment of the efficiency that can be achieved in the turbine.

The Ainley-Mathieson Method for performance estimation was selected over other methods because it seems to have a wider application and can be easily used for off-design performance calculations. The modifications proposed in reference (8) have been applied. These modifications can reduce the error from  $\pm 3\%$  to  $\pm 2\%$  for most turbines and are based on modern turbines while the data by Ainley and Mathieson were accumulated over 20 years ago on turbines that existed at that time. The calculation procedure for off-design performance is that proposed by Horlock in reference (12). A description of the Improved Ainley Mathieson Performance Estimation Method will be presented first followed by a description of off-design calculations which use loss coefficients to determine gas properties, isentropic efficiency and static efficiency at design and off-design points.

The computer program used pressure loss coefficient  $Y = (P_{01} - P_{02}) / (P_{02} - P_z)$  for the performance analysis. This coefficient is a function of aspect ratio ( $H/C$ ), pitch to chord ratio ( $P/C$ ), velocity, Mach number, trailing edge thickness, thickness to chord ratio ( $t/C$ ), incidence angle, tip clearance, Reynolds number, etc. This coefficient



Loss Coefficient

$$\gamma = \frac{P_{01} - P_{02}}{P_{02} - P_2}$$

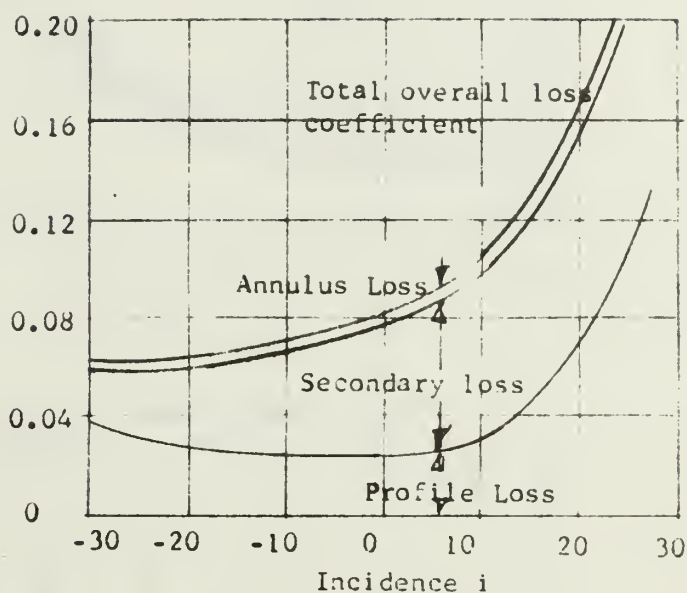


Figure 3-1. Analysis of losses in flow through a row of turbine blades (From D.G. Ainley and G.C.R. Mathieson(1))

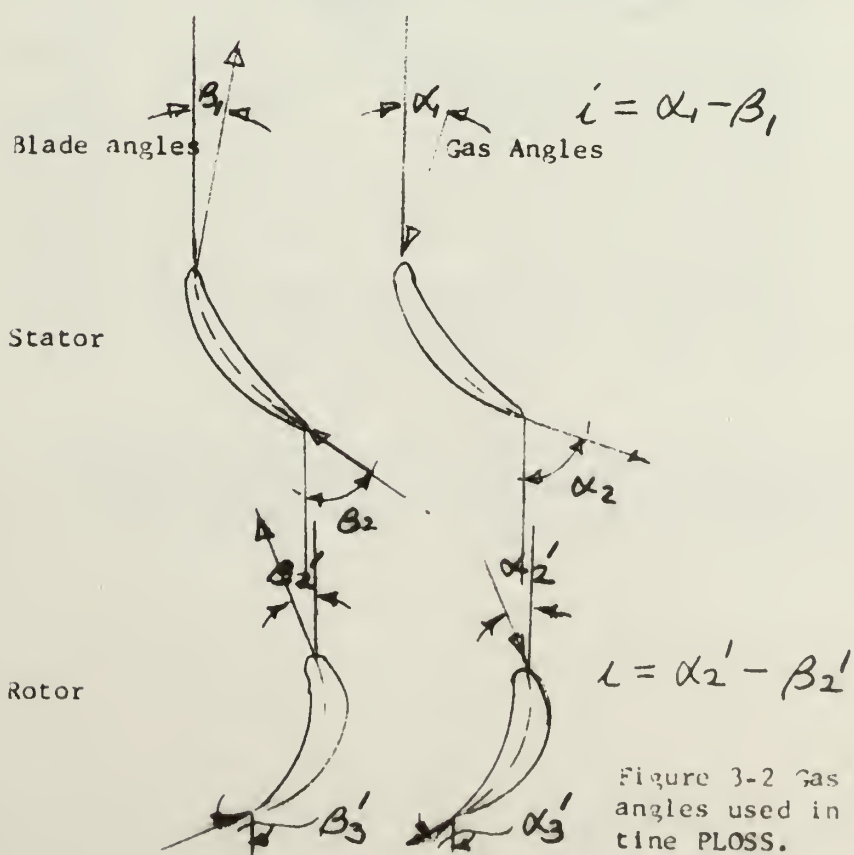
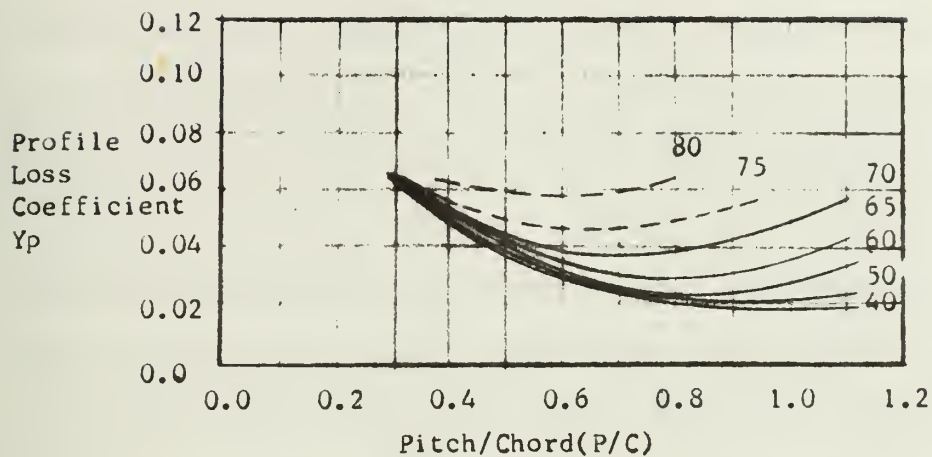
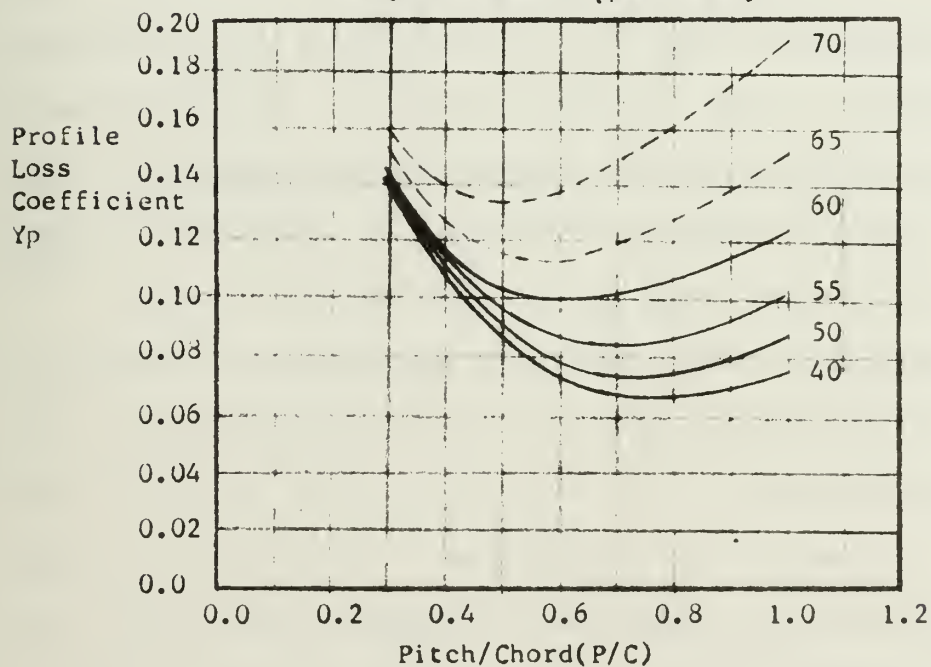


Figure 3-2 Gas and blade angles used in subroutine PLOSS.



Nozzle Blades ( $\beta_1 = 0$ )

(a)

Impulse Blades ( $\beta_1 = \alpha_2$ )

(b)

Figure 3-3. Profile-loss coefficients for conventional section blades at zero incidence.  $t/c = 20$  per cent;  $Re = 2 \times 10^5$ ;  $M = 0.6$ . From D.G. Ainley and G.C.R. Mathieson(2))



is broken down into three parts,  $Y_p$  (profile loss),  $Y_s$  (secondary loss), and  $Y_k$  (clearance loss). Figure (3-1) from reference (1) shows the first two,  $Y_p$  and  $Y_s$  in which the annulus loss is grouped into secondary losses. The modification of the Ainley-Mathieson method mostly affects the secondary and tip clearance loss coefficients, but has an affect on profile loss  $Y_p$  for Mach numbers greater than 1, and a Reynolds number correction to the profile and secondary loss coefficients.

3.2 The pressure loss coefficient  $Y$  is calculated in subroutine PLOSS. Figure (3-2) shows the sign convention used in this subroutine, where  $\beta$  angles are blade angles and  $\alpha$  angles are gas angles. The angles as shown are in the positive direction. The (') implies relative angles. This sign convention and notation applies only to subroutine PLOSS in order to have a standard form for stator and rotor.

Figure (3-3a) and 3-3b) coupled with equation (3.2-1) is used to calculate the profile loss coefficient  $Y_p$  for 0 incidence, where  $\beta_1 = 0$  values come from figure (3-3a) and  $\beta_1 = \alpha_2$  comes from figure (3-3b). The pitch to chord ratio ( $P/C$ ) is the argument for entering figures (3-3a) and (3-3b). The correction for thickness to chord ratio ( $t/C$ ) is done in equation (3.2-1) along with the correction for the ratio of inlet blade angle to exit gas angle ( $\beta_1/\alpha_2$ ). The value obtained is  $Y_p$  for zero incidence ( $i=0$ ).

For off-incidence calculations figures (3-4) a,b,c, and figure (3-5) are used. Figure (3-4) a,b,c determine the stalling incidence so that figure (3-5) can then be used to correct  $Y_p$  for off-incidence cases.





$$Y_p = \left\{ Y_p(\beta_1=0) + (\beta_1/\alpha_2)^2 [Y_p(\beta_1=\alpha_2) - Y_p(\beta_1=0)] \right\} \left[ \frac{t/C}{0.2} \right] (\beta_1/\alpha_2) \quad (3.2-1)$$

$$\alpha_2(P/C=.75) = \frac{\alpha_2}{(\alpha_2/(\alpha_2(P/C=.75)))} \quad (3.2-2)$$

Figure (3-4b) is entered with the argument of pitch to chord ratio (P/C). The ratio of exit gas angle  $\alpha_2$  to exit angle  $\alpha_2(P/C=.75)$  is then determined. It is now possible to determine  $\alpha_2(P/C=.75)$ . Having determined  $\alpha_2(P/C=.75)$ , figure (3-4c) is now entered with  $\alpha_2(P/C=.75)$  to get the stalling incidence  $i_s(P/C=.75)$ . Figure (3-4a) is now entered with exit gas angle  $\alpha_2$  and P/C to get  $\Delta i_s = i_s - i_s(P/C=.75)$ .  $i_s$  can then be solved for. Having incidence angle  $i$  and  $i_s$  the ratio  $(i/i_s)$  for entering figure (3-5) to determine ratio  $Y_p/Y_p(i=0)$  is known. Having previously solved for  $Y_p(i=0)$  in equation (3.2-1)  $Y_p$  is now known for off incidence.

$$Y_s = .0334(C/H)(\cos^2 \alpha_2 / \cos^3 \alpha_m) Z \quad (3.2-3)$$

For secondary losses  $Y_s$  is computed by equation (3.2-3) from reference (8). Where  $Z$  is the Ainley loading parameter from reference (2).

$$Z = \left[ \frac{C}{P/C} \right] \frac{\cos^2 \alpha_2}{\cos^3 \alpha_m} \quad (3.2-4)$$

$$\alpha_m = (\alpha_1' - \alpha_2)/2 \quad (\text{for stator}) \quad (3.2-5a)$$

$$\alpha_m = (\alpha_3' - \alpha_2')/2 \quad (\text{for rotor}) \quad (3.2-5b)$$



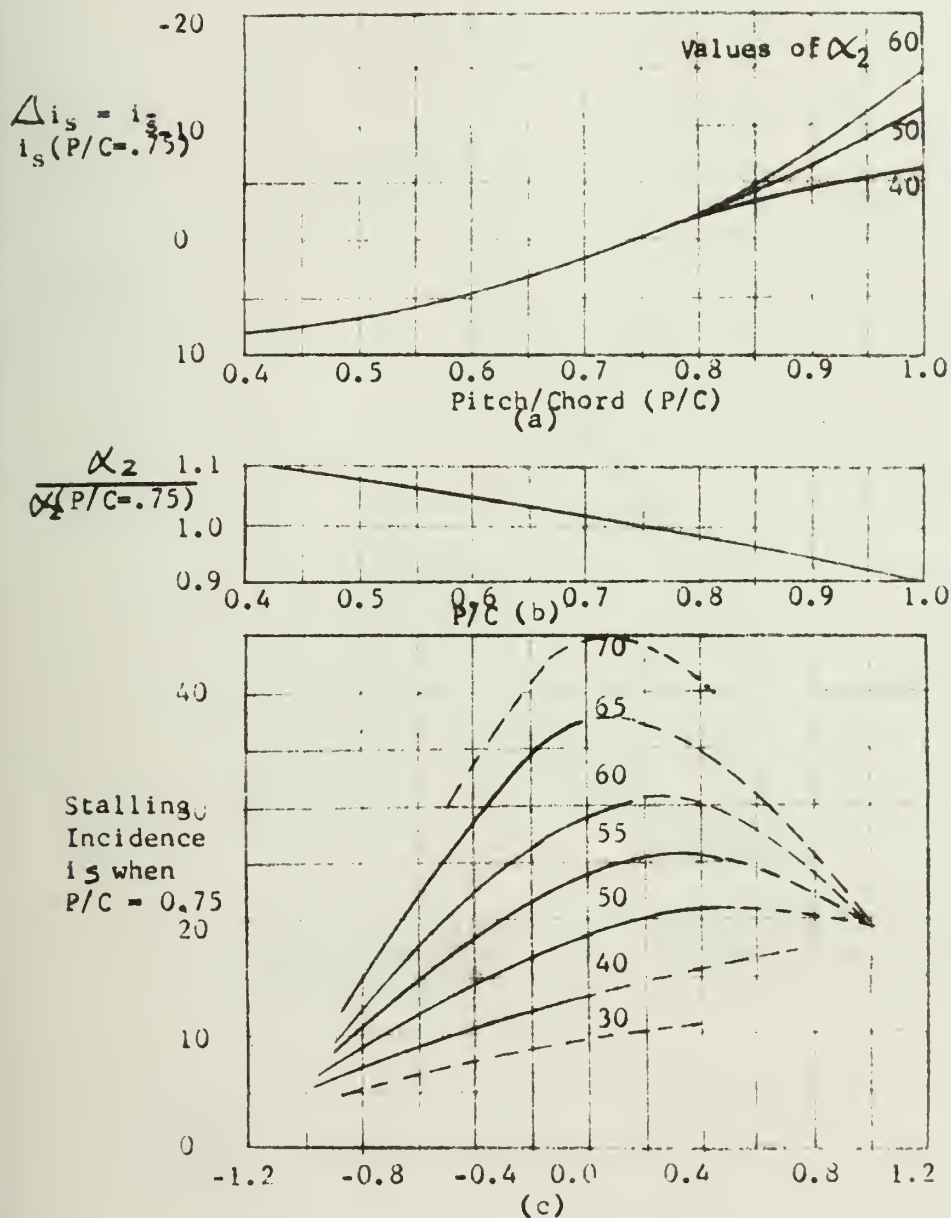


Figure 3-4. Off-design performance of cascades of turbine blades.  $Re = 2 \times 10^5$ . (a) variation of stalling incidence with pitch-chord ratio; (b) variation of  $\alpha_2$  with  $p/c$ ; (c) stalling incidence of turbine blade sections when  $p/c = 0.75$ . (From D. G. Ainley and G. C. R. Mathieson(2))



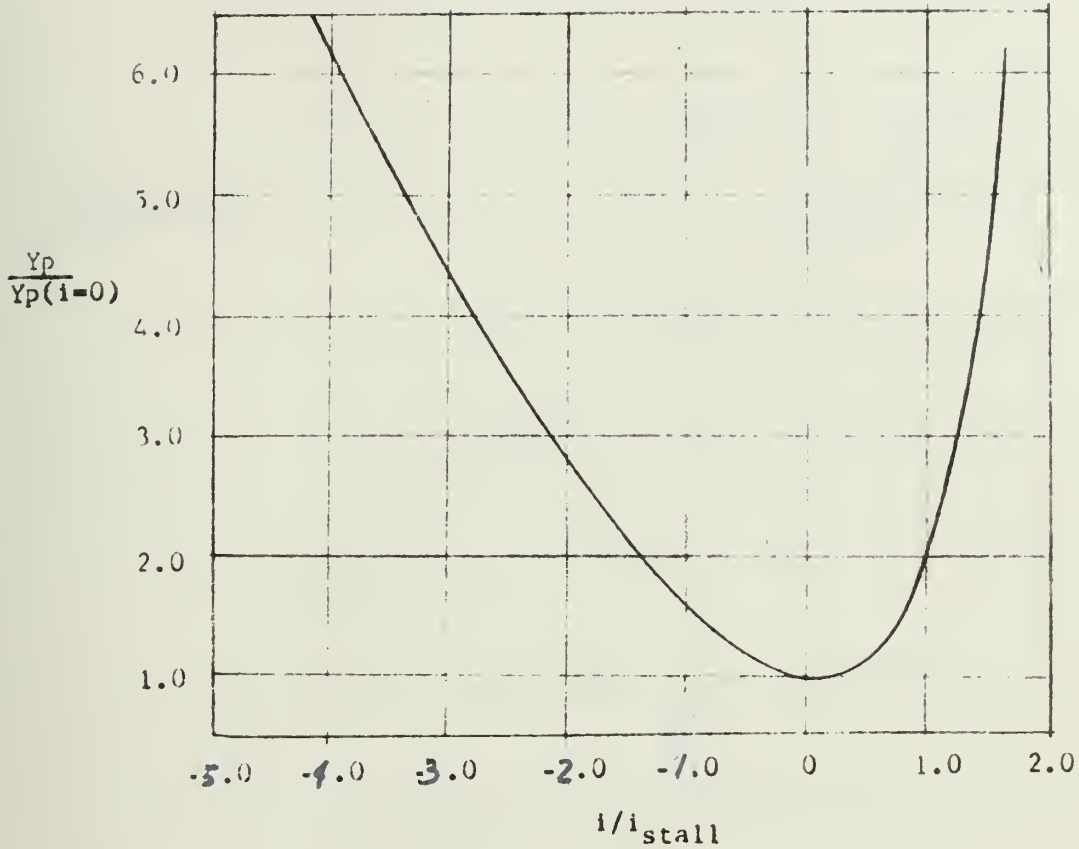


Figure 3-5. Variation of relative profile loss with relative incidence. (From D.G.Ainley and G.C.R. Mathieson(2))

$$\left[ \frac{C_L}{P/C} \right] = 2(\tan \alpha_1 + \tan \alpha_2) \quad (3.2-6a)$$

$$\left[ \frac{C_L}{P/C} \right] = 2(\tan \alpha_2' + \tan \alpha_3') \quad (3.2-6b)$$

For tip clearance losses  $\gamma_k$  is computed by equation (3.2-7) from reference (3).

$$\gamma_k = B(C/H)(k/C)^{.73} \quad (3.2-7)$$



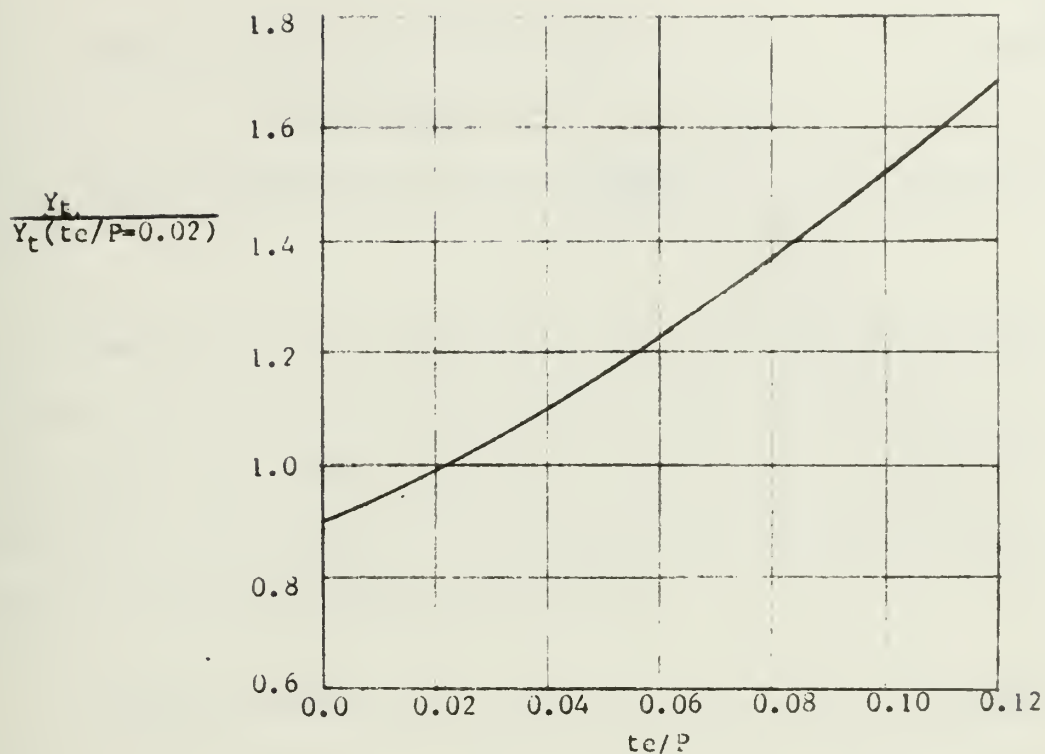


Figure 3-b. Effect of trailing edge thickness on blade coefficients. (From D. J. Ainley and G. C. R. Mathieson(2))

For equation (3.2-7)

$\beta = .37$  for shrouded blades

$\beta = .47$  for plain tip clearance

$\beta = .00$  for stator blades

$k$  = tip clearance

$C$  = chord

$H$  = blade height

Having determined  $Y_p$  and  $Y_s$  they can be corrected for Mach number and Reynold's number from equation (3.2-3) and (3.2-9) respectively.





$$(Y_p) \text{ corrected} = Y_p(1+60(Mn-1)^2) \quad (3.2-8)$$

for  $Mn > 1$

$$(Y_p + Y_s) \text{ corrected} = (Y_p + Y_s) \left[ \frac{Re}{2 \times 10^5} \right]^{-.2} \quad (3.2-9)$$

Now the total loss coefficient  $Y_t$  can be computed by equation (3.2-10). This coefficient is now corrected for thickness of trailing edge to pitch ratio ( $t_e/P$ ) from figure (3-6) to get the total loss coefficient for the blade row. Illustrations of an example calculation can be found in reference (2) and (12).

$$Y_t = Y_p + Y_s + Y_k \quad (3.2-10)$$



3.3 In this section it will be explained how the loss coefficient  $Y_t$  solved for in section 3.2 is coupled with compressible flow tables to estimate performance. The method explained is that outlined in reference (12). Another method is explained in reference (2). Calculations will be carried out only at the mean radius which should give a fairly accurate assessment of the turbine performance.

The calculations are started at the inlet of the turbine and worked through each stage where the exit conditions at one blade row is the inlet to the next blade row until the turbine exit of the last stage is reached.

For inlet conditions the values for  $T_{01}$  and  $P_{01}$  will remain constant, the mass flow and corrected speed  $(N/\sqrt{T_{01}})/(N/\sqrt{T_{01}})_{dp}$  will be varied. This will allow the values of efficiency verses pressure ratio and  $\left(\frac{\dot{m}\sqrt{T_{01}}}{P_{01}}\right) / \left(\frac{\dot{m}\sqrt{T_{01}}}{P_{01}}\right)_{dp}$  verses pressure ratio to be calculated as illustrated in Figure (3-7).

3.3a Flow through the nozzle.

- (1) Select the mass flow.
- (2) Assume an exit Mach number.

(3) With known blade angles subroutine SFOCRA is entered as in section (2.2) only this time the blade angles will be used to determine the exit gas angles.

(4) From compressible flow tables for the assumed Mach number the values of  $(P_2/P_{02})$  and  $(V_2/\sqrt{T_{02}})$  are determined, which will then determine  $V_2$  and  $T_2$

$$V_2 = \sqrt{T_{02}} (V_2/\sqrt{T_{02}}) \quad (3.3-1)$$



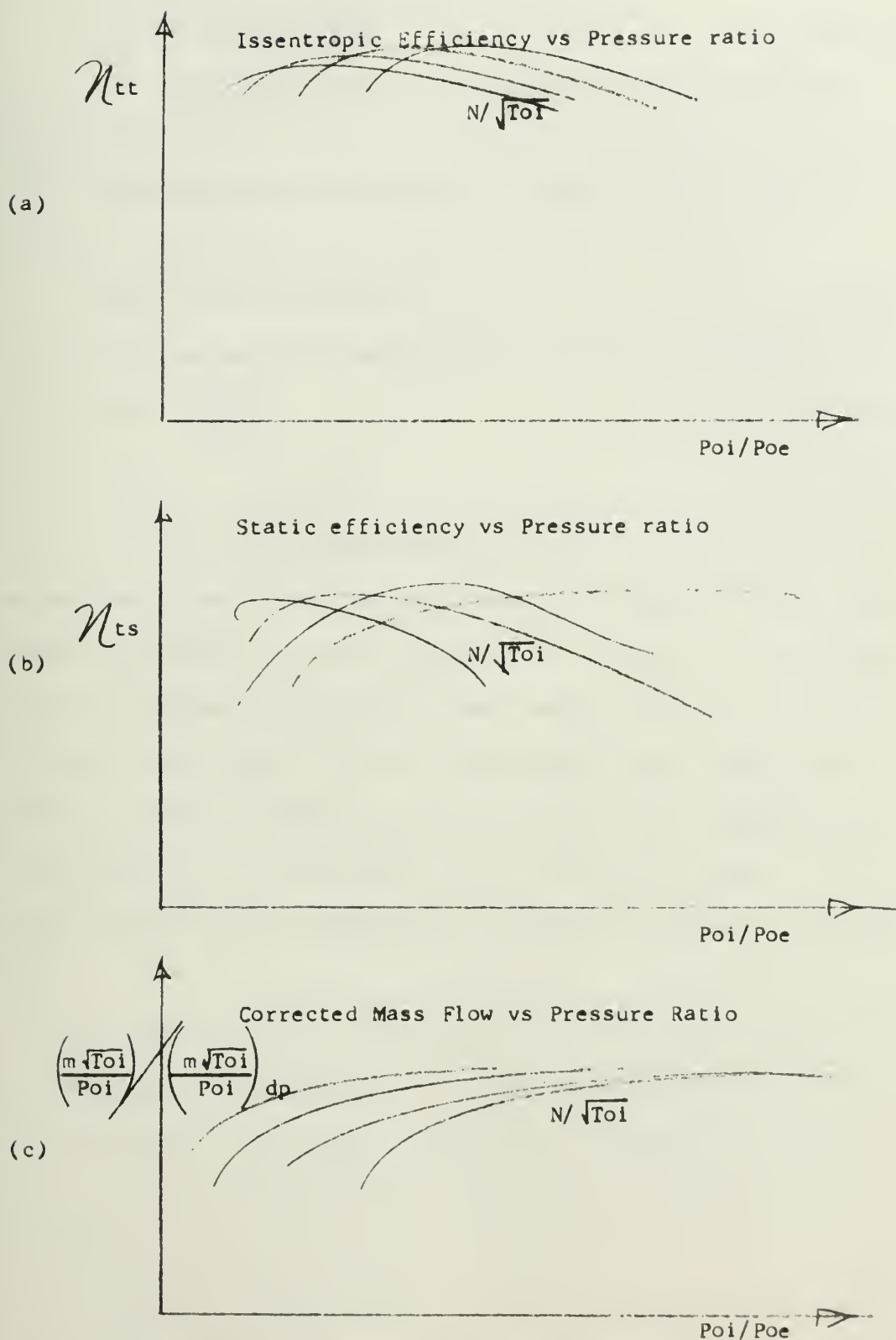


Figure 3-7. Performance Curves for Axial Turbine (a) Isentropic efficiency vs pressure ratio. (b) Static efficiency vs pressure ratio. (c) Corrected mass flow vs pressure ratio.



$$T_2 = T_{02}(P_2/P_{02})^{\frac{\gamma-1}{\gamma}} \quad (3.3-2) \quad 53$$

(5) Subroutine PLOSS is now entered to get  $Y_t$  (pressure loss coefficient).

(6)  $P_{02}$  can be determined from known values of  $Y_t$  and  $(P_2/P_{02})$  from equation (3.3-3)

$$P_{02} = P_{01}/(Y_t(1-P_2/P_{02}) + 1) \quad (3.3-3)$$

(7) The mass flow parameter  $Q_2$  can now be determined.

$$Q_2 = \frac{\dot{m}\sqrt{T_0}}{A_2 P_0} \quad (3.3-4)$$

$$\text{where } A_2 = A_{n2} \cos \alpha_2 \quad (3.3-5)$$

(8) From the compressible flow tables for the value of  $Q_2$  the Mach number can be determined, and if this value is close to the assumed Mach number the inlet conditions to the rotor can be solved for the new values of  $V_2$ ,  $f_2$ ,  $P_2$ , Mach number, and  $T_2$  will be used to iterate the process until the assumed Mach number approximately equals the calculated value. Also as the Mach number approaches (1) a calculation will be performed to see if the row is choked. (ie; maximum flow for given parameters). This will be covered in section 3.3c.

Now that the exit Mach number has been determined the exit conditions of the nozzle can be determined, which will be the inlet conditions to the rotor. Where  $V_2$  is determined from equation (3.3-1),  $T_2$  from equation (3.3-2) and  $P_2$  from compressible flow data.

$$P_2 = P_{02}(P_2/P_{02}) \quad (3.3-6)$$

$$f_2 = \frac{P_2}{RT_2} \quad (3.36a)$$





### 3.3b Flow through the rotor

(1) With  $\alpha_2$  and  $V_2$  known the relative rotor angle  $\beta_2$  can be calculated.

$$\tan \beta_2 = \tan \alpha_2 - \frac{U_{m2}}{V_2 \cos \alpha_2} \quad (3.3-7)$$

$$\text{where } U_m = r_{m2} \omega$$

and determine the incidence angle (i,) where  $\beta'_2$  is the blade angle and  $i = \beta_2 - \beta'_2$

(2) The relative velocity  $w_2$  and relative total temperature, can then be calculated.

$$w_2 = (V_2 \cos \alpha_2)^2 (1 + \tan^2 \beta_2) \quad (3.3-8)$$

$$T_{02\text{rel}} = T_2 + \frac{w_2^2}{2g_o J C_p} \quad (3.3-9)$$

(3)  $w_2 / \sqrt{T_{02\text{rel}}}$  can be calculated, which will be used to enter the gas tables to get  $P_2 / P_{02\text{rel}}$  which will allow  $P_{02\text{rel}}$  to be calculated.

(4) The same procedure of assuming exit Mach number and reiterating until the assumed Mach number equals the calculated one is followed.  $P_{03\text{rel}}, \beta_3, w_3, M_{3\text{rel}},$  and  $Q_{3\text{rel}}$  will then be known. From this relative Mach number ( $P_3 / P_{03\text{rel}}$ ) can be determined, which will enable  $P_3$  to be solved for. The absolute parameters  $\alpha_3, V_3, P_{03}, T_3$  and  $T_{03}$  can then be calculated.

$$\tan \alpha_3 = \tan \beta_3 + \frac{U_m}{w_3 \cos \beta_3} \quad (3.3-10)$$

$$\text{where } U_m = r_{m3} \omega$$



$$v_3 = (w_3 \cos \beta_3)^2 (1 + \tan^2 \alpha_3) \quad (3.3-11)$$

$$T_3 = T_{03rel} - \frac{w^2}{2g_0 J C_p} \quad (3.3-12)$$

$$P_{03} = P_3 (T_{03}/T_3)^{\frac{\gamma}{\gamma-1}} \quad (3.3-13)$$

(5) If this is the last stage, efficiency  $\eta$  can be calculated by equations (3.3-14) and (3.3-15). If not the exit conditions will determine the inlet conditions to the nozzle of the next stage. This process will be iterated until the parameters have been determined for every stage in the turbine..

$$\eta_{tt} = \frac{(T_{0i} - T_{0e})}{T_{0i} \left[ 1 - (P_{0e}/P_{0i})^{\frac{\gamma-1}{\gamma}} \right]} \quad (3.3-14)$$

$$\eta_{ts} = \frac{(T_{0i} - T_{0e})}{T_{0i} \left[ 1 - (P_e/P_{0i})^{\frac{\gamma-1}{\gamma}} \right]} \quad (3.3-15)$$

3.3c Determination of whether a row of turbine blades have choked is carried out when the Mach number approaches (1). For the computer program this process will be carried out when the Mach number is greater than 0.9.

(1) Select a slightly higher value of  $Q \left( \frac{\dot{m} \sqrt{T_0}}{A P_0} \right)$  calling it  $Q'$ .

(2) With this  $Q'$  enter the compressible flow tables to get corresponding values of  $P/P_0$  and solve for  $P_{0in}/P_{0out}$  assuming  $Y_t$  remains constant.



(3) Now calculate the new mass flow  $\dot{m}'$ . If this is less than  $\dot{m}$  the flow has choked.  $Y_t$  and  $Q$  will be assumed to remain constant with the flow going supersonic at the exit of the blade row. If  $\dot{m}'$  is not less than  $\dot{m}$  calculations will be resumed as was illustrated in section (3.3a) and (3.3b).

(4) Enter compressible flow tables with  $Q$  for supersonic flow to get corresponding values of  $(P_{out}/P_{o_{out}})$ ,  $V_{out} / \sqrt{T_{o_{out}}}$ , and  $M_{out}$ . From equation (3.3-3) determine  $P_{out}$ . With equation (3.3-4)  $A_{out}$  can be determined. Since  $A_{out} = A_{annulus} \cos \alpha$ ,  $\alpha$  can be solved for in equation (3.3-16). The calculations are then resumed in the same manner as in section (3.3a) and (3.3b). The validity of this method depends on whether  $Q$  and  $Y_t$  remain constant after the turbine blade row has choked.

$$\alpha = \cos^{-1} \left( \frac{\dot{m} \sqrt{T_{o_{out}}}}{A_{annulus} P_{oQ}} \right) \quad (3.3-16)$$



## Chapter 4

### Operation of Computer Program

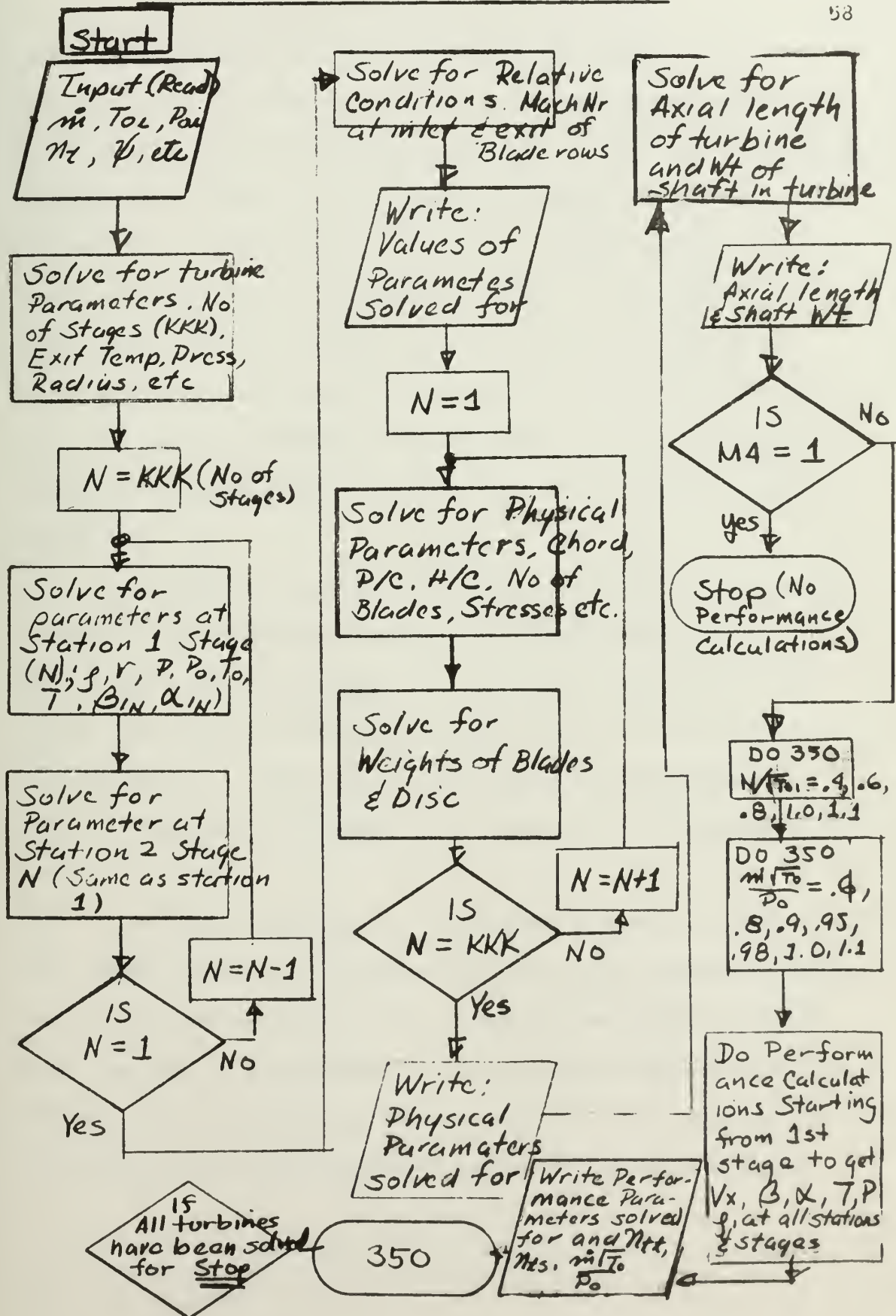
4.1 Chapters 1, 2, and 3 described how the calculations are performed in the computer program. This chapter will show how the flow of information is passed through the computer program to get the desired results. This information is not needed to use the computer program since that is covered in Appendix (2).

4.2 The steps of calculations in the computer are in the same sequential order as illustrated in chapters 1 through 3. This is shown in the simplified block diagram figure (4-1)

4.3 The method used in the computer program to interpolate curves is a 4 point Lagrange Polynomials with a worse case of a linear interpolation due to break down of Lagrange Polynomials for monatomic increasing or decreasing function. Subroutine FIG and BK are used to do this interpolating. Subroutine FIG selects the 4 data points closest to the argument X for passing to subroutine BK which actually does the interpolating to get the ordinate Y as an output. If the argument exceeds the limits of the data points the value closest to the argument X is used to get the ordinate Y. The reason this method was chosen over finding formulas by least squares or similar methods is this method can be adapted to changes in data without solving for new formulas or coefficients. The Lagrange Polynomials uses a monatomicly increasing base to get the ordinate Y. If the base is not monatomicly increasing such as









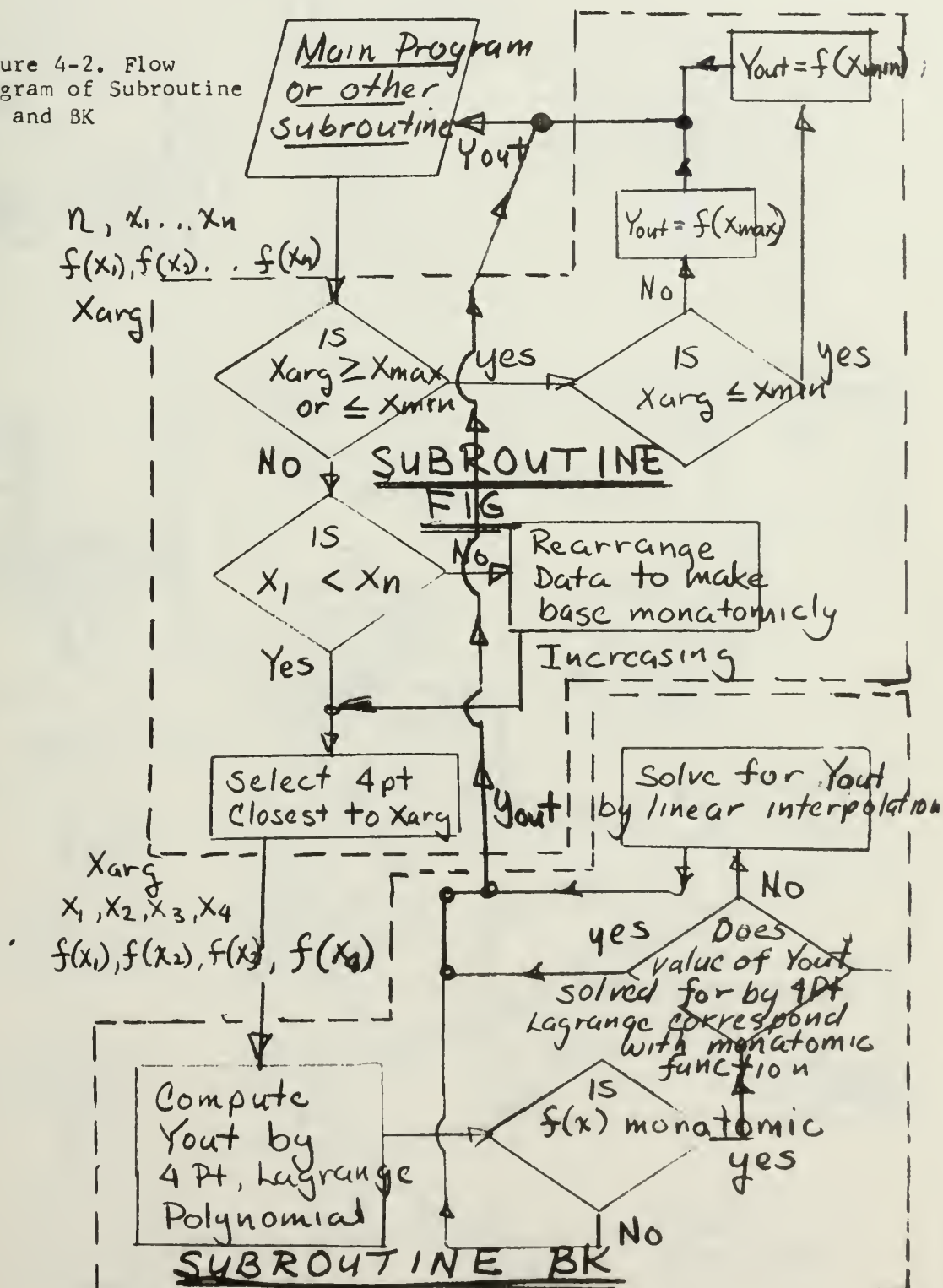
values of Mach Number vs  $P_s/P_o$ , subroutine FIG rearranges the base so that the base is monatomicly increasing, and the ordinate values are in corresponding relationship with the base. For values of base such as  $(\dot{m}\sqrt{T_{oi}})/A_{Poi}$  which both increase and decrease the data points are divided into two parts with the part that increases as one set of data points and the part that decreases as another. The ordinate does not have to be a monatomic function. The problem of increasing and decreasing bases can be encountered in the performance estimations because the base and ordinate are interchanged in some of the steps. Figure 4-2 illustrates how subroutine FIG and 3K are used to interpolate data.

4.4 Figure (4-3) illustrates the method the computer program uses to determine the flow area at station 1. In most cases for a value of epsilon ( $\epsilon$ ) = .001 the convergence is reached in less than 10 iterations. This method is used for solving for the exit flow area and also for station 2. The method for solving the other parameters were will documented in chapters 1 thru 3 and will not be repeated here.

4.5 From output generated by the computer program, it is felt that the computer program can be very useful from early preliminary design to start of detailed design, since it has the versatility to accept more input data as the designer firmly sets the characteristics of the Turbine. As for cooling requirements it is felt that compensations in the input data could account for losses attributed to cooling.



Figure 4-2. Flow  
Diagram of Subroutine  
FIG and BK

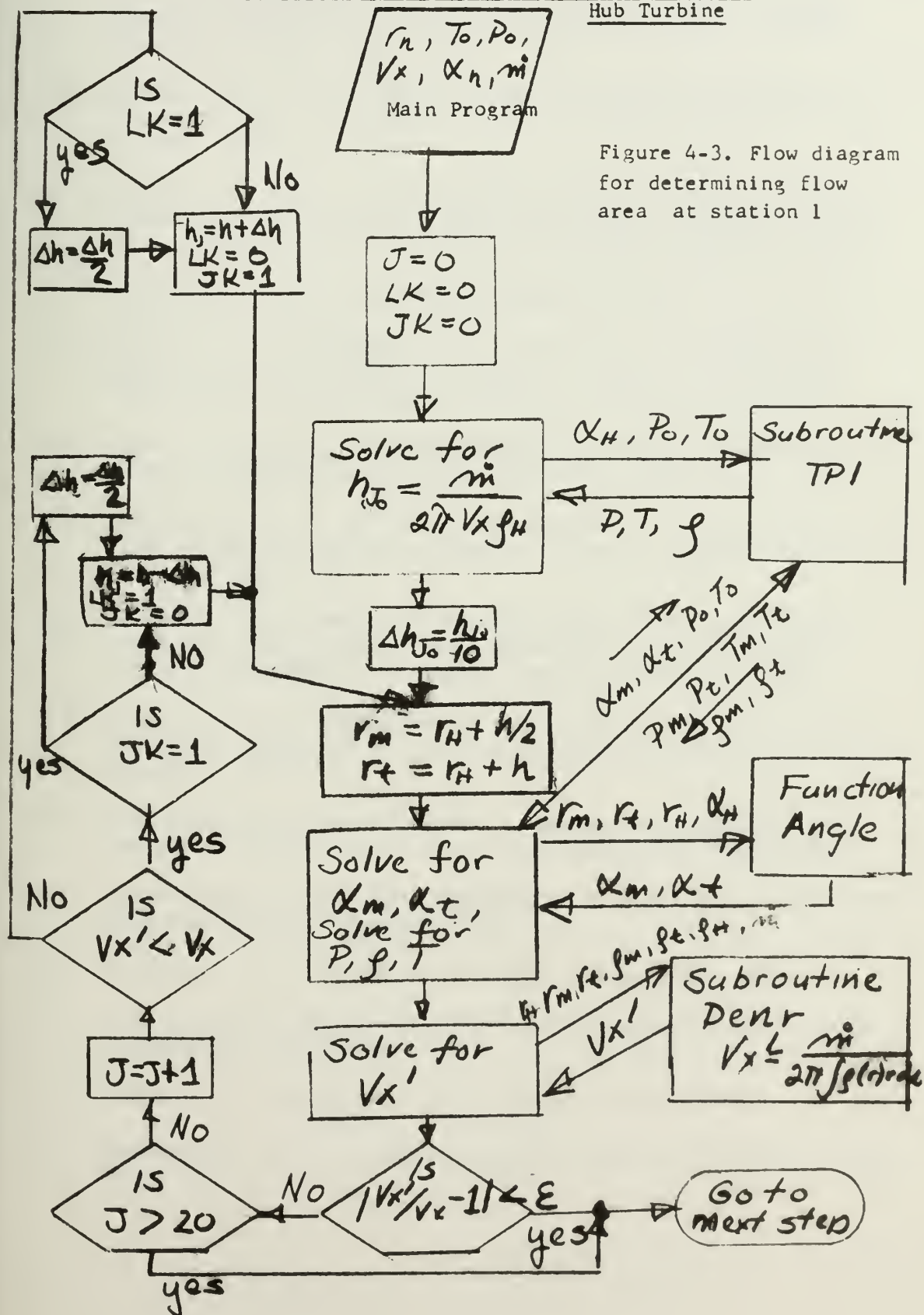






Block Diagram of Adjusting Flow Area To Match  
Axial Velocity  $V_x$  at Station 1 For Constant  
Hub Turbine

61







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## APPENDIX I

Symbols and Notation used in this Text.



## SYMBOLS USED IN TEXT

A	Area
C	Chord of turbine blade
C <sub>p</sub>	Specific heat (btu/lbm <sup>°</sup> R)
g <sub>0</sub>	Constant in Newtons Law 32.174 (ftlbm)/(lbfsec <sup>2</sup> )
h	Blade height
h <sub>0</sub>	Stagnation Enthalpy
H/C	Aspect ratio
i	Incidence angle
I	Moment of inertia
J	Energy conversion factor 778.16 (ftlbf)/(btu)
k	radius of gyration
M	Mach Number
M <sub>crit</sub>	$V_x / \left( \frac{2}{\gamma+1} g_0 R T_0 \right)^{1/2}$
$\dot{m}$	Mass Flow
N	Number of Stages
n	Number of Stage
P <sub>0</sub>	Stagnation Pressure
P	Static Pressure
P/C	Pitch to Chord ratio
R	Gas constant (ftlbf)/(lbm <sup>°</sup> R) $\frac{\gamma-1}{\gamma} J C_p$
Q	Corrected mass flow $\dot{m} \sqrt{T_0 / P_0 A}$
T <sub>0</sub>	Stagnation Temperature
T	Static Temperature





To	Thickness of Outer Rim
Ti	Thickness of Inner Rim
SM	Non Dimensional Section Modulus
U	Blade Speed
V	Velocity
Vx	Axial Velocity
W	Relative Velocity
Wo	Width of Outer Disc Rim
Wi	Width of Inner Disc Rim
Y	Pressure Loss
R	Radius
r	Radius
t	Thickness of Turbine Blade
t/C	Thickness to Chord Ratio
Z	Section Modulus
Zi	Thickness of Mid Section of Turbine Disc at Inner Radius
Zo	Thickness of Mid Section of Turbine Disc at Outer Radius
Z(r)	Thickness of Disc as a Function of Radius
$\alpha$	Absolute Gas Angles (except section 3.2)
$\beta$	Relative Gas Angles (except section 3.2)
$\epsilon$	Stagger Angle
$\gamma$	Ratio of Specific Heats Gamma
$\eta$	Efficiency
$\sigma$	Stress
$\rho$	Density
$\psi$	Psi Loading Coefficient $\equiv \frac{g_o J \Delta h_o}{U_h^2}$



$Q$	Reaction
$\Omega$	Speed (rpm)
$\omega$	Speed (rad/sec)

## Subscripts

i	Inlet
e	Exit
t	Total
s	Stage
p	Polytropic
s	Static
t	Tip
m	Mean
h	Hub
c	constant



## APPENDIX II

User's Guide To the Computer Program with Comparison of  
Turbine Designed with this Computer Program to  
one which was Designed by Airesearch. Sample  
Input and Output of Turbine  
Designed by Computer  
Program



II-1 The computer program can be used to calculate physical and thermodynamic properties of an axial flow turbine.. It has an optional ability to do performance calculations. The optional phase is very expensive to operate since it does both design and off design calculations. From operating the computer program and comparing the cycle calculations with performance calculations the cycle calculations are within 3% of performance calculations at design point. For the early preliminary phases of turbine design it is recommended that just the cycle calculations and mechanical design phases be carried out with the computer program punching output cards to have data to perform the performance calculations at a later time

A second program which reads in the punched output cards in the same order as they were punched in the first program is provided to do only performance calculations. This allows the performance estimations to be carried out with running the complete program again.

For future references program #1 will refer to the complete computer program while program #2 will refer to the program which only does performance estimations. The complete computer program can not be operated in computers which have less than 210 k of memory.

II-2 Pages 71 to 76 show how the computer cards are to be punched. The 1st card is a label which will be printed out in the same content as it was entered on the computer card.. Cards 2 thru 4 are the input data cards of known turbine parameters. Cards 5 thru NNN are the compressible flow tables and viscosity of the gases with corresponding temperatures. Starting from card 5, 1st the Mach Numbers are read in , 2nd the corrected mass flow  $\dot{m} \sqrt{T_0/P_0 A}$ , 3rd the pressure ratio  $P/P_0$ ,





4th the corrected velocity  $V/\sqrt{T_0}$ , 5th the temperatures which correspond to the viscosity data, 6th the viscosity data. Page 77 is a set of compressible flow tables and viscosity data for air.

II-3 Pages 73 thru 82 are comparison data of computer program designed turbine with one which was designed by Airesearch reference (20). As can be seen from the data some of the parameters are different because of lack of knowledge of some of the design parameters which was used in the design of the Airesearch Turbine, also some simplifications are incorporated into the computer program for solutions of output parameters.

II-4 A list of sample output and input is provided from page 83 to 93 for turbine which was compared with Airesearch turbine. As can be seen on page 91 of output data, the performance data is very similar to the cycle calculations at design point. This justifies the argument of doing only cycle calculations and mechanical calculations when off design data is not required. Figure II-3 is a plot of the data points from performance calculations. Where some of the curves are discontinuous is caused by a limit of 10 iterations for solution at a blade row..



Input Data Cards

\* (Must be specified)

# (If value of parameter is not known place value to right of #  
on computer card)

Card 1

Column 1-80                      Alphanumeric label (ie, "Gas Tur-  
bine number 1")  
# Blank Card

Card 2

Column 1\*                      M1 (Integer)  
0   compressible flow data to be  
entered.  
1   No compressible flow data to be  
entered.

Column 2 \*                      M2 (Integer).  
0   Power in horse power specified.  
1   Pressure ratio specified.

Column 3 and 4                  Blank

Column 5 \*                      M (Integer)   Type of turbine.  
1   Constant hub  
2   Constant mean  
3   Constant tip

Note      Where instructions inform user to place dummy values on  
cards these values are used to inform computer that they  
are not specified and to perform optional calculations to  
determine the parameter.



Column 6-10 (leave blank if M1 = 1)	KLM (Integer right justified) Number of compressible flow elements to be entered. (ie number of Mach Numbers to be entered) (Maximum 100)
Column 11-15 (leave blank if M1 = 1)	KTM (Integer right justified) Number of Temperatures at which values of (viscosity) $\times 10^7 \frac{\text{lbm}}{\text{sec ft.}}$ is defined.
Column 16-20 *	Gamma $\gamma$ (F5.3)
Column 21-25 *	CP (3tu/lbm °R) (F5.3)
Column 26-35 (lbm/sec ft)	Power (Horse Power) (F10.1) if M2 = 0 this must be specified # 9000.
Column 36-45 *	W (mass flow) (lbm/sec) (F 10.3)
Column 46-50 *	TC (thickness to chord ratio) (F5.3), normally about # 0.2
Column 51-55 *	TET (thickness of trailing edge to pitch ratio) (F5.3) normally about # 0.02
Column 56-60 *	TIPCLA (Tip clearance) (in) (F5.3) normally about # 0.015



Card 3

Column 1-5 \*

 $\rho$  PSI (F5.3)

Normally between 1.5 and 2.9

# 2.0

Column 6-10 \*

RTE (Max Tip radius in ft at exit)

(F5.3)

Column 11-15 \*

Ratio T ( $R_{te}/R_{he}$ ) Ratio of Tip to  
hub at exit (F5.3) # 1.5

Column 16-20

 $R_{hub}$  (exit) Reaction at hub (F5.3)

normally between .00 to 0.4 # .05

Column 21-30 \*

OMEGA (RPM) (F 10.0)

Column 31-32

KLM1 (INTEGER) Right justified if

Compressible flow data provided

the value at which  $\frac{\dot{m} \sqrt{T_{0c}}}{P_{0c} A}$ 

is a maximum in (for

sample problem KLM1 = 50) (Maximum 70)

Column 33-39

Blank

Column 40 \*

M4 (Integer)

0 performance calculations to be performed.

1 No Performance Calculations to be performed.

Column 41-45 \*

SPECWT (Specific Weight of turbine  
disc and blades (lb/in<sup>3</sup>) (F5.3)

# normally about 0.3





Column 46-50 \*

Y MOD (Young's Modulus psi) (f 15.0)  
Normally about # 30000000.

Column 61-70 \*

C1 (Value of allowable blade stress  
in psi) (F 10.0) Normally about  
# 50000.

Column 71-80 \*

C2 (Value of allowable blade steady  
stress in PSI) (F 10.0) Normally  
about # 100000.

Card 4

Column 1-5 \*

ENT (Value of total efficiency  
desired. # .9

Column 6-15

Radius of Shaft in ft (F 10.5) # - 0.1  
if not known

Column 16-20

P ratio (Pressure Ratio) (F5.3)  
if M2 = 1 this must be specified # 1.5

Column 21-30 \*

ULTSRE (Ultimate Tensile Strength of  
Material of Disc (psi) Value for disc  
stress is calculated from this. (F10.0)

$$\sigma_{avg} = \frac{.75 \text{ ULTSRE}}{2} \quad (1.2)$$

Normally about # 130000.

Column 31-35

ALPHAH (Exit angle in radians) (F5.3)  
Normally between (0.0 to - .3) # - .1

Column 36-45

EMACH =  $\frac{V_x}{\sqrt{\frac{2}{\gamma+1} g_o R T_o}}$  Normally Between  
.2-1.0 #-.5



0 Normal Blades

1 Shrouded Rotor Blades. #0

Column 47-48 \*

NKL (Number of turbines to be designed by computer program. (Integer right justified) All sets of cards except for last turbine design will have 0 in column 48. The last set of cards will have the number of sets that were entered. This allows the computer to stop after completing the last turbine design.

Column 49-50

N (Number of stage Integer right justified) (Maximum 10) If this value is not known, enter 0 and computer will solve for number of stages. #0

Column 51-55 \*

AREARA ( F5.4)

Tip to base area ratio of turbine blade normally between .25 to .35

# .25

Column 56-60

T00 (Gives radial thickness of outer rim of disc. by  $T_0 = W_0/T00$  normally between 2 to 4 (F5.3) # - 5.



## Cards 5-NNN

These cards are optional. If performance calculations are to be performed they must be furnished. If more than one turbine is to be designed compressible flow data must only be furnished for 1st turbine and the value of M1 on card 2 for each following turbines will be "1". If the compressible flow parameters are different a set must be furnished. A set of compressible flow data and viscosity data for air are provided with the computer program.

Card 5 - N	RMAC (Mach Numbers) (20F4.2)  same number as KLM
Card (N + 1)-K	WTAP (10F8.5) $\frac{\dot{m}\sqrt{T_o}}{P_o A}$ $\frac{1 \text{bm}^{\circ} R_1}{\text{sec lbf}}$ Same numbers as KLM
Card (K+1) -J	PSPT (Ps/Po) (10F8.5) same  number as KLM
Card (J+1)-L	VELTOT (V/ $\sqrt{T_o}$ ) (10F8.5) $\frac{(\text{ft/sec})}{\sqrt{T_o R}}$ same number as KLM
Card (L + 1) - LL	TEM ( $^{\circ}R$ ) (20F5.0) same number as KIM temperature values that cor- respond with viscosity data
Card (LL + 1)-NNN	V (Viscosity of fluid (lbm.sec/ft) $\times$ $10^7$ ) (20F5.0)



## Compressible Flow Tables for Combustion Products

 $\frac{1 \text{ lbm}}{\text{sec}} \sqrt{\frac{10^6}{R}}$   $(\text{ft/sec}) \sqrt{\frac{10^6}{R}}$  reference (2)

MACH NO	W*SQRT(T)/(A*P)	PS/PT	V/SQRT(T)
0.02000	0.01811	0.99973	0.96151
0.04000	0.03596	0.99893	1.91184
0.06000	0.05388	0.99760	2.86589
0.08000	0.07171	0.99574	3.82740
0.10000	0.08945	0.99335	4.77028
0.12000	0.10706	0.99040	5.72284
0.14000	0.12448	0.98700	6.67094
0.16000	0.14154	0.98317	7.61008
0.17000	0.15885	0.97870	8.56787
0.20000	0.17562	0.97380	9.50329
0.22000	0.19226	0.96835	10.45287
0.24000	0.20849	0.96260	11.36668
0.26000	0.22446	0.95630	12.31328
0.28000	0.24049	0.94950	13.26734
0.30000	0.25612	0.94200	14.22884
0.32000	0.27135	0.93440	15.14564
0.34000	0.28590	0.92650	16.04750
0.36000	0.30026	0.91800	16.99411
0.38000	0.31435	0.90920	17.90344
0.40000	0.32763	0.90030	18.82024
0.42000	0.34118	0.89080	19.75192
0.44000	0.35386	0.88100	20.63144
0.46000	0.36627	0.87060	21.54077
0.48000	0.37821	0.86000	22.45010
0.50000	0.38948	0.84950	23.33708
0.52000	0.40061	0.83860	24.27625
0.54000	0.41108	0.82720	25.16321
0.56000	0.42114	0.81570	26.05019
0.58000	0.43067	0.80400	26.95207
0.60000	0.43979	0.79220	27.80922
0.62000	0.44838	0.78020	28.65892
0.64000	0.45669	0.76780	29.54590
0.66000	0.46421	0.75550	30.41052
0.68000	0.47159	0.74270	31.29750
0.70000	0.47796	0.73070	32.09502
0.72000	0.48433	0.71800	32.94472
0.74000	0.49010	0.70520	33.80190
0.76000	0.49560	0.69140	34.65160
0.78000	0.50030	0.67910	35.47894
0.80000	0.50459	0.66650	36.29883
0.82000	0.50815	0.65500	37.05164
0.84000	0.51170	0.64080	37.93861
0.86000	0.51479	0.62760	38.76595
0.88000	0.51734	0.61410	39.60822
0.90000	0.51948	0.60250	40.39828
0.92000	0.52123	0.58900	41.14365
0.94000	0.52257	0.57550	42.03061
0.96000	0.52337	0.56500	42.70889
0.98000	0.52391	0.55100	43.41698
1.00000	0.52418	0.54000	44.16234
1.02000	0.52391	0.52800	44.98222
1.04000	0.52337	0.51700	45.69031
1.06000	0.52257	0.50400	46.36113
1.08000	0.52136	0.49100	47.17357
1.10000	0.51989	0.48000	47.85184
1.12000	0.51801	0.46820	48.59720
1.14000	0.51599	0.45650	49.34256
1.16000	0.51358	0.44530	50.02827
1.18000	0.51076	0.43330	50.74382
1.20000	0.50795	0.42250	51.39229

for  $\gamma = 1.333$ 

$$g = 37.174 \frac{\text{ft}}{\text{sec}}$$

$$C_p = 0.274 \frac{\text{Btu}}{\text{lbm}^\circ\text{R}}$$

Viscosity of  
Air from Reference (14)
 Temperature  $(^\circ\text{R})$       Viscosity  $\times 10^7 \frac{\text{lbm}}{\text{sec ft}}$ 

400	100
600	135
800	166
1000	192
1200	218
1400	242
1600	264
1800	284
2000	302
2200	320
2400	338





COMPARISON OF COMPUTER PROGRAM TURBINE DESIGN WITH  
AIRESEARCH 9000 HP FREE VORTEX AXIAL FLOW TURBINE

78

Inlet Conditions

Gas	Air
Poi(psi)	312.5
Toi( $^{\circ}$ R)	1767.8
m(lb/sec)	119.421

Design Parameters

$\psi$	1.8
$\alpha_e$	0.0

Assumed Design Parameters

$\eta_{tt}$	0.9168
$C_p(\text{btu/lb}^{\circ}\text{R})$	.2715
Gamma $\gamma$	1.333

Results

Figure II-1 is a comparison of the turbine wheel dimensions.

Figure II-2 is a comparison of design point performance calculations of total and static efficiencies as a function of tip clearance.



COMPARISON OF OUTPUT DATA FROM COMPUTER PROGRAM WITH  
AIRESEARCH GAS TURBINE

Parameter	Airesearch	Computer Program
Toe( $^{\circ}$ R)	1571.4	1554.0
$M_{crit}$ at exit	-	.292
Poe (psi)	186.0	177.3

Velocity Diagrams

hub(station 1)

radius(ft)	.4383	.429
$\beta_1$ (deg)	-62.02	-63.96
$\alpha_1$ (deg)	0.0	0.0

tip(station 1)

radius(ft)	.530	.429
$\beta_1$ (deg)	-67.15	-70.93
$\alpha_1$ (deg)	0.0	0.0

Hub(station 3)

radius(ft)	.4383	.429
$\beta_3$ (deg)	-60.95	-63.96
$\alpha_3$ (deg)	0.0	0.0

Tip(station 3)

radius(ft)	.6215	.636
$\beta_3$ (deg)	-68.26	-72.99
$\alpha_3$ (deg)	0.0	0.0
$\delta_{cf}$ (psi)	37,400.	37,396.

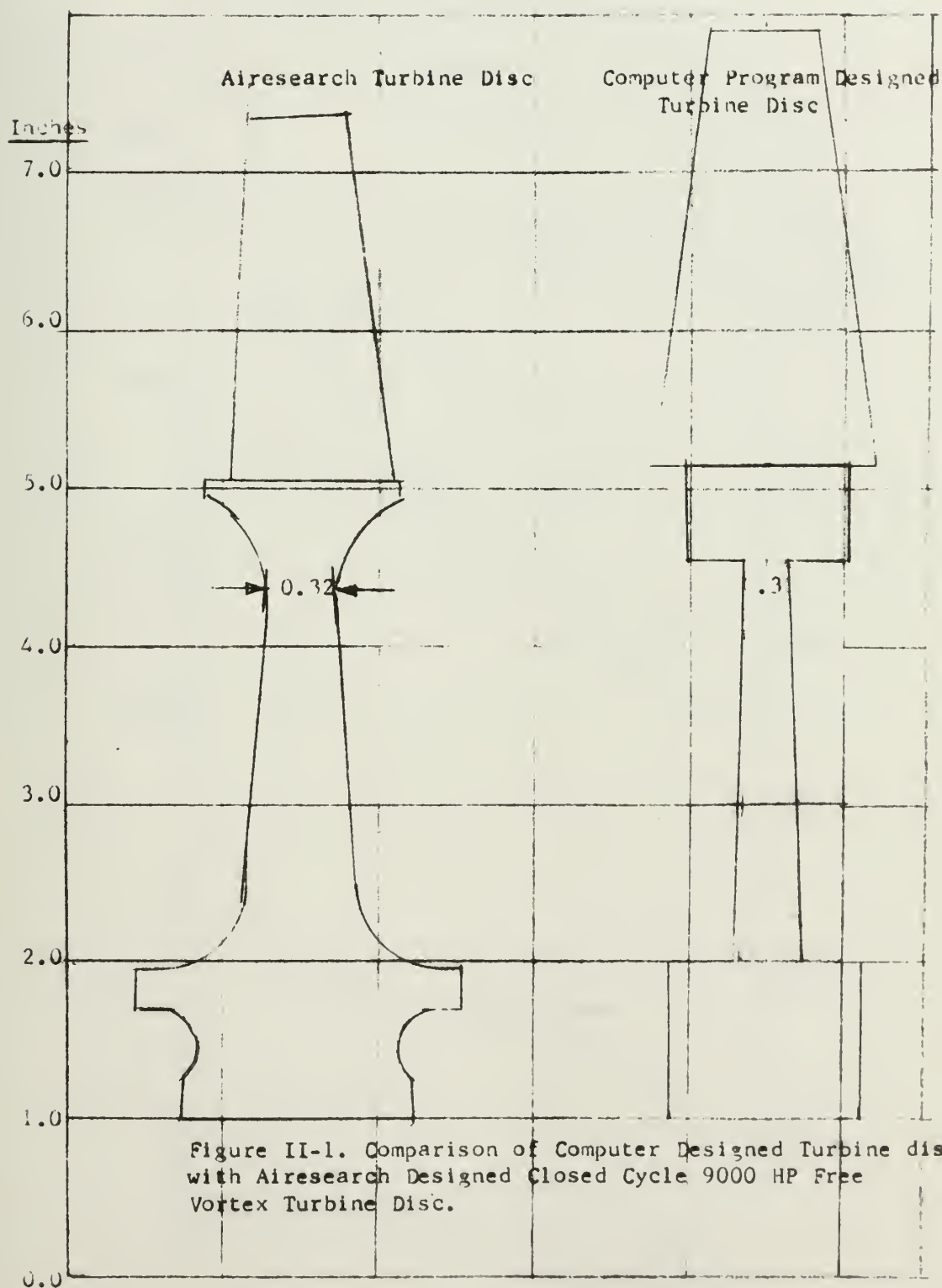
Wheel Weight(lbf)	18.7	21.46
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### Conclusions:

Most of the discrepancies in parameter values can be attributed to different input values for the computer model. This is caused by not knowing the assumptions or definitions that were used in designing the Airsearch Gas Turbine. From this limited comparison it is felt that the computer model can be used for designing marine gas turbines since it has the versatility to change parameters which are attributed to specific turbines. For high performance turbines (aircraft) it should be compared with known aircraft turbines to verify the validity of the computer program and adjust the variable parameters to match them.









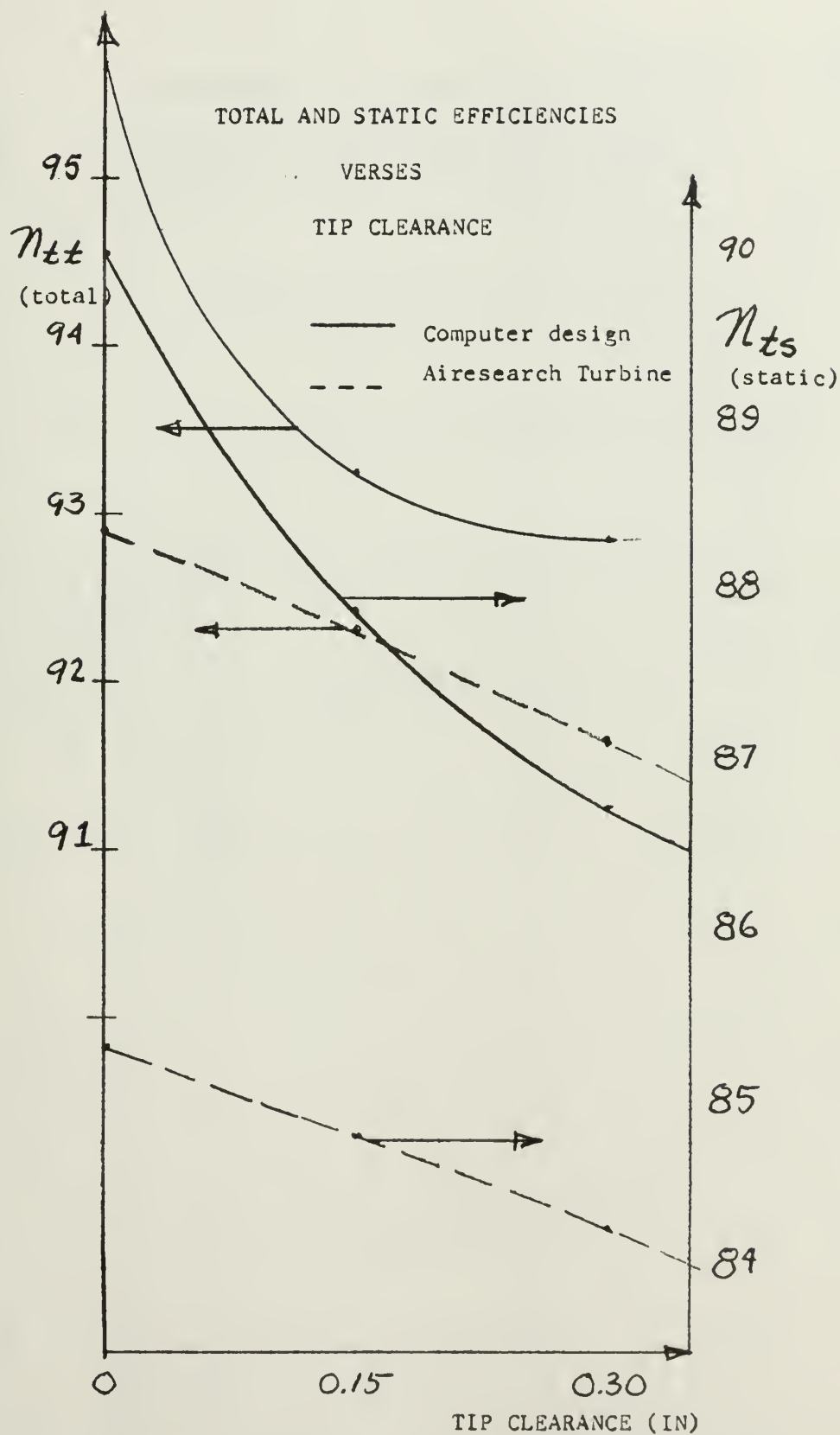


Figure II-2, Comparison of Efficiencies verses Tip Clearance of Performance Calculations.



Input Data Cards for Run 9000 HP turbine with Performance Calculations  
 Calculations and 13673 HP turbine with Performance Calculations

Card

\$ENTRY 4 Control Card

Turbine #1

9000 HP SINGLE STAGE GAS TURBINE ENGINE

00 1 60 111.333.2715000. 119.42 .2 .02 .015 1767.8 312.48

1.8 .62141.414.1 20000. 00 0.25 29000000. 20000. 93000.

.916.04333 1.640130000. .000 .292 0 0 1.25 .1

0.020.040.060.080.100.120.140.160.170.200.220.240.260.280.300.320.340.360.380.400

M 0.420.440.460.480.500.520.540.560.580.600.620.640.660.680.700.720.740.760.780.800

0.820.840.860.880.900.920.940.960.980.1000.1020.1040.1060.1080.1100.1120.1140.1160.1180.1200

0.01811 0.03596 0.05348 0.07171 0.08945 0.10706 0.12448 0.14154 0.15885 0.17562

0.19226 0.20849 0.22446 0.24049 0.25612 0.27135 0.28590 0.30026 0.31435 0.32763

0.34118 0.35386 0.36627 0.37821 0.38948 0.40061 0.41108 0.42114 0.43067 0.43979

0.44838 0.45569 0.46421 0.47129 0.47796 0.48433 0.49010 0.49560 0.50030 0.50450

0.50815 0.51170 0.51479 0.51734 0.51948 0.52123 0.52257 0.52337 0.52391 0.52412

0.52391 0.52337 0.52257 0.52136 0.51989 0.51801 0.51599 0.51358 0.51076 0.50795

0.99973 0.99893 0.99760 0.99574 0.99335 0.99040 0.98700 0.98317 0.97870 0.97380

0.96835 0.96260 0.95630 0.94930 0.94200 0.93440 0.92650 0.91800 0.90920 0.90030

0.89080 0.88100 0.87050 0.85900 0.84650 0.83300 0.81800 0.80400 0.79220 0.77920

0.74020 0.76780 0.75550 0.74270 0.73070 0.71800 0.70520 0.69140 0.67910 0.66650

0.65500 0.64060 0.62760 0.61410 0.60250 0.58900 0.57550 0.56500 0.55100 0.54000

0.52800 0.51700 0.50400 0.49100 0.48000 0.46820 0.45650 0.44530 0.43330 0.42250

0.96151 1.91184 2.85589 3.82700 4.77028 5.72284 6.67094 7.61008 8.56787 9.50320

10.45287 11.36668 12.31321 13.26734 14.22884 15.14504 16.04750 16.99411 17.90344 18.82020

19.75192 20.63144 21.54077 22.45010 23.37042 24.27625 25.16321 26.05019 26.95207 27.80922

28.65892 29.54590 30.41052 31.24703 32.04502 32.74472 33.80190 34.65160 35.47894 36.29083

37.05164 37.93861 38.75535 39.50422 40.39823 41.14365 42.03061 42.70889 43.41698 44.16234

44.98222 45.69031 46.35113 47.17327 47.85184 48.59720 49.34256 50.02827 50.74382 51.39220

T 400. 600. 800. 1000. 1200. 1400. 1600. 1800. 2000. 2200. 2400. Temperature

$\mu$  100. 135. 165. 192. 214. 242. 264. 284. 302. 320. 338. Viscosity  $\times 10^7$

13673 HP SINGLE STAGE AIRCRAFT GAS TURBINE ENGINE Turbine #2

10 2 60 111.333.274 13513. 100. .2 .02 .015 2960. 88.2

2.8 1.034 2.0 0.0 13555. 00 2.25 29000000. 45000. 93000.

0.9 .1 6. 130000. -.5 -.25 0 2 1.25 .1

Sample Input Data Cards



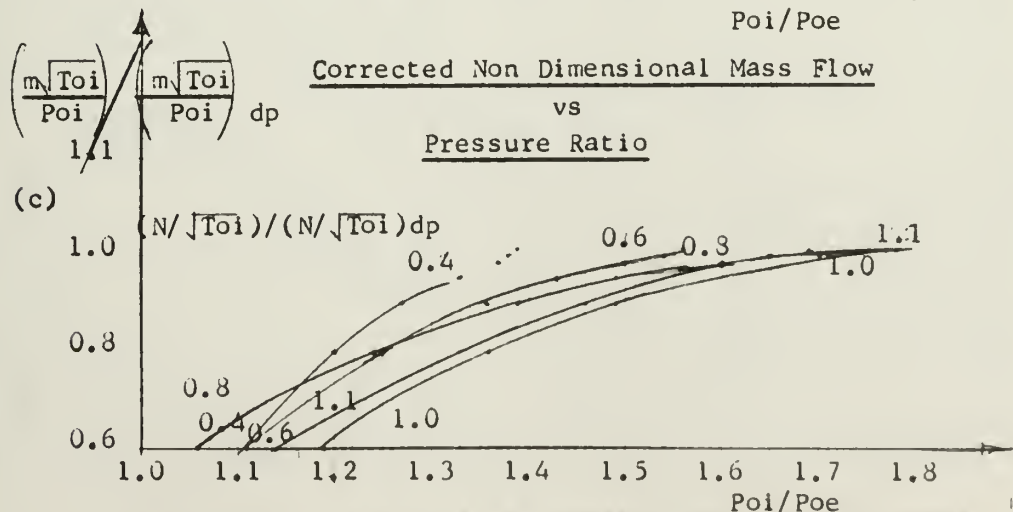
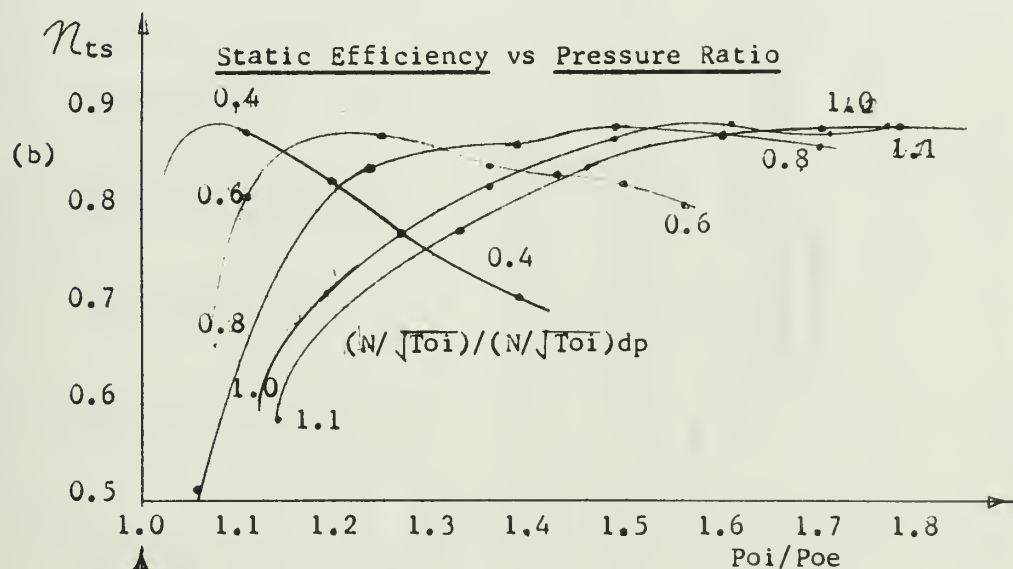
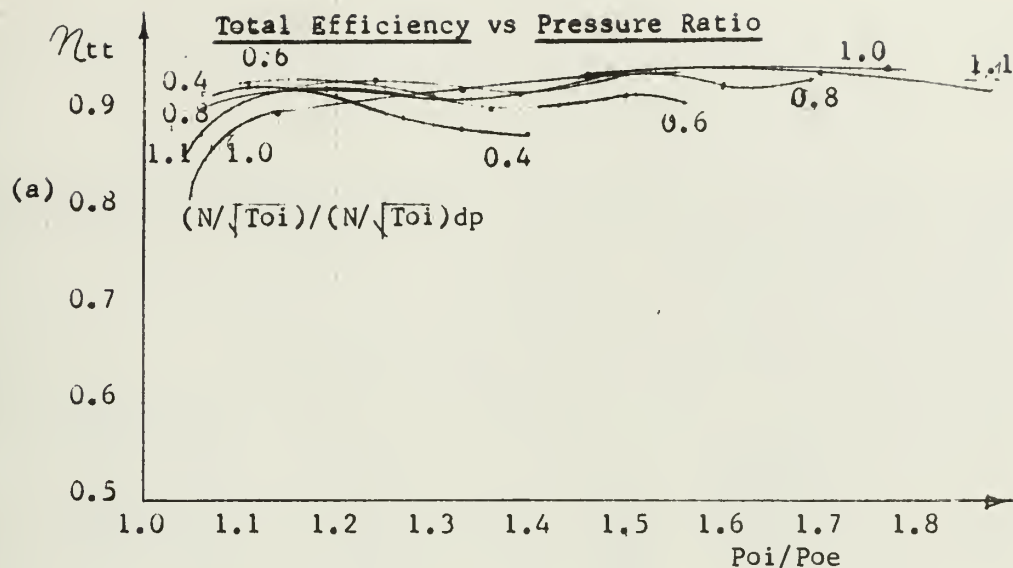


Figure II-3. Performance Curves from data calculated with Computer Program. (a) Isentropic Efficiency (b) Static Efficiency. (c) Corrected mass flow.



## 9030 HP SINGLE STAGE GAS TURBINE ENGINE

MASS FLOW(R/S) POWER(HP) INLET TEMP(R) EXIT TEMP(R) INLET PRESS(P/SI) EXIT PRESS(P/SI) ISSN DFTIC POLY PFTIC  
 119.42 9600. 1768. 1554. 312.480 177.270 0.917 0.911

ORCA(REQ) GAMA CP(RDS/DRP) REACTION HUB PSI WX(F/S) CRIT MACH NO AT EXIT  
 20000. 1.333 0.2715 0.100 1.500 439.2 0.2920

LOCATION	STATION	STAGE	TOTAL TEMP	STATIC TEMP	TOTAL PRESS	STATIC PRESS	DENSITY	ALPHA	REA	REACTION	RADIUS
1	1	1	1767.40	1752.61	312.48	312.56	0.4779	0.00	-32.40	0.100	0.729
2	1	1	1767.53	1753.61	312.44	312.56	0.4754	0.00	-37.96	0.102	0.518
3	1	1	1767.90	1753.61	312.48	312.56	0.4768	0.00	-70.93	0.549	0.607
1	2	1	1767.63	1751.53	302.51	311.49	0.3172	74.82	58.59	0.100	0.229
2	2	1	1767.60	1751.76	302.51	312.20	0.3182	71.20	70.24	0.100	0.238
3	2	1	1767.80	1750.05	302.51	293.51	0.3391	67.73	31.94	0.685	0.449
1	3	1	1553.86	1539.41	177.27	178.88	0.4328	0.00	-51.96	0.100	0.229
2	3	1	1553.84	1539.61	177.27	178.88	0.4328	0.00	-54.38	0.468	0.257
3	3	1	1553.80	1535.61	177.27	178.89	0.4328	0.00	-72.59	0.647	0.686

LOCATION STAGE RELATIVE TOTAL TEMP FOR ROTOR REACTION NO FOR STATOR RELATIVE MACH NO FOR ROTOR

INLET	EXIT	INLET	EXIT
1613.24	0.2204	0.9479	0.4493
1647.37	0.2204	0.7690	0.2436
1665.19	0.2204	0.5961	0.2692

LOC	STAGE	STATOR ANGIN	GAS ANGIN	STATOR ANGOUT	GAS ANGOUT	20138 ANGIN	GAS ANGIN	ROTOR ANGOUT	GAS ANGOUT
1	1	0.0	0.0	74.7	74.8	54.6	59.6	-65.7	-64.0
2	1	0.0	0.0	71.6	71.6	53.2	56.2	-70.1	-56.4
3	1	0.0	0.0	65.6	67.7	-32.9	-32.9	-73.1	-73.0

THE DATA EXCEEDED THE LIMITS OF THE PROGRAM AND S/C'S ASSUMED TO EQUAL 1

LOC STAGE NO BLADES TOTAL BLADE WTS

1	1	46	2.151
---	---	----	-------

ROW STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-BEND FREQ HARMONIC P/C H/C BENDING STRESS

1	1	0.752	1.189	1.504	2.376	4076.6	14.0	0.728	1.997	5264.6
---	---	-------	-------	-------	-------	--------	------	-------	-------	--------

ROW STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-BEND FREQ HARMONIC P/C H/C BENDING STRESS

2	1	1.574	1.245	0.787	2.049	5637.9	10.9	1.772	2.289	2026.5
---	---	-------	-------	-------	-------	--------	------	-------	-------	--------

WOBIN WZTEN WZDZ WZDZ WZDZ WZDZ WZDZ WZDZ WZDZ WZDZ WZDZ

4.65	3.34	5.55	13.54	21.46	0.667	0.429	28723.	1.44
------	------	------	-------	-------	-------	-------	--------	------

DISC DIMENSIONS FROM FLAT TO HUB(ENGLISH)

R(1)	1.00	2(1)	1.229
R(2)	2.00	2(2)	1.228
R(3)	3.00	2(3)	0.959
R(4)	4.00	2(4)	0.466
R(5)	5.00	2(5)	0.463
R(6)	6.00	2(6)	0.468
R(7)	7.00	2(7)	0.463
R(8)	8.00	2(8)	0.397
R(9)	9.00	2(9)	0.370
R(10)	10.00	2(10)	0.363
R(11)	11.00	2(11)	0.326
R(12)	12.00	2(12)	0.326
R(13)	13.00	2(13)	0.326
R(14)	14.00	2(14)	1.423
R(15)	15.00	2(15)	1.023

STAGE STATION DISTANCE FROM STATION 1, STAGE 1

1	1	0.000
2	1	1.7671
3	1	3.5342

LENGTH(IN) RADHUBINLET(IN) RADTOTINLET(IN) RADHUBEXIT(IN) RADTOTEXIT(IN) WTOPSHAFT(LD) WTOP DISCS+BLADES(LB)

3.53	5.15	7.28	5.15	9.23	3.21	24.37
------	------	------	------	------	------	-------

Output: from cycle and  
mechanical design of  
turbine







# Output: From Performance Calculations

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSPEED MATHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (C01) = 0.40 P01/P02 = 1.11

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (R)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	299.3	0.368	0.455	71.2
3	1	1553.6	1722.3	177.3	279.2	0.303	0.442	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	413.	191.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSPEED MATHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (C01) = 0.40 P01/P02 = 1.20

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (R)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	222.2	0.368	0.430	71.2
3	1	1553.6	1694.5	177.3	254.4	0.303	0.411	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	302.	262.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSPEED MATHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (C01) = 0.40 P01/P02 = 1.27

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (R)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	251.1	0.368	0.411	71.2
3	1	1553.6	1676.7	177.3	236.7	0.303	0.389	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	356.	309.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSPEED MATHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (C01) = 0.40 P01/P02 = 1.33

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (R)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	242.6	0.368	0.398	71.2
3	1	1553.6	1640.4	177.3	223.7	0.303	0.372	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	341.	341.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSPEED MATHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (C01) = 0.40 P01/P02 = 1.37

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (R)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	233.7	0.368	0.397	71.2
3	1	1553.6	1623.6	177.3	212.7	0.303	0.357	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439	407	367					







1 439. 302. 265.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SOPT (C01) = 0.60 FOI/PO2 = 1.26

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (P)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1753.6 1753.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	316.5 309.7	220.2 251.4	0.368 0.411	71.2 71.0	20.2 48.3
3	1	1553.8 1553.8	1539.6 1539.6	177.3 228.5	170.9 226.7	0.303 0.374	0.0 -22.1	-69.4 -68.9
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	356.	320.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SOPT (C01) = 0.60 FOI/PO2 = 1.43

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (P)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1753.6 1753.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	306.5 309.2	220.2 242.6	0.368 0.398	71.2 71.0	20.2 48.7
3	1	1553.8 1553.8	1539.6 1539.6	177.3 218.3	170.9 211.9	0.303 0.357	0.0 -31.1	-69.4 -69.0
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	353.	350.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SOPT (C01) = 0.60 FOI/PO2 = 1.50

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (P)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1753.6 1753.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	306.5 304.9	220.2 233.7	0.368 0.387	71.2 71.1	20.2 51.4
3	1	1553.8 1553.8	1539.6 1539.6	177.3 208.6	170.9 195.5	0.303 0.341	0.0 -38.5	-69.4 -69.1
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	447.	385.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SOPT (C01) = 0.60 FOI/PO2 = 1.54

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (P)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1753.6 1753.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	316.5 306.9	220.2 238.9	0.368 0.391	71.2 71.1	20.2 53.3
3	1	1553.8 1553.8	1539.6 1539.6	177.3 203.2	170.9 183.5	0.303 0.332	0.0 -41.7	-69.4 -69.2
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	497.	403.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SOPT (C01) = 0.60 FOI/PO2 = 1.56

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP (P)	TOTAL PRES (PSI)	STATIC PRES (PSI)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1753.6 1753.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	316.5 308.5	223.2 225.5	0.368 0.376	71.2 71.2	20.2 54.0
3	1	1553.8 1553.8	1539.6 1539.6	177.3 200.2	170.9 180.5	0.303 0.336	0.0 -43.2	-69.4 -69.2
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439	436	414					





1 439. 436. 414.

FOR W/SORT(TOI) TOTAL TO 0.60

PO1/PO2	TOTAL EFFICIENCY	STATIC EFFICIENCY	W-SORT(TOI)/PO1
1.11	0.9281	0.3130	0.600
1.25	0.9736	0.809	0.809
1.36	0.9954	0.8435	0.880
1.43	0.9954	0.9376	0.950
1.50	0.9954	0.9885	0.980
1.58	0.9954	0.9933	1.000
1.56	0.9954	0.9958	1.010

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR W/SORT(TOI) = 0.60 PO1/PO2 = 1.06

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LR/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471 0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	299.3	0.368 0.455	71.2	70.9
3	1	1553.8	1745.1	177.3	294.6	0.303 0.456	0.0	68.9
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	213.	180.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR W/SORT(TOI) = 0.60 PO1/PO2 = 1.24

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LR/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471 0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	299.3	0.368 0.455	71.2	70.9
3	1	1553.8	1692.0	177.3	252.2	0.303 0.402	0.0	42.9
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	302.	265.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR W/SORT(TOI) = 0.60 PO1/PO2 = 1.29

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LR/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471 0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	299.3	0.368 0.455	71.2	70.9
3	1	1553.8	1640.1	1529.6	1631.8	0.303 0.369	0.0	16.3
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	365.	323.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR W/SORT(TOI) = 0.60 PO1/PO2 = 1.49

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LR/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471 0.471	0.0	0.0
2	1	1767.8	1753.6	306.5	299.3	0.368 0.455	71.2	70.9
3	1	1553.8	1611.3	1539.6	1601.5	0.303 0.349	0.0	-2.6
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439	383	365					





1 439. 383. 365.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED ARNOLD MATHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
POP N/SORT(T01)= 0.80 POP/P02= 1.60

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(P)	STATIC PRES(P)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1763.6 1763.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	306.5 306.5	220.2 220.2	0.368 0.368	71.2 71.2	20.2 35.0
3	1	1553.8 1553.8	1539.6 1539.6	177.3 177.3	170.9 170.9	0.303 0.303	0.0 -16.2	-69.4 -69.2
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	407.	400.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED ARNOLD MATHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
POP N/SORT(T01)= 1.80 POP/P02= 1.64

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(P)	STATIC PRES(P)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1763.6 1763.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	306.5 306.5	220.2 220.2	0.368 0.368	71.2 71.2	20.2 19.1
3	1	1553.8 1553.8	1539.6 1539.6	177.3 177.3	170.9 170.9	0.303 0.303	0.0 -22.0	-69.4 -69.2
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	427.	416.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED ARNOLD MATHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
POP N/SORT(T01)= 0.80 POP/P02= 1.69

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(P)	STATIC PRES(P)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1763.6 1763.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	306.5 306.5	220.2 220.2	0.368 0.368	71.2 71.2	20.2 42.7
3	1	1553.8 1553.8	1539.6 1539.6	177.3 177.3	170.9 170.9	0.303 0.303	0.0 -26.3	-69.4 -69.3
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439.	436.	430.					

FOR N/SORT(T01) EQUAL TO 0.80

POP/P02	TOTAL EFFICIENCY	STATIC EFFICIENCY	N/SORT(T01)/POP
1.64	0.9762	0.9417	0.600
1.24	0.9305	0.9031	0.800
1.39	0.9178	0.8662	0.900
1.60	0.9342	0.8829	0.950
1.60	0.9264	0.8705	0.980
1.64	0.9327	0.8700	1.000
1.69	0.9304	0.9625	1.010

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED ARNOLD MATHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
POP N/SORT(T01)= 1.30 POP/P02= 0.88

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(P)	TOTAL PRES(P)	STATIC PRES(P)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8 1767.8	1763.6 1763.6	312.5 312.5	302.6 302.6	0.471 0.471	0.0 0.0	-68.0 0.0
2	1	1767.8 1767.8	1631.3 1631.3	306.5 306.5	220.2 220.2	0.368 0.368	71.2 70.9	20.2 -67.4
3	1	1553.8 1553.8	1539.6 1539.6	177.3 177.3	170.9 170.9	0.303 0.303	0.0 76.2	-69.4 -68.9
STAGE VX STATION 1 VX STATION 2 VX STATION 3								
1	439	423	415					



176.

213.

439.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
 CALCULATIONS OF IMPROVED AINSLEY METHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
 FOR N/SOFT (COI) = 1.00 P01/P02 = 1.19

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.9	1767.8	312.5	302.5	0.471	0.471	0.0
2	1	1767.8	1767.8	306.5	220.2	0.368	0.430	71.2
3	1	1533.9	1590.1	177.3	170.9	0.303	0.406	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	402.	285.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
 CALCULATIONS OF IMPROVED AINSLEY METHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
 FOR N/SOFT (COI) = 1.00 P01/P02 = 1.36

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.9	1767.8	312.5	302.5	0.471	0.471	0.0
2	1	1767.8	1767.8	306.5	220.2	0.368	0.430	71.2
3	1	1533.9	1590.1	177.3	170.9	0.303	0.406	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	402.	285.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
 CALCULATIONS OF IMPROVED AINSLEY METHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
 FOR N/SOFT (COI) = 1.00 P01/P02 = 1.49

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.9	1767.8	312.5	302.5	0.471	0.471	0.0
2	1	1767.8	1767.8	306.5	220.2	0.368	0.430	71.2
3	1	1533.9	1590.1	177.3	170.9	0.303	0.406	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	402.	285.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
 CALCULATIONS OF IMPROVED AINSLEY METHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
 FOR N/SOFT (COI) = 1.00 P01/P02 = 1.61

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.9	1767.8	312.5	302.5	0.471	0.471	0.0
2	1	1767.8	1767.8	306.5	220.2	0.368	0.430	71.2
3	1	1533.9	1590.1	177.3	170.9	0.303	0.406	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	402.	285.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
 CALCULATIONS OF IMPROVED AINSLEY METHESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
 FOR N/SOFT (COI) = 1.00 P01/P02 = 1.71

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.9	1767.8	312.5	302.5	0.471	0.471	0.0
2	1	1767.8	1767.8	306.5	220.2	0.368	0.430	71.2
3	1	1533.9	1590.1	177.3	170.9	0.303	0.406	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	402.	285.					

At Design Point

from Cycle Calculation  $Vx_1 = Vx_2 = Vx_3 = 493$



1 439. 477. 428.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SORT(TOI) = 1.00 P01/P02 = 1.77

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	-68.0
2	1	1767.8	1631.3	306.5	220.2	0.368	71.2	20.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.0	-69.4
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	477.	428.					

FOR N/SORT(TOI) EQUAL TO 1.00

P01/P02	TOTAL EFFICIENCY	STATIC EFFICIENCY	N/SORT(TOI)/P01
0.99	1.3723	-0.4225	0.600
1.19	0.9222	0.7128	0.600
1.36	0.9196	0.8223	0.600
1.40	0.9200	0.9709	0.600
1.61	0.9200	0.9861	0.980
1.71	0.9200	0.9792	1.000
1.77	0.9412	0.8962	1.000

At-Design Point

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SORT(TOI) = 1.10 P01/P02 = 0.91

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	-68.0
2	1	1767.8	1631.3	306.5	220.2	0.368	71.2	20.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.0	-69.4
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	477.	428.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SORT(TOI) = 1.10 P01/P02 = 0.91

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	-68.0
2	1	1767.8	1631.3	306.5	220.2	0.368	71.2	20.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.0	-69.4
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439.	477.	428.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE  
CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT  
FOR N/SORT(TOI) = 1.10 P01/P02 = 1.33

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRES(PST)	STATIC PRES(PST)	DENSITY (LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	-68.0
2	1	1767.8	1631.3	306.5	220.2	0.368	71.2	20.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.0	-69.4
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	439	477	428					





1 436. 356. 322.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINNEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT(TOI) = 1.10 P01/P02 = 1.46

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(R)	TOTAL PRPS(PST)	STATIC PRPS(PST)	DENSITY(LB/PT**3)	ALPHA	BETA
1	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
3	1	1553.8	1553.8	177.3	170.9	0.303	0.0	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	436.	356.	322.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINNEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT(TOI) = 1.10 P01/P02 = 1.60

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(R)	TOTAL PRPS(PST)	STATIC PRPS(PST)	DENSITY(LB/PT**3)	ALPHA	BETA
1	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
3	1	1553.8	1553.8	177.3	170.9	0.303	0.0	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	436.	356.	322.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINNEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT(TOI) = 1.10 P01/P02 = 1.70

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(R)	TOTAL PRPS(PST)	STATIC PRPS(PST)	DENSITY(LB/PT**3)	ALPHA	BETA
1	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
3	1	1553.8	1553.8	177.3	170.9	0.303	0.0	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	436.	356.	322.					

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINNEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT(TOI) = 1.10 P01/P02 = 1.79

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(R)	TOTAL PRPS(PST)	STATIC PRPS(PST)	DENSITY(LB/PT**3)	ALPHA	BETA
1	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1767.8	312.5	302.6	0.471	0.0	0.0
3	1	1553.8	1553.8	177.3	170.9	0.303	0.0	0.0
STAGE	VX STATION 1	VX STATION 2	VX STATION 3					
1	436.	356.	322.					

FOR N/SORT(TOI) EQUAL TO 1.10

P01/P02	TOTAL EFFICIENCY	STATIC EFFICIENCY	N-SORT(TOI)/P01
0.93	1.1022	-1.8346	0.600
1.14	0.8002	0.8813	0.800
1.39	0.9222	0.7779	0.800
1.44	0.9381	0.9444	0.950
1.60	0.9466	0.8766	0.950
1.70	0.9422	0.8416	1.000
1.78	0.9419	0.8346	1.010





1 436. 356. 322.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPILED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR W/SORT(M01) = 1.10 P01/P02 = 1.46

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRESS(PST)	STATIC PRESS(PST)	DENSITY(LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1631.3	306.5	220.2	0.368	0.398	71.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.368	0.0
STAGE	VX STATION 1	439.	353.				42.2	-69.4
1			365.					-59.0

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPILED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR W/SORT(M01) = 1.10 P01/P02 = 1.46

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRESS(PST)	STATIC PRESS(PST)	DENSITY(LB/FT**3)	ALPHA	RPTA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1631.3	306.5	220.2	0.368	0.387	71.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.327	0.0
STAGE	VX STATION 1	439.	403.				29.3	-69.4
1								-59.2

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPILED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR W/SORT(M01) = 1.10 P01/P02 = 1.79

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRESS(PST)	STATIC PRESS(PST)	DENSITY(LB/FT**3)	ALPHA	RPTA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1631.3	306.5	220.2	0.368	0.381	71.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.312	0.0
STAGE	VX STATION 1	439.	429.				19.2	-69.4
1								-69.3

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPILED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY HARRISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR W/SORT(M01) = 1.10 P01/P02 = 1.79

STATION	STAGE	TOTAL TEMPERATURE	STATIC TEMP(°F)	TOTAL PRESS(PST)	STATIC PRESS(PST)	DENSITY(LB/FT**3)	ALPHA	BETA
1	1	1767.8	1753.6	312.5	302.6	0.471	0.0	0.0
2	1	1767.8	1631.3	306.5	220.2	0.368	0.376	71.2
3	1	1553.8	1539.6	177.3	170.9	0.303	0.301	0.0
STAGE	VX STATION 1	439.	409.				11.6	-69.4
1								-69.4

FOR W/SORT(M01) EQUAL TO 1.10

P01/P02 TOTAL EFFICIENCY STATIC EFFICIENCY W/SORT(M01)/P01

0.93	1.1022	-1.8246	0.600
1.14	0.8002	0.5813	0.800
1.33	0.9223	0.7774	0.800
1.46	0.9181	0.9000	0.900
1.60	0.9466	0.8766	0.900
1.70	0.9422	0.8816	1.000
1.78	0.9419	0.8846	1.010



## APPENDIX III

Computer Programs developed for the Design  
of a Free Vortex Axial Flow Turbine.

Computer Program 1 Complete design

Computer Program 2 Performance Calculations Only



PGM10001  
PGM10002  
PGM10003  
PGM10004  
PGM10005  
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PGM10032  
PGM10033  
PGM10034  
PGM10035  
PGM10036

# COMPUTER PROGRAM #1

```
// ENDO, REGION=220K, CLASS=C
//MITID USRB=(M13562, P14715, , , VME)
//SRI LOW
//MAIN TIME=6, LINES=2, CARDS=1
// EXEC WAIFV
//C.SYSIN DD *, DCB=BLKSIZE=2000
$JOB
    ENDO, NOLIST, NOSURCHK, TIME=60
    COMMON PI, GO, PJ, HJ, GAMMA, VX, SOMEGA, DHOS, CP, GAMMA2, GAMMA1, KK
    COMMON/AREA1/TET, TC, TIPCIA
    DIMENSION SPRES(3,3,12), RADIDS(3,3,12), ALPHA(3,3,12), BETA(3,3,12)
    DIMENSION DEN(3), PAD(3), TEMP(3,12), TERPS(3,12), STEMP(3,3,12)
    DIMENSION PEACT(3,3,12), RHO(3,3,12), REPLMS(3,2,12), REIMP(3,2,12),
    TEMPR(3,2,12), VAXIAL(3,12)
    DIMENSION RBLADE(3,2,12), SBLADE(3,2,12), CHOPPR(3,2,12), SIGMAB(2,12
    1), SIGMAC(12), BLADEH(2,12), PTOCHO(2,12), ASPECB(2,12), TEM(50), V(50)
    DIMENSION BLADWT(2,12), WTDISC(12), P(15), Z(15), NOBLAD(2,12)
    DIMENSION SPEED(5), WDOTP(7), OMEGAB(5), POIS(7), PHOC(3,10), BPTAC(3,1
    10), ALPHAC(3,10), TOTC(3,10), TSC(3,10), POC(3,10), PSC(3,10), PPOC(3,10
    3), VEL(3,10), FVEL(5,10), PMAC(100), WTAP(100), DSPT(100), VEITOT(100), P
    101PO2(7), ENTMJ(7), ENSMJ(7), PTOC(3,10), A(6), P(6)
    DIMENSION WTAP11(70), WTAP12(70), RMAC11(70), RMAC12(70), VPOT11(70), V
    1EOT12(70), PSPT11(70), PSPT12(70)
    CHARACTER WORD*80
    DATA SPEED/.4,.6,.8,1.,1.1,1.1/
    DATA WDOTP/.6,.8,.9,.95,.98,1.,1.01/
    DATA P/1.6048,1.339,1.1857,1.1,1.04,1./
    DATA A/0.,.2,.4,.6,.8,1./
    IL=5
    KK=6
    E=.001
    HJ=778.16
    PI=3.14159264
    GO=32.174
    LKL=0
    LKL=LKL+1
    READ(IL,217)WORD
```



217	FORMAT(A80)	PGM10037
218	WRITE(KK,218)WORD	PGM10038
	FORMAT('1',APC)	PGM10039
219	READ(LL,219)M1,M2,M,KLM,KTM,GAMMA,CP,POWER,W,TC,TET,TIPCLA,TOI,POI	PGM10040
220	FORMAT(211,I3,215,2F5.3,F10.1,F10.3,3F5.3,2F10.1)	PGM10041
221	READ(LL,211)PSI,PTE,PATROT,REPCTH,OMEGA,KLM1,M4,SPECWT,YMOD,C1,C2	PGM10042
	FORMAT(4F5.3,F10.0,I2,I8,F5.3,F15.0,2F10.0)	PGM10043
	READ(LL,212)ENI,RSHAFT,PRATIO,ULTSRE,ALPHAH,EMACH,M3,NKI,N,APERRA,	PGM10044
	1T00	PGM10045
212	FORMAT(F5.3,F10.5,F5.3,F10.0,F5.3,F10.4,I1,I2,I2,F5.4,F5.2)	PGM10046
	POI=POI*144.	PGM10047
	IF(M1.E0.1)GO TO 214	PGM10048
	READ(LL,200)(RMAC(I),I=1,KLM)	PGM10049
	READ(LL,201)(WTAP(I),I=1,KLM)	PGM10050
	READ(LL,201)(PSPT(I),I=1,KIM)	PGM10051
	READ(LL,201)(VELTOT(I),I=1,KLM)	PGM10052
200	FORMAT(20F4.2)	PGM10053
201	FORMAT(10F8.5)	PGM10054
	READ(LL,227)(TFM(I),I=1,KTM)	PGM10055
	READ(LL,227)(V(I),I=1,KTM)	PGM10056
227	FORMAT(20F5.C)	PGM10057
	KLM2=KLM-KLM1	PGM10058
	DO 230 I=1,KLM1	PGM10059
	II=I+KLM1-1	PGM10060
	RMAC11(I)=RMAC(I)	PGM10061
	WTAP11(I)=WTAP(I)	PGM10062
	PSPT11(I)=PSPT(I)	PGM10063
	VEOT11(I)=VEITOT(I)	PGM10064
	IF(I.GT.KLM2)GO TO 230	PGM10065
	RMAC12(I)=RMAC(II)	PGM10066
	WTAP12(I)=WTAP(II)	PGM10067
	PSPT12(I)=PSPT(II)	PGM10068
	VEOT12(I)=VEITOT(II)	PGM10069
230	CONTINUE	PGM10070
214	SOMEGA=OMEGA*PI/30.	PGM10071
	GAMMA1=GAMMA/(GAMMA-1.)	PGM10072





```

5      LK=0
      TTEMP(3,N)=TCE
      TPRES(3,N)=PCE
      PHI=VX/(SOMEGA*RHE)
      PSI=GO*HJ*DHCS/(SOMEGA*RHE)*2
      IF((PSI.GT.2.0).AND.(REACTH.LT.0.05)) REACTH=0.05
      ALPHAH=ATAN(1.-REACTH-PSI/2.)/PHI
      IF(PSI.LT.2.0) ALPHAH=0.0
      ALPHA(1,3,N)=ALPHAH
      ALPHA(2,3,N)=ANGLE(RHE,ALINAH,PME)
      ALPHA(3,3,N)=ANGLE(RHE,ALPHAH,RTT)
      IF(EMACH.GT.0.0) GO TO 600
3      DO 4 I=1,3
4      CALL TP1(TTEMP(3,N),STEMP(I,3,N),ALPHA(I,3,N),TPRES(3,N),SPRES(I,3
      1,N),DEN(I))
      RAD(1)=RHE
      RAD(2)=RHF
      RAD(3)=RTE
      CALL DENE(RAD,DEN,N,VXP)
      IF(ABS(VXP/VX-1.).LE.7) GO TO 6
      J=J+1
      IF(J.GT.20) GO TO 6
      IF(VXP.GT.2.*VX) GO TO 656
      IF(J.EQ.2) GO TO 655
      IF(ABS(VXP/VX-1.).LE.0.1) GO TO 655
      IF(VXP/VX.GT.1.) GO TO 654
      IF(LK.EQ.1) GO TO 657
      VX=VX-DELTA V
      JK=1
      LK=0
      GO TO 5
657  DELTA V=DELTA V/2.
      VX=VX-DELTA V
      JK=1
      LK=0
      GO TO 5

```

PGM10100  
 PGM10110  
 PGM10111  
 PGM10112  
 PGM10113  
 PGM10114  
 PGM10115  
 PGM10116  
 PGM10117  
 PGM10118  
 PGM10119  
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 PGM10143  
 PGM10144



```

654 IF(JK.EQ.1)GO TO 658
    VY=VX+DELTAV
    LK=1
    JK=0
    GO TO 5
658 DELTAV=DELTAV/2.
    VX=VX+DELTAV
    LK=1
    JK=C
    GO TO 5
655 VX=VXP
    GO TO 5
656 VX=1.5*VX
    J=1
    GO TO 5
659 H=0.9*(PTE-RHE)
    RMF=PTE-H/2.
    RHE=PTE-H
    J=1
    LK=0
    JK=0
    GO TO 5
6 IF(EMACH.LT.C.) VX=VXP
    IF(ABS(ALPHA*180./PI).GT.25.)GO TO 659
    KKK=N
    DO 8 I=1,3
        RADIUS(I,3,N)=RAD(I)
        RHO(I,3,N)=DFN(I)
        REACT(I,3,N)=REACTI(RADIUS(I,3,N),ALPHA(T,3,N))
        BETA(I,3,N)=ANGLEB(ALPHA(I,3,N),RADIUS(T,3,N))
        GO TO 100
    J=0
    VX=EMACH*SORT((2./(GAMMA+1.))*GO*PJ*TOE)
    H=.98*RHE
    DELTA=H/10.
    LK=0
    GO TO 100
    PGM10145
    PGM10146
    PGM10147
    PGM10148
    PGM10149
    PGM10150
    PGM10151
    PGM10152
    PGM10153
    PGM10154
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    PGM10180

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PGM10073  
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PGM10106  
PGM10107  
PGM10108

```

RJ=CP*HJ/GAMMA1
J=1
RHE=RTE/EATIOR
RME=(RTE+PHE)/2.
IF(M2.EQ.1)GO TO 215
POWERP=POWER*550./HJ
DHO=POWERB/(W*ENT)
TOE=TOI-DHO/CP
TRATIO=TCE/TOI
ENP=ALOG(TRATIO)/ALOG((TRATIO-1.)/ENT+1.)
GAMMA2=GAMMA1/FNP
PCE=POI*TRATIO*GAMMA2
GO TO 216

215  PRATIO=1./PRATIO
      ENP=(ALOG(1.-ENT*(1.-PRATIO*(1./GAMMA1)))/LOG(PRATIO))*GAMMA1
      GAMMA2=GAMMA1/FNP
      TOE=TOI*PRATIO*(1./GAMMA2)
      DHO=CP*(TOI-TOE)
      POWER=DHO*ENT*W*HJ/550.
      POE=POI*PPATIO
      PHE=SOMEGA*RHE
      P01=POI/144.
      P02=POE/144.
      DHOS=PSI*(PHE*SOMEGA)**2/(GC*HJ)
      VX=.75*SOMEGA*PHE
      DELTAV=VX/1C.
      IF(N.GT.0)GO TO 2
      N=INT(DHO/DHOS)
      N=N+1
      RN=FLOAT(N)
      DHOS=DHO/RN
      IF((ALPHAH.EQ.0.0).AND.(PST.LT.2.0))FRACHTH=1.-PSI/2.
      IF((RPACHTH.LT.0.95).AND.(ALPHAH.EQ.0.0))PSI=2.*(1.-C.C5)
      IF((PSI.LT.2.0).AND.(ALPHAH.EQ.0.0))PHE=SQRT(DHOS*GO*HJ/(PSI*SOMEGA
1A**2))
      JK=0

```



```

603 JK=0
      J=J+1
      RAD(1)=RHE
      RAD(2)=RHE+H/2.
      RAD(3)=RHE+H
      IF(PAD(3)/PHF.GT.2.0)GO TO 602
      PSI=GO*HJ*DHOS/(SOMEGA*RAD(1))*2
      PHI=VX/(SOMEGA*RAD(1))
      IF((PSI.GT.2.0).AND.(REACTH.LT.0.05)) REACTH=.05
      ALPHA=ATAN((1.-REACTH-PSI/2.)/PHI)
      IF(PSI.LT.2.)ALPHAH=C.0
      DO 611 I=1,3
        ALPHA(I,3,N)=ANGLE(PAD(1),ALPHAH,RAD(I))
        CALL TP1(TTEMP(3,N),STEMP(I,3,N),ALPHA(I,3,N),TDEES(3,N),SPRES(I,3
1,N),DEN(I))
        CALL DENR(RAD,DEN,H,VXP)
        RME=RAD(2)
        RHE=RAD(1)
        RTE=RAD(3)
        IF(ABS(VXP/VX-1.).LE.F)GO TO 6
        IF(J.GT.20)GO TO 6
        IF(VXP/VX.LT.1.)GO TO 604
        IF(LK.EQ.1)GO TO 607
        H=H+DELTA
        JK=1
        LK=0
        IF(H.GT.0.5*PTE)VX=VXP
        GO TO 603
        DELTA=DEITA/2.
        H=H+DELTA
        JK=1
        LK=0
        IF(H.GT.0.5*PTE)VX=VXP
        GO TO 603
        IF(JK.EQ.1)GO TO 608
        H=H-DELTA
604
607

```





```

LK=1
JK=0
GO TO 603
DELTA=DELTA/2.
H=H-DELTA
LK=1
JK=0
GO TO 603
RHE=RAD(3)/1.00
GO TO 601
J=3
K=J-2
TEMP(K,N)=ITEMP(J,N)+DHOS/CP
TPRES(K,N)=TPRES(J,N)/(TEMP(J,N)/TEMP(K,N))*GAMMA2
RADIUS(M,K,N)=RADIUS(M,J,N)
IF(N.EQ.1)GO TO 9
ALPHA(M,K,N)=ALPHA(M,J,N)
GO TO 10
9 ALPHA(M,K,N)=0.
10 BETA(M,K,N)=ANGLEB(ALPHA(M,K,N),RADIUS(M,K,N))
CALL TP1(TEMP(K,N),STEMP(M,K,N),ALPHA(M,K,N),TPRES(K,N),SPRES(M,K,N),DEN(M))
H=W/(2.*PI*VX*DEN(M)*RADIUS(M,K,N))
LK=0
JK=0
JJ=0
DELTA=H/10.
JJ=JJ+1
IF(M.EQ.1)GO TO 75
IF(M.EQ.3)GO TO 76
RAD(2)=PME
RAD(1)=PME-H/2.
RAD(3)=PME+H/2.
GO TO 77
75 RAD(1)=PHE
RAD(2)=PHE+H/2.

```



```

76 RAD(3)=RHE+H
   GO TO 77
   RAD(3)=RTE
   RAD(2)=RTE-H/2.
   RAD(1)=RTE-H
77 DO 11 I=1,3
   ALPHA(I,K,N)=ANGLE(RAD(M),ALPHA(M,K,N),RAD(I))
11 CALL IPI(TTEMP(K,N),STEMP(I,K,N),ALPHA(I,K,N),TEMPES(K,N),SPRES(I,K,
   1,N),DEN(I))
   CALL DENE(RAD,DEN,W,VXP)
   IF(ABS(VXP/VX-1.).LE.E)GO TO 15
   IF(JJ.EQ.20)GO TO 15
   IF(VXP/VX.LT.1.)GO TO 14
   IF(LK.EQ.1)GO TO 17
   H=H+DELTA
   JK=1
   LK=0
   GO TO 13
17 DELTA=DELTA/2.
   H=H+DELTA
   JK=1
   LK=0
   GO TO 13
14 IF(JK.EQ.1)GO TO 18
   H=H-DELTA
   LK=1
   JK=0
   GO TO 13
18 DELTA=DELTA/2.
   H=H-DELTA
   LK=1
   JK=0
   GO TO 13
15 DO 16 I=1,3
   RHO(I,K,N)=DEN(I)
   RADIUS(I,K,N)=RAD(I)

```



```

16  REACT(I,1,N)=REACTI(RADIUS(I,K,N),ALPHA(I,K,N))
    BETA(I,K,N)=ANGLEB(ALPHA(I,K,N),RADIUS(I,K,N))
C SOLUTION FOR STATION 2
    ALPHA(M,2,N)=ANGLE2(RADIUS(M,3,N),ALPHA(M,3,N))
    TTEMP(2,N)=TTEMP(1,N)
    CALL TPD2(TTEMP(2,N),STEMP(M,1,N),STEMP(M,2,N),SPRES(M,
11,N),SPRES(M,2,N),ALPHA(M,2,N),DEN(M))
    H=W/(2.*PI*VX*DEN(M)*RADIUS(M,K,N))
    DELTA=H/10.
    JK=J
    LK=0
    JJ=0
    JJ=JJ+1
33  IF(M.EQ.1)GO TO 78
    IF(M.EQ.3)GO TO 79
    RAD(2)=PME
    RAD(1)=RAD(2)-H/2.
    RAD(3)=RAD(2)+H/2.
    GO TO 80
78  RAD(1)=RHE
    RAD(2)=RHE+H/2.
    RAD(3)=RHE+H
    GO TO 80
79  RAD(3)=RTE
    RAD(2)=RTE-H/2.
    RAD(1)=RTE-H
    DO 25 I=1,3
80  ALPHA(I,2,N)=ANGLE(PAD(M),ALPHA(M,2,N),RAD(I))
25  CALL TPD2(TTEMP(2,N),STEMP(I,1,N),STEMP(I,2,N),SPRES(I,
11,N),SPRES(I,2,N),ALPHA(I,2,N),DEN(I))
    CALL DENE(PAD,DEN,H,VXP)
    IF(ABS(VXP/VX-1.)<.1E-5)GO TO 30
    IF(VXP/VX<.1E-1)GO TO 26
    IF(JJ.EQ.20)GO TO 30
    IF(LK.EQ.1)GO TO 27
    H=H+DELTA

```

PGM10289  
 PGM10290  
 PGM10291  
 PGM10292  
 PGM10293  
 PGM10294  
 PGM10295  
 PGM10296  
 PGM10297  
 PGM10298  
 PGM10299  
 PGM10300  
 PGM10301  
 PGM10302  
 PGM10303  
 PGM10304  
 PGM10305  
 PGM10306  
 PGM10307  
 PGM10308  
 PGM10309  
 PGM10310  
 PGM10311  
 PGM10312  
 PGM10313  
 PGM10314  
 PGM10315  
 PGM10316  
 PGM10317  
 PGM10318  
 PGM10319  
 PGM10320  
 PGM10321  
 PGM10322  
 PGM10323  
 PGM10324



```

27  JK=1
    IK=0
    GO TO 33
    DELTA=DELTA/2.
    H=H+DELTA
    JK=1
    LK=0
    GO TO 33
34  IF (JK.EQ.1) GO TO 38
    H=H-DELTA
    LK=1
    JK=0
    GO TO 33
38  DELTA=DELTA/2.
    H=H-DELTA
    LK=1
    JK=0
    GO TO 33
    DO 36 I=1,3
    RHO(I,2,N)=DEN(I)
    RADIUS(I,2,N)=RAD(I)
    REACT(I,2,N)=REACT2(RADIUS(I,2,N),ALPHA(I,2,N))
    BETA(I,2,N)=ANGLPB(BETA(I,2,N),RADIUS(I,2,N))
    IF (N.EQ.1) GO TO 40
    NN=N-1
    DO 41 I=1,3
    ALPHA(I,3,NN)=ALPHA(I,1,N)
    BETA(I,3,NN)=BETA(I,1,N)
    RADIUS(I,3,NN)=RADIUS(I,1,N)
    STEMP(I,3,NN)=STEMP(I,1,N)
    SPRES(I,3,NN)=SPRES(I,1,N)
    RHO(I,3,NN)=RHO(I,1,N)
    TT2MP(3,NN)=TT2MP(1,N)
    TPRES(3,NN)=TPRES(1,N)
    N=N-1
    GO TO 100

```

```

PGM10325
PGM10326
PGM10327
PGM10328
PGM10329
PGM10330
PGM10331
PGM10332
PGM10333
PGM10334
PGM10335
PGM10336
PGM10337
PGM10338
PGM10339
PGM10340
PGM10341
PGM10342
PGM10343
PGM10344
PGM10345
PGM10346
PGM10347
PGM10348
PGM10349
PGM10350
PGM10351
PGM10352
PGM10353
PGM10354
PGM10355
PGM10356
PGM10357
PGM10358
PGM10359
PGM10360

```





```

40 WRITE(KK,43)W,POWER,TOI,TOE,PO1,PO2,EN*,END
43 FORMAT('0','MASS FLOW(IB/S) POWER(HP) INLET TEMP(R) EXIT TEMP(R)
1) INLET PRESS(PSSI) EXIT PRESS(PSSI) ISSN EFFIC POLY EFFIC',/
1,F12.2,F12.0,2F14.0,F14.3,F17.3,F16.3,F12.3)
EMACH=VX/SORT((2./(GAMMA+1.))*RJ*GO*TOE)
WRITE(KK,70)OMEGA,GAMMA,CP,REACTH,PSI,VX,EMICH
70 FORMAT('0','OMEGA(REF) GAMMA CP(RIU/LBMR) REACTION HUB PSI VX
1(FT/S) CRIT MACH NO AT EXIT',/
11,F15.4)
DO 45 K=1,KKK
DO 45 J=1,3
DO 45 I=1,3
IF(J.EQ.3)GO TO 37
REALMS(I,J,K)=SQRT((TIME(J,K)/STEMP(I,J,K)-1.)*2./(GAMMA-1.))
IJ=J+1
STEMPR(I,J,K)=STEMP(I,IJ,K)+(VX/COS(BETA(I,IJ,K)))*2/(2.*CP*HJ*GO
1)
RELMR(I,J,K)=SQRT((ITEMPR(I,J,K)/STEMP(I,IJ,K)-1.)*2./(GAMMA-1.))
CONTINUE
37 ALPHA(I,J,K)=180.*ALPHA(I,J,K)/PI
BETA(I,J,K)=BETA(I,J,K)*180./PI
CONTINUE
45 WRITE(KK,42)
42 FORMAT('0','LOCATION STATION DENSITY ALPHA BETA TOTAL TEMP STATIC TEMP TOT
1AL PRESS STATIC PRESS DENSITY ALPHA BETA REACTION RADIUS')
DO 250 K=1,KKK
DO 250 J=1,3
DO 250 I=1,3
IF((J.EQ.3).AND.(K.NE.KKK))GO TO 250
TOTPRE=TPRES(J,K)/144.
STPRE=SPRES(I,J,K)/144.
WRITE(KK,50)I,J,K,TOTEMP(J,K),STEMP(I,J,K),TOTPRE,SPRES,PHO(I,J,K)
50 1,ALPHA(I,J,K),BETA(I,J,K),REACT(I,J,K),RADIUS(I,J,K)
FORMAT(' ',I5,I6,I8,F14.2,F12.2,F12.2,F12.2,F12.2,F12.2,F12.2,F12.2,F
18.3,F8.3)
250 CONTINUE

```

PGM10361  
 PGM10362  
 PGM10363  
 PGM10364  
 PGM10365  
 PGM10366  
 PGM10367  
 PGM10368  
 PGM10369  
 PGM10370  
 PGM10371  
 PGM10372  
 PGM10373  
 PGM10374  
 PGM10375  
 PGM10376  
 PGM10377  
 PGM10378  
 PGM10379  
 PGM10380  
 PGM10381  
 PGM10382  
 PGM10383  
 PGM10384  
 PGM10385  
 PGM10386  
 PGM10387  
 PGM10388  
 PGM10389  
 PGM10390  
 PGM10391  
 PGM10392  
 PGM10393  
 PGM10394  
 PGM10395  
 PGM10396



```

128      WRITE(KK,128)
      FORMAT('0','LOCATION STAGE RELATIVE POI:1 TEMP FOR ROTOR REAL M
129      TACH NO FOR STATOR RELATIVE MACH NO FOR ROTOR')
      WRITE(KK,129)
      FORMAT(' ',25X,'INLET',10X,'EXIT',7X,'INLET',10X,'EXIT',7X,'EX
1,10X,'EXIT')
      WRITE(KK,130)((I,K,TEMPR(I,1,K),TEMPR(I,2,K),REALMS(I,1,K),REALM
1S(I,2,K),RELMP(I,1,K),RELMP(I,2,K),I=1,3),K=1,KKK)
130      FORMAT(' ',I8,I7,F13.2,F16.2,4X,F10.4,F12.4,F16.4)
C CALCULATE BLADE ANGLES OF STATOR AND ROTOR
      DO 60 K=1,KKK
      DO 60 J=1,2
      DO 60 I=1,3
      IF(J.EQ.2)GO TO 49
      RBLADE(I,J,K)=BETA(I,2,K)
      SBLADE(I,J,K)=ALPHA(I,J,K)
      GO TO 60
49      CALL BLADE(ALPHA(I,2,K),REALMS(I,2,K),SBLADE(I,J,K),1)
      BETAOT=-BETA(I,3,K)
      CALL BLADE(BETAOT,RELMP(I,2,K),BLADE3,1)
      RBLADE(I,2,K)=-BLADE3
      GO TO 60
60      CONTINUE
      WRITE(KK,145)
145      FORMAT('0','LOC STAGE STATOR ANGIN GAS ANGIN STATOR ANGOUT GA
1S ANGOUT ROTOR ANGIN GAS ANGIN ROTOR ANGOUT GAS ANGOUT')
      WRITE(KK,150)((J,K,SBLADE(J,1,K),ALPHA(J,1,K),SBLADE(J,2,K),ALPHA(
1J,2,K),RBLADE(J,1,K),BETA(J,2,K),RBLADE(J,3,K),BETA(J,3,K),J=1,3),
2K=1,KKK)
150      FORMAT(' ',I2,I6,8F12.1)
      SIGMAT=.75*ULTSRF/1.2**2
      OMEGAT=SOMEGA
      STPESC=0.0
      TF=.49+.51*APBARA
      CALL FIG(6,A,B,ARPA,CF)
      DO 103 K=1,KKK
      DO 103 J=1,2

```



```

L=J+1
HEIGHT=(RADIUS(2,I,K)-RADIUS(1,I,K)+RADIUS(2,J,K)-RADIUS(1,J,K))/2.
1.
IF(J.EQ.1)GO TO 101
SMTANG=RRPLADE(1,1,K)-RRPLADE(1,2,K)
BETA3=-BETA(2,3,K)
CALL STOCRA(BETA(2,2,K),BETA3,PCPAT)
AREA=PI*(RADIUS(2,2,K)+RADIUS(2,3,K))*HEIGHT
STRESC=4.51*TF*SPICWT*AREA*144.*(OMEGA/1000.)**2
A2=PI*ALPHA(2,2,K)/180.
A3=PI*ALPHA(2,3,K)/180.
IF(J.EQ.2)GO TO 102
SMTANG=SRPLADE(3,2,K)-SRPLADE(3,1,K)
A1=-ALPHA(2,1,K)
A2=ALPHA(2,2,K)
CALL STOCRA(A1,A2,PCPAT)
A2=PI*A2/180.
A3=PI*A1/180.
CALL SECMOD(SMTANG,TC,SM,AREAC2)
STRESB=W*VX*(TAN(22)-TAN(A3))*PCPAT*HEIGHT/(SM*2.*PI*(RADIUS(2,J,K)
1)+RADIUS(2,L,K))*GO)
PPEO=11.0*CF*SORT(YMOD*SM*TC/(SPECWT*AREAC2))/(HEIGHT**2*144.)
C4=1.3*41.*(OMEGA/60.)*STRESB/PRFO
C5=C4/(C1*(1.-STRFSC/C2))
IF(J.EQ.1)C5=C4/C1
CHORD=C5*(1./3.)
FREQ1=FFFQ*CHORD
HARMON=FFEO1/(OMEGA/60.)
IF(CHORD.LT..6)CHORD=.6
CK1=AREAC2*CHORD**2
CK2=CK1*(1.-7FEAPA)
CHORDR(2,J,K)=SORT((CK1-CK2/2.)/AREAC2)
IF(J.EQ.1)GO TO 61
CHORDP(1,J,K)=CHORD
CHORDR(3,J,K)=SORT((CK1-CK2)/(AREAC2))
GO TO 62

```

101

102

PGM10433  
PGM10434  
PGM10435  
PGM10436  
PGM10437  
PGM10438  
PGM10439  
PGM10440  
PGM10441  
PGM10442  
PGM10443  
PGM10444  
PGM10445  
PGM10446  
PGM10447  
PGM10448  
PGM10449  
PGM10450  
PGM10451  
PGM10452  
PGM10453  
PGM10454  
PGM10455  
PGM10456  
PGM10457  
PGM10458  
PGM10459  
PGM10460  
PGM10461  
PGM10462  
PGM10463  
PGM10464  
PGM10465  
PGM10466  
PGM10467  
PGM10468



```

61 CHORDR(3,J,K)=CHORD
62 CHORDP(1,J,K)=SQRT((CK1-CK2)/AREA2)
CONTINUE
SIGMAB(J,K)=STEESB/CHORD**2
BLADEH(J,K)=HEIGHT*12.
ASPECR(J,K)=BLADEH(J,K)/CHORDP(2,J,K)
PTOCHO(J,K)=PCFAT
IF(J.EQ.2)SIGMAC(K)=STEESC
PITCH=PCFAT**CHORDP(2,J,K)
BLADNR=12.*PI*(RADIUS(2,J,K)+RADIUS(2,L,K))/PITCH
KL=INT(BLADNR)
IF(J.EQ.2)GO TO 63
KL=(KL/2)*2
GO TO 64
63 CALL TESTP(KL,KL)
IF(KL.EQ.1)GO TO 64
KL=KL-1
GO TO 63
64 NOBLAD(J,K)=KL
PITCH=12.*PI*(RADIUS(2,J,K)+RADIUS(2,L,K))/FLOAT(KL)
PTOCHO(J,K)=PITCH/CHORDP(2,J,K)
RHUB=(RADIUS(1,L,K)+RADIUS(1,J,K))/2.
RTIP=RHUB+HEIGHT
ALPHA=(1.-AFBARA)/PIIP
BIADWT(J,K)=12.*CK1*SPECWT*((ALPHA**2-B**IP**2)+PTIP-3HU
1B)*FLOAT(NOBLAD(J,K))
IF(J.EQ.1)GO TO 88
WO=0.65*CHORDP(1,J,K)
IP(WO,LT.0.7)WO=0.7
IP(RSHAFT,LT.0.01)PSHAFT=PAD(M)/6.
PI=2.*PSHAFT
TI=PSHAFT
TC=WO/24.
RO=RHUB-TO
ROPT=EO/RTIP
PHPT=RHUB/RTIP
PGM10469
PGM10470
PGM10471
PGM10472
PGM10473
PGM10474
PGM10475
PGM10476
PGM10477
PGM10478
PGM10479
PGM10480
PGM10481
PGM10482
PGM10483
PGM10484
PGM10485
PGM10486
PGM10487
PGM10488
PGM10489
PGM10490
PGM10491
PGM10492
PGM10493
PGM10494
PGM10495
PGM10496
PGM10497
PGM10498
PGM10499
PGM10500
PGM10501
PGM10502
PGM10503
PGM10504

```





```

PIPT=PI/RTIP
SIGROW=12.*SPECWT*OMEGAT**2*RTIP**2/(SIGMAT*GO)
ALPPT=ALPHAW*RTIP
ABWOT=FLOAT(NBELAD(J,K))*CK1/(WO*GO*12.)
CELAAD1=(1.-PHRT**2-(2.*ALPPT/3.)*(1.-PHRT**3))
CBLADE=(ABWOT/(4.*PI))*CBLAD1
DFLTAR=(GO-PI)/10.
ZO=(WO*TO/RO)*( (POPT**2+CBLADF)*SIGROW-1.)
IP(ZO,LT,WO/5.)ZO=WO/5.
IF(ZO,LT,0.3)ZO=0.3
ZI=ZO*EXP((SIGROW/2.)*(ROPT**2-PIPT**2))
WI=(ZI*RI/TI)/(1.-RIPT**2*SIGROW)
IF(WI,LT,1.2*WO)WI=1.2*WO
WTMID=PI*SPECWT*298.*ZO*RTIP**2*EXP((SIGROW/2.)*FOPT**2)*(EXP(-(SIGROW/2.)*RIPT**2)-EXP(-(SIGROW/2.)*FOPT**2))/SIGROW
WTIPM=SPECWT*WI*PI*144.*(PI**2-3*SHARP**2)
WTOPI=SPECWT*WO*PI*144.*(RHUR**2-PO**2)
WTDISC(K)=WTIPM+WTCPIM+WTMID
WTOI=WTDISC(K)+ELADF*(J,K)
R(1)=RSHAFT
Z(1)=WI
R(2)=RI
Z(2)=WI
E(3)=RI
Z(3)=ZI
P(14)=RO
Z(14)=WO
R(15)=PHUR
Z(15)=WO
SUM=0.
SUM2=0.
DO 84 II=4,13
IJ=II-1
P(II)=P(IJ)+DFLTAR
Z(II)=ZO*EXP((SIGROW/2.)*(PORT**2-(P(II)/RTIP)**2))
SUMZ=(Z(II)+Z(IJ))/2.

```

PGM10505  
 PGM10506  
 PGM10507  
 PGM10508  
 PGM10509  
 PGM10510  
 PGM10511  
 PGM10512  
 PGM10513  
 PGM10514  
 PGM10515  
 PGM10516  
 PGM10517  
 PGM10518  
 PGM10519  
 PGM10520  
 PGM10521  
 PGM10522  
 PGM10523  
 PGM10524  
 PGM10525  
 PGM10526  
 PGM10527  
 PGM10528  
 PGM10529  
 PGM10530  
 PGM10531  
 PGM10532  
 PGM10533  
 PGM10534  
 PGM10535  
 PGM10536  
 PGM10537  
 PGM10538  
 PGM10539  
 PGM10540



```

SUM=SUM+SUMZ*DELTA
SUM2=SUM2+SUMZ*(P(I1)+P(IJ))/2.)*2*DELTAP
CONTINUE

```

84

```

AREADT=24.*(WC*TO+WT*TI+SUM)
FORORI=WC*(RHUB**3-FC**3)
FORIPI=WI*(RI**3-RSHAFT**3)
FORMID=SUM2
FORCT=(SPECWT/GO)*(288.*(FORIPI/3.+FOROP/3.+FORMID)+FLOCAT(NOBLEAD(
1J,K))*RTIP**2*CK1*6.*CBLAD1/PI)
BURST=FORCT/AREADT
SPEEDB=SQRT(ULTSPE/BURST)*30./PI
BSOS=SPEEDR/CM*GA
WRITE(KK,89)J,K,NOBLAD(J,K),BLADWT(J,K)
FORMAT('0','POW STAGE NO BLADES TOTAL BLADE WTS.'/,I3,I7,-11,
1F10.2)

```

88  
89

```

IF(J.EQ.1)GO TO 95
WRITE(KK,121)J,K,CHORDP(1,J,K),CHOEDR(2,J,K),CHORDP(3,J,K),BLADEH(
1J,K),PREO1,HARMON,PCHO(J,K),ASPEC(J,K),SIGMAP(J,K),SIGMAC(K)
FORMAT('0','POW STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-REND RPE
10 HARMONIC P/C H/C BENDING STRESS CENTRIFUGAL STRESS.'/,I3,
3I7,4F8.3,F10.1,F10.1,F10.3,F6.3,F10.1,F17.1)
WRITE(KK,92)WTORIM,WTIRIM,WTMD,WTDISC(K),WTOI,RTIP,RHUB,SPEEDB,BS
10S

```

121

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FORMAT('0','WTORIM WTIRIM WTMD WTDISC WTC RTIP RHUB BURST
1 SPEED BS/OMEGA'//,F6.2,F8.2,F-.2,F-.2,F7.2,2F6.3,F10.2,F9.2)
DO 96 I=1,15
R(I)=R(I)*12.
WRITE(KK,97)
FORMAT('0','DISC DIMENSIONS FROM RSHAFT TO FHUB(INCHES)')
WRITE(KK,93)I,R(I),I,Z(I),I=1,15)
FORMAT(' ','F(I,I2)=' ,F5.2,' Z(' ,I2,' ) = ' ,F5.3)
GO TO 98

```

92  
96

```

WRITE(KK,120)J,K,CHORDP(1,J,K),CHOEDR(2,J,K),CHOEDR(3,J,K),BLADEH(
1J,K),PREO1,HARMON,PCHO(J,K),ASPEC(J,K),SIGMAP(J,K)
FORMAT('0','POW STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-BEND RPE
10 HARMONIC P/C H/C BENDING STRESS.'/,I3,I7,4F8.3,2F10.1,F10.

```

120

PGM10541  
PGM10542  
PGM10543  
PGM10544  
PGM10545  
PGM10546  
PGM10547  
PGM10548  
PGM10549  
PGM10550  
PGM10551  
PGM10552  
PGM10553  
PGM10554  
PGM10555  
PGM10556  
PGM10557  
PGM10558  
PGM10559  
PGM10560  
PGM10561  
PGM10562  
PGM10563  
PGM10564  
PGM10565  
PGM10566  
PGM10567  
PGM10568  
PGM10569  
PGM10570  
PGM10571  
PGM10572  
PGM10573  
PGM10574  
PGM10575  
PGM10576



```

13,P6.3,P10.1)
CONTINUE
CONTINUE
WRITE(KK,403)
FORMAT(' ',' STAGE STATION DISTANCE FROM STATION 1, STAGE 1')
J=1
K=1
SUM=0.0
WRITE(KK,402)K,J,SUM
FORMAT(' ','I7,I8,F15.4)
J=J+1
IF(J.EQ.1)J1=2
IF(J.EQ.2)J1=1
IF(J.EQ.1)K1=K-1
IF(J.EQ.2)K1=K
IF(J.EQ.1)J3=1
IF(J.EQ.2)J3=2
SUM=SUM+.11875*CHORDR(2,J1,K1)+1.1875*CHORDR(2,J,K)+.11875*CHORDR(
12,J3,K)
WRITE(KK,402)K,J,SUM
IF(J.EQ.2).AND.(K.EQ.KKK))GO TO 401
IF(J.EQ.1)GO TO 415
K=K+1
IF(J.EQ.2)J=0
GO TO 415
SUM=SUM+1.36625*CHORDF(2,2,KKK)+.11875*CHORDR(2,1,KKK)
J=3
WRITE(KK,402)K,J,SUM
WRITE(KK,404)
FORMAT('0',' LENGTH(IN) RADHUBINLET(IN) RADTIPINLET(IN) RADHUBE
1XIT(IN) RADTIPEXIT(IN) WTOPSHAFT(LB) WTOP DISCS+BLADES(LB)')
RAD1H=RADIUS(1,1,1)*12.
RAD1T=RADIUS(3,1,1)*12.
RAD2H=RADIUS(1,3,KKK)*12.
RAD2T=RADIUS(3,3,KKK)*12.
SUM1=0.0

```

98  
103

403

402  
415

401

404





```

SUM2=0.0
SHAFFT=SPECWT*PI*(RSHAFT*12.)*2*SUM
DO 410 K=1,KK
DO 409 J=1,2
409 SUM1=SUM1+BLADWT(J,K)
410 SUM2=SUM2+SUM1+WTDISC(K)
WRITE(KK,405)SUM,PAD1H,PAD1T,PAD2H,PAD2T,SHAFFT,SUM2
405 FORMAT(' ',F9.2,F15.2,5F16.2)
WRITE(7,779)CP,SO,HJ,PJ,VX,TOI,POI,E,W,KKK
779 FORMAT(F5.3,F10.4,F8.3,F10.4,F10.1,F10.1,F10.3,F5.3,F10.3,I2)
WRITE(7,780)GAMMA1,PI,SOMEGA,IET,TC,TIPCLA,GAMMA,KLM1,KLM2,PIA
780 FORMAT(F10.5,F10.3,F10.3,4F5.3,I2,2F10.3)
WRITE(7,782)((RADIUS(I,J,K),I=1,3),J=1,3),K=1,KKK)
782 FORMAT(8F10.4)
WRITE(7,781)((TEMP(J,K),J=1,3),K=1,KKK)
781 FORMAT(8F10.1)
WRITE(7,781)((TEMP(2,J,K),J=1,3),K=1,KKK)
WRITE(7,781)((TPFFS(J,K),J=1,3),K=1,KKK)
WRITE(7,781)((SPFFS(2,J,K),J=1,3),K=1,KKK)
WRITE(7,782)((SHC(2,J,K),J=1,3),K=1,KKK)
WRITE(7,782)((ALPHA(2,J,K),J=1,3),K=1,KKK)
WRITE(7,782)((BETA(2,J,K),J=1,3),K=1,KKK)
WRITE(7,782)((PEALMS(2,J,K),J=1,2),K=1,KKK)
WRITE(7,782)((PEIMR(2,J,K),J=1,2),K=1,KKK)
WRITE(7,782)((SBLADE(2,J,K),J=1,2),K=1,KKK)
WRITE(7,782)((PBLADE(2,J,K),J=1,2),K=1,KKK)
WRITE(7,782)((CHORDE(2,J,K),J=1,2),K=1,KKK)
WRITE(7,782)((ASEFCB(J,K),J=1,2),K=1,KKK)
WRITE(7,782)((PTOCHO(J,K),J=1,2),K=1,KKK)
IF(M4.FO.1)GO TO 399
DO 350 MK=1,5
OMEGAB(MK)=SOMEGA*SEFFD(MK)
DO 350 MJ=1,7
K=1
BETAC(1,K)=0.0
ALPHAC(1,K)=0.0

```

PGM10613  
PGM10614  
PGM10615  
PGM10616  
PGM10617  
PGM10618  
PGM10619  
PGM10620  
PGM10621  
PGM10622  
PGM10623  
PGM10624  
PGM10625  
PGM10626  
PGM10627  
PGM10628  
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PGM10630  
PGM10631  
PGM10632  
PGM10633  
PGM10634  
PGM10635  
PGM10636  
PGM10637  
PGM10638  
PGM10639  
PGM10640  
PGM10641  
PGM10642  
PGM10643  
PGM10644  
PGM10645  
PGM10646  
PGM10647  
PGM10648





```

TOTC(1,K)=TOI
VAXIAL(1,K)=VX
TSC(1,K)=TOI-VX**2/(2.*CP*HJ*GO)
POC(1,K)=POI
PSC(1,K)=POC(1,K)/(TOTC(1,K)/TSC(1,K))*GAMMA1
RHOC(1,K)=PSC(1,K)/(TSC(1,K)*RJ)
POI=POC(1,K)
TOI=TOI
GANG1=0.0
J=1
L=J+1
JJ=0
U=OMEGAB(MK)*RADIUS(2,L,K)
PHO2=PHO(2,L,K)
RMAC2=.15*REALMS(2,2,K)
IF(J.EQ.2)RMAC2=.15*FIMR(2,2,K)
BLADE2=SBLADE(2,2,K)
IF(J.EQ.2)BLADE2=SBLADE(2,2,K)
ANNUA=PI*(RADIUS(3,L,K)**2-RADIUS(1,L,K)**2)
JJ=JJ+1

```

300

```

IF(J.EQ.2)BLADE2=-BLADE2
CALL BLADE(GANG2,RMAC2,BLADE2,2)
IF(J.EQ.2)GANG2=-GANG2
IF(J.EQ.2)BLADE2=-BLADE2
CALL FIG(KLM,PMAC,VFI TOT,RMAC2,VFI TOT)
V2=SQRT(TC1)*VFI TOT
CALL FIG(KLM,PMAC,PSPT,PMAC2,PSPT)
T2=TOI*PSPT**2*(1./GAMMA1)
IF(PSPT2.LT.C.C)GO TO 390
BLADE1=SBLADE(2,1,K)
IF(J.EQ.2)BLADE1=SBLADE(2,1,K)
IF(J.EQ.1)KLEAP=1
IF(J.EQ.2)KLEAP=3
IF(J.EQ.2)KLEAP=3
IF(J.EQ.2)KLEAP=3
CALL PLOSS(GANG1,GANG2,PHO2,T2,BLADE1,PTOCHO(J,K),ASPEC(J,K),PMAC
12,V2,CHOPDR(2,J,K),J,K,KLEAP,Y,KMM,TEN,V)

```

300

PGM10649  
PGM10650  
PGM10651  
PGM10652  
PGM10653  
PGM10654  
PGM10655  
PGM10656  
PGM10657  
PGM10658  
PGM10659  
PGM10660  
PGM10661  
PGM10662  
PGM10663  
PGM10664  
PGM10665  
PGM10666  
PGM10667  
PGM10668  
PGM10669  
PGM10670  
PGM10671  
PGM10672  
PGM10673  
PGM10674  
PGM10675  
PGM10676  
PGM10677  
PGM10678  
PGM10679  
PGM10680  
PGM10681  
PGM10682  
PGM10683  
PGM10684



```

316 P02=PC1/(YT*(1.-PSPT2)+1.)
    Q2=WDOTP(MJ)*W*SORT(TO1)/(ANNULA*COS(GANG2*PI/180.)*PO2)
    IF(Q2.GT.WTAP(KLM1))Q2=WTAP(KLM1)*.99
    IF(RMAC2.GT.RMAC(KLM1))GO TO 316
    CALL FIG(KLM1,WTAP11,RMAC11,Q2,RMAC2P)
    CALL FIG(KLM1,WTAP11,VFOT11,Q2,VFOT2)
    CALL FIG(KLM1,WTAP11,PSPT11,Q2,PSPT2)
    GO TO 317
317 CALL FIG(KLM2,WTAP12,RMAC12,Q2,RMAC2P)
    CALL FIG(KLM2,WTAP12,VFOT12,Q2,VFOT2)
    CALL FIG(KLM2,WTAP12,PSPT12,Q2,PSPT2)
    P02=PO1/(YT*(1.-PSPT2)+1.)
    P2=PO2*PSPT2
    T2=TO1*PSPT2*(1./GAMMA1)
    RHO2=P2/(T2*PJ)
    V2=SORT(TO1)*VFOT2
    IF(ABS(RMAC2P/RMAC2-1.).LT.E)GO TO 302
    IF((RMAC2P.GT..9).AND.(JJ.GT.3))GO TO 304
    IF(JJ.EQ.10)GO TO 302
    IF(ABS(RMAC2P/RMAC2-1.).LT..0.03)GO TO 308
    RMAC2=RMAC2P
    GO TO 300
318 RMAC2=RMAC2+(RMAC2P-RMAC2)/2.
    GO TO 300
304 O2PRIM=1.0001*Q2
    IF((RMACP.GT.1.0))GO TO 300
    CALL FIG(KLM1,WTAP11,PSPT11,Q2PPRIM,PSPTP)
    GO TO 331
330 CALL FIG(KLM2,WTAP12,PSPT12,Q2PPRIM,PSPTP)
331 PO2PO1=1./(YT*(1.-PSPT)+1.)
    WPRIME=O2PPRIM*PO2PO1*ANNULA*COS(GANG2*PI/180.)*PO1/SORT(TO1)
    IF(WPRIME.LT.W*VDOCF(MJ))GO TO 305
    GO TO 315
305 WRITE(KK,310)J,K,SPEED(MK),PO1/100.
310 FORMAT(' ','DOWN',I2,' IN STAGE',I3,' FOR N/SORT(IO1) = ',F4.1,' FO
1R PO1 = ',F8.1,' CHOKED')

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PGM10685  
 PGM10686  
 PGM10687  
 PGM10688  
 PGM10689  
 PGM10690  
 PGM10691  
 PGM10692  
 PGM10693  
 PGM10694  
 PGM10695  
 PGM10696  
 PGM10697  
 PGM10698  
 PGM10699  
 PGM10700  
 PGM10701  
 PGM10702  
 PGM10703  
 PGM10704  
 PGM10705  
 PGM10706  
 PGM10707  
 PGM10708  
 PGM10709  
 PGM10710  
 PGM10711  
 PGM10712  
 PGM10713  
 PGM10714  
 PGM10715  
 PGM10716  
 PGM10717  
 PGM10718  
 PGM10719  
 PGM10720



```

CALL FIG(KLM2,WTAP12,PSPT12,Q2,PSPT2)
CALL FIG(KLM2,WTAP12,VEOT12,Q2,VEOT2)
V2=SORT(TO1)*VELTOT2
PO2=PO1/(YT*(1.-PSPT2)+1.)
P2=PO2*PSPT2
T2=TO1*PSPT2**(1./GAMMA1)
RHO2=P2/(T2*PJ)
GANG2=ARCCOS(W*WDOCTP(MJ)*SQRT(TO1)/(PO2*Q2*ANNUIA))*180./PI
IF(J.EQ.2)GO TO 307
J=J+1
ALPHAC(2,K)=GANG2
BETAC(2,K)=(ATAN(TAN(GANG2*PI/180.)-U/(V2*CCS(GANG2*PI/180.)))*180./PI
10./PI
RHO2(2,K)=RHO2
VWSQ=(V2*CCS(GANG2*PI/180.))*2*(1.+TAN(BETAC(2,K)*PI/180.))*2)
VAXIAL(2,K)=V2*CCS(GANG2*PI/180.)
TOTC(2,K)=TO1
TSC(2,K)=TO1-V2**2/(2.*CP-HJ*GO)
TO1=T2+VWSQ/(2.*CP-HJ*GO)
PSC(2,K)=P2
POC(2,K)=PC2
RVOTR=SORT(VWSQ)/SORT(TO1)
CALL FIG(KLM,VELTCT,PSPT,RVOTR,PSPT2)
PO1=P2/PSPT2
GANG1=BETAC(2,K)
GO TO 306
TSC(3,K)=TO1-V2**2/(2.*GO*HJ*CP)
VAXIAL(3,K)=V2*CCS(GANG2*PI/180.)
PSC(3,K)=PO2*PSPT2
RHO2(3,K)=RHO2
BETAC(3,K)=GANG2
U=OMEGAB(MK)*RADIUS(2,3,K)
ALPHAC(3,K)=(ATAN(TAN(GANG2*PI/180.)+U/(V2*CCS(GANG2*PI/180.)))*180./PI
180./PI
VELSQ=(V2*CCS(GANG2*PI/180.))*2*(1.+TAN(ALPHAC(3,K)*PI/180.))*2)
TOTC(3,K)=TSC(3,K)+VELSQ/(2.*GO*HJ*CP)

```

302

307

PGM10721  
PGM10722  
PGM10723  
PGM10724  
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PGM10726  
PGM10727  
PGM10728  
PGM10729  
PGM10730  
PGM10731  
PGM10732  
PGM10733  
PGM10734  
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PGM10736  
PGM10737  
PGM10738  
PGM10739  
PGM10740  
PGM10741  
PGM10742  
PGM10743  
PGM10744  
PGM10745  
PGM10746  
PGM10747  
PGM10748  
PGM10749  
PGM10750  
PGM10751  
PGM10752  
PGM10753  
PGM10754  
PGM10755  
PGM10756





```

POC(3,K)=PSC(3,K)*(TOTC(3,K)/TSC(3,K))*GAMMA1
IF(K.EQ.KKK)GO TO 309
L=K
K=K+1
ALPHAC(1,K)=ALPHAC(3,L)
VAXIAL(1,K)=VAXIAL(3,L)
BETAC(1,K)=BETAC(3,L)
TSC(1,K)=TSC(3,L)
POC(1,K)=POC(3,L)
PSC(1,K)=PSC(3,L)
TOTC(1,K)=TOTC(3,L)
RHOC(1,K)=RHOC(3,L)
GANG1=ALPHAC(1,K)
T01=TOTC(1,K)
P01=POC(1,K)
GO TO 308
309 P01P02(MJ)=P01/POC(3,KKK)
ENTMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*(1.-(POC(3,KKK)/POC(1,1)))+(1./GAMMA1))
ENSMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*(1.-(PSC(3,KKK)/POC(1,1)))+(1./GAMMA1))
WRITE(KK,311)SPEED(MK),P01P02(MJ)
FORMAT('0','FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF
1 CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE','CALCHL
2ATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CAL
3CULATIONS AT DESIGN POINT','FOR N/SQRT(T01)='',P5.2,' P01/PO2='
3,F5.2)
WRITE(KK,312)
FORMAT('0','STATION TOTAL TEMPERATURE STATIC TEMP(3) TOT
1AL PRES(P5I) STATIC PPES(P5I) DENSITY(LB/FT**3) ALPHA BETA')
FORMAT(' ',I4,I8,F11.1,F9.1,F10.1,F8.1,F9.1,F7.1,F7.1,F6.1,
13,4F6.1)
DO 320 K1=1,KKK
DO 320 J1=1,3
TPRES2=TPRES(J1,K1)/144.
TPRF3C=POC(J1,K1)/144.

```

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PGM10757
PGM10758
PGM10759
PGM10760
PGM10761
PGM10762
PGM10763
PGM10764
PGM10765
PGM10766
PGM10767
PGM10768
PGM10769
PGM10770
PGM10771
PGM10772
PGM10773
PGM10774
PGM10775
PGM10776
PGM10777
PGM10778
PGM10779
PGM10780
PGM10781
PGM10782
PGM10783
PGM10784
PGM10785
PGM10786
PGM10787
PGM10788
PGM10789
PGM10790
PGM10791
PGM10792

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320 SPRES2=SPRES(2,J1,K1)/144.
    SPRES=PSC(J1,K1)/144.
    IF(J1.EQ.3).AND.(K1.NE.KKK)GO TO 321
    WRITE(KK,313)J1,K1,TEMP(J1,K1),TOTC(J1,K1),STEM2(2,J1,K1),TSC(J1,
1K1),TPRES2,TPRES,SPRES2,SPRES,RHO(2,J1,K1),RHOC(J1,K1),ALPHA(2,J
11,K1),ALPHAC(J1,K1),BETA(2,J1,K1),BETAC(J1,K1)
    CONTINUE
321 WRITE(KK,345)
    FORMAT(' ','STAG' VX STATION 1 VX STATION 2 VX STATION 3')
322 WRITE(KK,344)(K1,(VAXIAL(J1,K1),J1=1,3),K1=1,KKK)
323 FORMAT(' ','I4,3F13.0')
    IF(MJ.NE.7)GO TO 350
    WRITE(KK,326)SPEED(MK)
324 FORMAT('0','FOR V/SOPT(TC1) EQUAL TO ',F5.2)
    WRITE(KK,328)
325 FORMAT('0',' PO1/PO2 TOTAL EFFICIENCY STATIC EFFICIENCY M*EORT(
1TO1)/PO1')
    WRITE(KK,327)(PO1PO2(I),ENTMJ(I),ENSMJ(I),WDOIP(I),I=1,7)
326 FORMAT(' ','F7.2,F15.4,F18.4,F17.3)
327 CONTINUE
328 CONTINUE
    IF(LKL.NE.NKL)GO TO 398
    STOP
    END
    SUBROUTINE PLOSS(ANGIN,ANGOUT,RHO,TEMP,BLADIN,PIUCHO,ASPECT,PERMACH
1,VELOT,CHCED,J,K,KLEAR,YI,KTM,TFM,U)
    COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
    COMMON/AREA1/TFI,TC,TIPCLA
    DIMENSION B180(9),B275(9),A3(9),B370(9),B365(9),B360(9),B350(9),B3
140(9),A5(5),B5(5),A15(5),B1540(5),B1550(5),B1560(5),B1570(5),A7(7)
1,B7(7),A811(11),B865(11),B860(11),B855(11),B850(11),B840(11),A8(9)
3,B830(9),PCA(7),PCB(7),YF2A(7),YF2B(7),TEX(7),TEY(7),ANG80(7),ANG7
40(6),ANG30(7),YF1A(6),YF1B(7),SI75(7),ANG47(4),SIDA(4),PI(9),TFM(K
5TM),U(KTM),BI70(9),BI65(9),BI60(9),BI55(9),BI50(9),BI40(9)
    DATA A3/.3,.4,.5,.6,.7,.8,.9,1.,1.1/
    DATA B180/.07149,.06468,.0594,.05889,.06128,.06553,.07285,.0834,.0

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PGM10793  
 PGM10794  
 PGM10795  
 PGM10796  
 PGM10797  
 PGM10798  
 PGM10799  
 PGM10800  
 PGM10801  
 PGM10802  
 PGM10803  
 PGM10804  
 PGM10805  
 PGM10806  
 PGM10807  
 PGM10808  
 PGM10809  
 PGM10810  
 PGM10811  
 PGM10812  
 PGM10813  
 PGM10814  
 PGM10815  
 PGM10816  
 PGM10817  
 PGM10818  
 PGM10819  
 PGM10820  
 PGM10821  
 PGM10822  
 PGM10823  
 PGM10824  
 PGM10825  
 PGM10826  
 PGM10827  
 PGM10828



19532/  
 DATA B275/.06979,.05872,.05021,.04766,.04817,.05140,.05651,.06383,  
 1.074/  
 DATA B276/.06994,.05532,.04511,.03966,.03830,.04,.0434,.04936,.057  
 187/  
 DATA B365/.06809,.05499,.04255,.03574,.03098,.02028,.03115,.03643,  
 1.04494/  
 DATA B366/.06638,.05362,.04068,.03149,.02689,.02417,.02451,.02879,  
 1.03404/  
 DATA B350/.06519,.05055,.03940,.0303,.02502,.02264,.02128,.02077,.  
 102247/  
 DATA B340/.06468,.0497,.03745,.02911,.02383,.02043,.01855,.01804,.  
 101872/  
 DATA A1/.3,.4,.5,.6,.7,.8,.9,1,.1/  
 DATA B170/.16112,.14186,.13640,.13981,.14868,.16198,.17596,.19006,  
 1.2077/  
 DATA B165/.15618,.13026,.11867,.11628,.12389,.12907,.13981,.15102,  
 1.1668/  
 DATA B160/.14834,.11799,.10571,.10196,.10401,.10827,.11475,.12351,  
 1.1329/  
 DATA B155/.14749,.11509,.09787,.08747,.08491,.08730,.09379,.10315,  
 1.1166/  
 DATA B150/.14408,.11168,.09122,.07843,.0731,.07502,.0798,.08781,.0  
 1987/  
 DATA B140/.13981,.10656,.0861,.07315,.06591,.0653,.0682,.07398,.03  
 1/  
 DATA A5/.4,.5,.6,.7,.8/  
 DATA B5/8.0576,6.8921,4.6043,1.4532,-2.3022/  
 DATA A15/.8,.85,.9,.95,1/  
 DATA B1540/-2.3022,-3.3813,-4.6762,-5.5396,-6.4023/  
 DATA B1550/-2.3044,-4.4804,-6.6187,-8.9928,-11.3669/  
 DATA B1560/-2.3066,-4.7626,-8.0432,-11.3669,-14.777/  
 DATA B1570/-2.3088,-5.3227,-9.6403,-14.1726,-19.1367/  
 DATA A7/-9,-6,-4,-2,0,.2,.4/  
 DATA B7/11.218,25.671,33.437,40.988,44.223,43.576,40.34/  
 DATA A811/-0,-.8,-.6,-.4,-.2,.0,.2,.4,.6,.8,1./

PGM10829  
 PGM10830  
 PGM10831  
 PGM10832  
 PGM10833  
 PGM10834  
 PGM10835  
 PGM10836  
 PGM10837  
 PGM10838  
 PGM10839  
 PGM10840  
 PGM10841  
 PGM10842  
 PGM10843  
 PGM10844  
 PGM10845  
 PGM10846  
 PGM10847  
 PGM10848  
 PGM10849  
 PGM10850  
 PGM10851  
 PGM10852  
 PGM10853  
 PGM10854  
 PGM10855  
 PGM10856  
 PGM10857  
 PGM10858  
 PGM10859  
 PGM10860  
 PGM10861  
 PGM10862  
 PGM10863  
 PGM10864



```

DATA B865/17.355,14.453,21.788,28.562,34.30,37.105,37.105,34.732,3
10.503,24.808,10.199/
DATA B866/8.025,11.563,17.397,22.435,26.318,28.993,30.547,29.899,2
17.57,23.384,19.199/
DATA B867/7.982,10.01,14.324,17.915,21.141,23.643,25.283,25.369,24
1.032,21.788,10.1990/
DATA B868/7.162,8.802,11.735,14.367,16.611,18.532,19.976,20.892,20
1.796,20.192,19.631/
DATA B869/6.04,6.903,8.845,10.355,11.865,13.289,14.626,15.532,16.4
181,17.56,18.984/
DATA A8/-9,-8,-6,-4,-2,0,2,4,6/
DATA B830/4.099,5.177,6.472,7.55,8.629,9.406,10.268,10.786,11.045/
DATA PCA/4,5,6,7,8,9,1/
DATA PCB/1,1069,1.0784,1.0483,1.0155,.9867,.9405,.9/
DATA YP2A/-4,-3,-2,-1,0,1,1.5/
DATA YP2B/6.1573,4.2565,2.7395,1.5663,1.2,0.922,4.5599/
DATA TEX/0,.02,.04,.06,.08,.1,.12/
DATA IFY/.91304,1.1,1.1087,1.239,1.3826,1.5304,1.6913/
DATA ANG89/40,50,60,65,70,75,80/
DATA ANG70/40,50,55,60,65,70/
DATA ANG47/40,50,60,70/
DATA ANG30/30,40,50,55,60,65,70/
U1=U(KTM)
IF(TEMP.GT.TPM(KTM)) GO TO 1
CALL FIG(KTM,TFM,U,TEMP,U1)
RF=((RHO*CHOED*VELOT)/(U1*12.))*1.37
YPI=0.2
A1=ANGIN
IF(J.EQ.1) A1=-ANGIN
A2=ANGOUT
IF(J.EQ.2) A2=-ANGOUT
B1=BLADIN*PI/180.
CINCI=ANGIN-BLADIN
IF(J.EQ.1) CINCI=BLADIN-ANGIN
IF((PTOCHO.IT.C.4).OE.(PTOCHO.GT.1.1)) GO TO 25
IF((ABS(ANGOUT).GT.80.).OP.(ABS(ANGOUT).IT.30.)) GO TO 25

```

PGM10865  
 PGM10866  
 PGM10867  
 PGM10868  
 PGM10869  
 PGM10870  
 PGM10871  
 PGM10872  
 PGM10873  
 PGM10874  
 PGM10875  
 PGM10876  
 PGM10877  
 PGM10878  
 PGM10879  
 PGM10880  
 PGM10881  
 PGM10882  
 PGM10883  
 PGM10884  
 PGM10885  
 PGM10886  
 PGM10887  
 PGM10888  
 PGM10889  
 PGM10890  
 PGM10891  
 PGM10892  
 PGM10893  
 PGM10894  
 PGM10895  
 PGM10896  
 PGM10897  
 PGM10898  
 PGM10899  
 PGM10900





```

1 IF (ABS(BLADIN).LT.0.5) GO TO 6
2 IF (ABS(ANGOUT).GT.7C.) GO TO 2
3 IF (ABS(ANGOUT).LT.4C.) GO TO 4
4 CALL FIG(9,AI,BI40,PTOCHO,YPNA(1))
5 CALL FIG(9,AI,BI50,PTOCHO,YPNA(2))
6 CALL FIG(9,AI,BI55,PTOCHO,YPNA(3))
7 CALL FIG(9,AI,BI60,PTOCHO,YPNA(4))
8 CALL FIG(9,AI,BI65,PTOCHO,YPNA(5))
9 CALL FIG(9,AI,BI70,PTOCHO,YPNA(6))
10 CALL FIG(6,ANG70,YPNA,B2,YPI)
11 GO TO 6
12 WRITE(KK,3)J,K
13 FORMAT(' ','THE ANGLE OUT WAS GREATER THAN 70 DEG AND THE DATA WAS
14 1 FROM 70 DEG FOR ROW',I3,'STAGE',I3)
15 GO TO 6
16 WRITE(KK,5)J,K
17 FOPMAT(' ','THE ANGLE OUT WAS LESS
18 1 FROM 40 DEG FOR ROW',I3,'STAGE',I3)
19 CALL FIG(9,AI,BI40,PTOCHO,YPI)
20 IF (ABS(ANGOUT).LT.4C.) GO TO 7
21 CALL FIG(9,A3,B340,PTOCHO,YPNA(1))
22 CALL FIG(9,A3,B350,PTOCHO,YPNA(2))
23 CALL FIG(9,A3,B360,PTOCHO,YPNA(3))
24 CALL FIG(9,A3,B365,PTOCHO,YPNA(4))
25 CALL FIG(9,A3,B370,PTOCHO,YPNA(5))
26 CALL FIG(9,A3,B375,PTOCHO,YPNA(6))
27 CALL FIG(9,A3,B380,PTOCHO,YPNA(7))
28 CALL FIG(7,ANG80,YPNA,A2,YPN)
29 GO TO 9
30 WRITE(KK,5)J,K
31 CALL FIG(9,A3,B340,PTOCHO,YPN)
32 YP=(YPN+(BLADIN/ANGOUT)*2*(YPI-YPN))*(PC/.2)*+(-BLADIN/ANGOUT)
33 IF (ABS(CINCI).LT.0.5) GO TO 13
34 IF (PTOCHO.GT.1.) GO TO 35
35 CALL FIG(7,PCA,PCB,PTOCHO,C75)
36 GO TO 34

```

PGM10901  
 PGM10902  
 PGM10903  
 PGM10904  
 PGM10905  
 PGM10906  
 PGM10907  
 PGM10908  
 PGM10909  
 PGM10910  
 PGM10911  
 PGM10912  
 PGM10913  
 PGM10914  
 PGM10915  
 PGM10916  
 PGM10917  
 PGM10918  
 PGM10919  
 PGM10920  
 PGM10921  
 PGM10922  
 PGM10923  
 PGM10924  
 PGM10925  
 PGM10926  
 PGM10927  
 PGM10928  
 PGM10929  
 PGM10930  
 PGM10931  
 PGM10932  
 PGM10933  
 PGM10934  
 PGM10935  
 PGM10936





35  
34

```
C75=PCF(7)-.4*(PTOCHO-1.)
A275=A2/C75
AB=BLADIN/A275
IF(J.EQ.2) AB=-BLADIN/A275
IF((AB.LT.-.9).OR.(AB.GT.1.))GO TO 25
IF((A275.GI.65.).AND.(AB.GT.0.4))GO TO 3
IF((A275.LT.40.).AND.(AB.GT.0.6))GO TO 11
CALL FIG(9,AB,B830,AB,SI75(1))
CALL FIG(11,A811,B840,AB,SI75(2))
CALL FIG(11,A811,B850,AB,SI75(3))
CALL FIG(11,A811,B855,AB,SI75(4))
CALL FIG(11,A811,B860,AB,SI75(5))
CALL FIG(11,A811,B865,AB,SI75(6))
CALL FIG(7,A7,B7,AB,SI75(7))
CALL FIG(7,ANG30,SI75,A275,SI751)
GO TO 12
```

9

```
WRITE(KK,10)J,K
```

10

```
FORMAT(' ','THE ANGLE OUT WAS GREATER THAN 65 DEG AND THE DATA WAS
1 FROM 65 DEG FOR POW',I3,'STAGE',I3)
CALL FIG(11,AP11,B865,AB,SI751)
GO TO 12
```

11

```
WRITE(KK,5)J,K
```

```
CALL FIG(9,A8,B830,AB,SI751)
```

12

```
IF(PTOCHO.GT.0.8)GO TO 13
```

```
CALL FIG(5,A5,B5,PTOCHO,DELTSI)
```

```
GO TO 16
```

13

```
IF(PTOCHO.GT.1.)GO TO 14
```

```
IF((A2.LT.40.).OR.(A2.GT.70.))GO TO 14
```

```
CALL FIG(5,A15,B1540,PTOCHO,SIDA(1))
```

```
CALL FIG(5,A15,B1550,PTOCHO,SIDA(2))
```

```
CALL FIG(5,A15,B1560,PTOCHO,SIDA(3))
```

```
CALL FIG(5,A15,B1570,PTOCHO,SIDA(4))
```

```
CALL FIG(4,ANG47,SIDA,A2,DELTSI)
```

```
GO TO 16
```

```
WRITE(KK,15)J,K
```

14

```
FORMAT(' ','THE P/C RATIO WAS GREATER THAN DATA FOR OFF INCIDENCE
```

15

PGM10937  
PGM10938  
PGM10939  
PGM10940  
PGM10941  
PGM10942  
PGM10943  
PGM10944  
PGM10945  
PGM10946  
PGM10947  
PGM10948  
PGM10949  
PGM10950  
PGM10951  
PGM10952  
PGM10953  
PGM10954  
PGM10955  
PGM10956  
PGM10957  
PGM10958  
PGM10959  
PGM10960  
PGM10961  
PGM10962  
PGM10963  
PGM10964  
PGM10965  
PGM10966  
PGM10967  
PGM10968  
PGM10969  
PGM10970  
PGM10971  
PGM10972



```

16 1CALCULATIONS, 3 VALUE FOR DELTA INCIDENCE IS TAKEN FOR P/C=1./,
20 FOR POW,I3, STAGE,I3)
   IF(A2.GT.70.)DELTSI=P1570(5)
   IF(A2.LT.70.)DELTSI=P1570(5)
   IF(A2.LT.60.)DELTSI=P1560(5)
   IF(A2.LT.50.)DELTSI=P1550(5)
   IF(A2.LT.40.)DELTSI=P1540(5)
   SI=DELTSI+SI751
   SIR=CINCI/SI
   IF((SIR.GT.1.5).OR.(SIR.LT.-4.))GO TO 17
   CALL FIG(7,YP2A,YP2B,SIR,YPC)
   YP=YP*YPC
   GO TO 18
   YP=YP*10.
   WRITE(KK,29)J,K
29 FORMAT(' ',THE DATA LIMITS WERE EXCEEDED IN POW,I3, STAGE,I3)
18 A6=ANGIN*PI/180.
   IF(J.EQ.1)A6=-A6
   A4=ANGOUT*PI/180.
   IF(J.EQ.2)A4=-A4
   ANGMEN=ATAN(.5*(TAN(A4)-TAN(A6)))
   CLSC=2.*(TAN(A6)+TAN(A4))*COS(ANGMEN)
   ZETA=CLSC**2*(COS(A4)**2/COS(ANGMEN)**3)
   YS=.0334*(COS(A4)/COS(P1))*ZETA/ASPECT
   IF(KLEAP.EQ.1)B=.0
   IF(KLEAP.FQ.2)B=.37
   IF(KLEAP.FQ.3)B=.47
   YK=B*(TIPCLF/CHORD)**.78*7*ETA/ASPECT
   IF(REMACH.GT.1.)YP=YP*(1.+60.*(REMACH-1.))**2)
   YPS=(YP+YS)*(PF/2.P5)**(-.2)
   YT=YPS+YK
   CALL FIG(7,TEX,TEY,TFC,YDEC)
   YT=YT*YDEC
24 RETURN
25 YT=0.5
   WRITE(KK,26)J,K

```



```

FORMAT(' ', THE INPUT DATA WAS GREATER THAN LIMITS OF PROGRAM & A
1 VALUE OF Y*=.5 WAS ASSIGNED FOR ROW', I2, ' STAGE', I2)
GO TO 24

```

```

END

```

```

FUNCTION REACT2(A,B)

```

```

COMMON PI,GO,PJ,HJ,GAMMA,VX,SCOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK

```

```

C COMPUTES REACTION OF ROTOR AT STATION 2

```

```

REACT2=1.+DHOS*GO*HJ/((SOMEGA*A)**2*2.)-(VX/(SOMEGA*A))*TAN(B)

```

```

RETURN

```

```

END

```

```

SUBROUTINE SECMOD(SMTANG,IC,SM,AREAC2)

```

```

COMMON PI,GO,PJ,HJ,GAMMA,VX,SCOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK

```

```

REAL A(10),B(10),C(10),N,D(8),E(8)

```

```

DATA A/10.,20.,30.,40.,50.,60.,80.,100.,120.,140./

```

```

DATA B/1.9417,1.8561,1.7705,1.6849,1.5992,1.5136,1.3424,1.1712,1.,
1.8288/

```

```

DATA C/1049.5,1024.2,991.52,945.67,896.36,827.879,661.212,458.788,
1270.30,66.67/

```

```

DATA D/0.,20.,40.,60.,80.,100.,120.,140./

```

```

DATA E/.02131,.04,.06423,.09525,.13097,.1695,.21305,.25692/

```

```

I=10

```

```

CALL FIG(I,A,B,SMTANG,N)

```

```

CALL FIG(I,A,C,SMTANG,X)

```

```

SM=(1./X)*(10.*IC)**N*2.25

```

```

CALL FIG(8,D,E,SMTANG,AREAC2)

```

```

RETURN

```

```

END

```

```

SUBROUTINE STOCRA(ANGLIN,ANGLOT,PCORAT)

```

```

COMMON PI,GO,PJ,HJ,GAMMA,VX,SCOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK

```

```

DIMENSION A40(5),B40(5),A50(4),B50(4),A(8),B(8),C(0),D(8),E(7),F(7)

```

```

1) ,G(6),H(6),X(5),Y(5)

```

```

DATA B40/40.,50.,60.,67.817,70./

```

```

DATA B50/50.,60.,67.817,70./

```

```

DATA A/0.,10.,20.,30.,40.,50.,60.,70./

```

```

DATA B/.72759,.71675,.68719,.65961,.62611,.58966,.55517,.53153/

```

```

DATA C/0.,10.,20.,30.,40.,50.,60.,70./

```

```

PGM11009
PGM11010
PGM11011
PGM11012
PGM11013
PGM11014
PGM11015
PGM11016
PGM11017
PGM11018
PGM11019
PGM11020
PGM11021
PGM11022
PGM11023
PGM11024
PGM11025
PGM11026
PGM11027
PGM11028
PGM11029
PGM11030
PGM11031
PGM11032
PGM11033
PGM11034
PGM11035
PGM11036
PGM11037
PGM11038
PGM11039
PGM11040
PGM11041
PGM11042
PGM11043
PGM11044

```





```

3 DATA D/.75517,,74138,,71478,,63325,,64375,,61527,,57980,,55419/
DATA E/O.,10.,20.,30.,40.,50.,60./
DATA F/.83399,,81626,,79458,,76108,,71675,,68719,,64581/
DATA G/O.,10.,20.,30.,40.,50./
DATA H/.90985,,88424,,85074,,81626,,76995,,74138/
DATA X/O.,10.,20.,30.,40./
DATA Y/.96601,,93547,,89015,,84778,,78571/
IF (ANGLOT.GT.70.) GO TO 3
IF ((ANGLOT.GF.67.817).AND.(ANGLIN.LE.70.)) GO TO 4
IF ((ANGLOT.GF.60.).AND.(ANGLIN.LE.60.)) GO TO 5
IF ((ANGLOT.GF.50.).AND.(ANGLIN.LE.50.)) GO TO 6
IF ((ANGLOT.GE.40.).AND.(ANGLIN.LE.40.)) GO TO 7
CALL FIG(8,A,B,ANGLIN,PCPAT)
GO TO 2
4 CALL FIG(8,C,D,ANGLIN,A681)
CALL FIG(8,A,B,ANGLIN,A682)
PCPAT=A681-(A681-A682)*(70.-ANGLOT)/2.183
GO TO 2
5 CALL FIG(7,F,F,ANGLIN,A601)
CALL FIG(8,C,D,ANGLIN,A602)
PCPAT=A601-(A601-A602)*(70.-ANGLOT)/10.
GO TO 2
6 CALL FIG(6,G,H,ANGLIN,A50(1))
CALL FIG(7,F,F,ANGLIN,A50(2))
CALL FIG(8,C,D,ANGLIN,A50(3))
CALL FIG(8,A,B,ANGLIN,A50(4))
CALL FIG(4,B50,A50,ANGLOT,PCPAT)
GO TO 2
7 CALL FIG(5,X,Y,ANGLIN,A40(1))
CALL FIG(6,S,H,ANGLIN,A40(2))
CALL FIG(7,F,F,ANGLIN,A40(3))
CALL FIG(8,C,D,ANGLIN,A40(4))
CALL FIG(8,A,B,ANGLIN,A40(5))
CALL FIG(5,B40,A40,ANGLOT,PCPAT)
2 RETURN
END

```

```

PGM11045
PGM11046
PGM11047
PGM11048
PGM11049
PGM11050
PGM11051
PGM11052
PGM11053
PGM11054
PGM11055
PGM11056
PGM11057
PGM11058
PGM11059
PGM11060
PGM11061
PGM11062
PGM11063
PGM11064
PGM11065
PGM11066
PGM11067
PGM11068
PGM11069
PGM11070
PGM11071
PGM11072
PGM11073
PGM11074
PGM11075
PGM11076
PGM11077
PGM11078
PGM11079
PGM11080

```





SUBROUTINE BLADE(A,B,C,J)  
 C CALCULATES BLADE OF GAS ANGLES FOR ROTOR AND STATORS  
 DIMENSION D(6),F(6)  
 DATA D/24.34,40.50,60.70,80.70,943,78.868/  
 DATA F/36.,47.717,53.962,62.453,70.943,78.868/  
 I=6

IF(J.EQ.2)GO TO 3  
 IF(B.LT.1.)GO TO 1  
 C=?

GO TO 2  
 1 CALL FIG(Z,D,F,A,C)  
 IF(B.LT.0.5)GO TO 2  
 C=C-((B-.5)/.5)\*(C-A)  
 2 RETURN  
 3 IF(B.LT.1.)GO TO 5  
 A=C

GO TO 2  
 5 CALL FIG(F,P,D,C,A)  
 IF(P.LT.0.5)GO TO 2  
 A=A+((B-.5)/.5)\*(C-A)  
 GO TO 2  
 END

SUBROUTINE FIG(I,D,F,X,Y)  
 DIMENSION D(I),F(I),DD(100),FF(100),A(4),B(4)  
 IF(((X.GT.D(I)).AND.(D(1).LT.D(I))).OR.((X.LT.D(1)).AND.(D(I).GE.D(1(1)).OR.((X.LT.D(I)).AND.(D(1).GT.D(I))).OR.((2X.GT.D(1)).AND.(D(1).GT.D(I))))GO TO 31  
 IF(D(1).GT.D(I))GO TO 9  
 IF(I.EQ.4)GO TO 2  
 N=I-1  
 J=2

IF(X.GE.D(N))GO TO 1  
 IF(X.LE.D(J))GO TO 2  
 L=3  
 IF(X.LT.D(L))GO TO 3  
 L=L+1

PGM11081  
 PGM11082  
 PGM11083  
 PGM11084  
 PGM11085  
 PGM11086  
 PGM11087  
 PGM11088  
 PGM11089  
 PGM11090  
 PGM11091  
 PGM11092  
 PGM11093  
 PGM11094  
 PGM11095  
 PGM11096  
 PGM11097  
 PGM11098  
 PGM11099  
 PGM11100  
 PGM11101  
 PGM11102  
 PGM11103  
 PGM11104  
 PGM11105  
 PGM11106  
 PGM11107  
 PGM11108  
 PGM11109  
 PGM11110  
 PGM11111  
 PGM11112  
 PGM11113  
 PGM11114  
 PGM11115  
 PGM11116



```

3      GO TO 4
      L=L-2
      DO 8 K=1,4
        A(K)=D(L)
        B(K)=F(L)
        L=L+1
      GO TO 7
1      IJ=I-4
      DO 5 II=1,4
        A(II)=D(IJ)
        B(II)=F(IJ)
        IJ=IJ+1
      GO TO 7
5      GO TO 7
      DO 6 II=1,4
        A(II)=D(II)
        B(II)=F(II)
      CALL BK(A,B,X,Y)
      RETURN
32     DD(1)=D(I)
      FF(1)=F(I)
      J=I
9      DO 15 M=2,I
      J=J-1
      DD(M)=D(J)
      FF(M)=F(J)
15     DO 16 M=1,I
      D(M)=DD(M)
      F(M)=FF(M)
16     GO TO 17
      IF((X.GT.D(I)).AND.(D(1).LT.D(I)).OR.((X.LT.D(I)).AND.(D(1).GT.D
1(I))))Y=F(I)
31     IF(Y.NF.F(I))Y=F(1)
      GO TO 32
      END
      SUBROUTINE BK(X,Y,XARG,YOUT)
      DIMENSION X(4),Y(4)
PGM111117
PGM111118
PGM111119
PGM111120
PGM111121
PGM111122
PGM111123
PGM111124
PGM111125
PGM111126
PGM111127
PGM111128
PGM111129
PGM111130
PGM111131
PGM111132
PGM111133
PGM111134
PGM111135
PGM111136
PGM111137
PGM111138
PGM111139
PGM111140
PGM111141
PGM111142
PGM111143
PGM111144
PGM111145
PGM111146
PGM111147
PGM111148
PGM111149
PGM111150
PGM111151
PGM111152
127

```







```

1LE,X(4)).AND.(XARG.GT.X(3)).AND.((YOUT.LT.Y(3)).OR.(YOUT.GT.Y(4)))
2) YOUT=Y(3)-(Y(3)-Y(4))*(XARG-X(3))/(X(4)-X(3))
RETURN
END
SUBROUTINE TESTP(I,N)
C DETERMINES IF AN INTEGER IS A PRIME NO. GIVES AN OUTPUT OF 1 IF PRIME 0 IF NOT
INTEGER TEST
I=0
TEST=2
IF(N/TEST*TEST.NE.N)GO TO 8
GO TO 11
TEST=1
TEST=TEST+2
IF((N/TEST*TEST.EQ.N).AND.(TEST.NE.N))GO TO 11
IF(TEST.EQ.N)GO TO 10
GO TO 3
I=1
RETURN
END
FUNCTION REACT1(A,B)
C COMPUTES REACTION OF ROTOR AT STATION 2
COMMON PI,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KF
REACT1=1.-DHOS*GO*HJ/((SOMEGA*4)*2*2.)-(VX/(SOMEGA*A))*TAN(R)
RETURN
END
SUBROUTINE TDD2(A,B,C,D,E,F,G,H)
C DETERMINES TEMPERATURE,PRESSURE, AND DENSITY AT STATION 2
COMMON PI,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
V=VX/COS(G)
IF(V.LT.0.)V=-V
C=A-V**2/(CP*GO*HJ**2.)
IF(C.LT.0.5*A)C=0.5*A
F=E*(C/B)**GAMMA2
D=F*(A/C)**GAMMA1
H=F/(C*PJ)
RETURN

```





```

END
SUBROUTINE TP1(A,B,C,D,E,F)
C COMPUTES TEMPERATURE,PRESSURE, AND DENSITY AT STATION 1 AND 3
COMMON PI,GO,EJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
V=VX/COS(C)
IF(V.LT.C.C)V=-V
B=A-V**2/(CP*GO*HJ*2.)
IF(B.LT.0.5*I)B=0.5*I
E=D*(B/A)**GAMMA1
F=E/(PJ*B)
RETURN
END
FUNCTION ANGLE2(A,B)
C DETERMINES ANGLE ALPHA 2
COMMON PI,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
ANGLE2=ATAN(DHOS*GO*HJ/(VX*SOMEGA*A)+TAN(B))
RETURN
END
FUNCTION ANGLE(A,E)
C DETERMINES RELATIVE ANGLE RETA
COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
ANGLEB=ATAN(TAN(A)-SOMEGA*B/VX)
RETURN
END
SUBROUTINE DEN2(A,B,C,D)
COMMON PI,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
C COMPUTES INTEGRAL OF DENSITY TIMES AREA TO GET AXIAL VELOCITY BY SIMPSON'S RULE
REAL A(3),B(3)
DENSIT=(1./3.)*((A(3)-A(1))/2.)*(A(1)*R(1)+4.*A(2)*B(2)+A(3)*B(3))
D=C/(2.*PI*DENSIT)
RETURN
END
FUNCTION ANGLE(A,B,C)
C DETERMINES ANGLE AT ANOTHER LOCATION BY FREE VORTEX DEFINITION OF F*V (THETA)=K
ANGLE=ATAN(A*TAN(B)/C)
RETURN

```



END

\$ENTRY

9000 HP SINGLE STAGE GAS TURBINE ENGINE

00 1 60 111.333.27159000. 119.42 .2 .02 .015 1767.8 312.48

1.8 .62161.418.1 20000. 50 0.289 20000000. 20000. 93000.

.9168.08333 1.680130000. .000 .292 0 0 1.25 .1

0.020.040.060.080.100.120.140.160.170.200.220.240.260.280.300.320.340.360.380.40

0.420.440.460.480.500.520.540.560.580.600.620.640.660.680.700.720.740.760.780.80

0.820.840.860.880.900.920.940.960.981.001.021.041.061.081.101.121.141.161.181.20

0.01811 0.03596 0.05388 0.07171 0.08945 0.10706 0.12448 0.14154 0.15885 0.17562

0.19226 0.20849 0.22446 0.24049 0.25612 0.27135 0.28590 0.30026 0.31435 0.32763

0.34118 0.35386 0.36627 0.37821 0.38948 0.40061 0.41108 0.42114 0.43067 0.43979

0.44838 0.45669 0.46421 0.47159 0.47796 0.48433 0.49010 0.49560 0.50030 0.50459

0.50815 0.51170 0.51479 0.51734 0.51948 0.52123 0.52257 0.52337 0.52391 0.52418

0.52391 0.52337 0.52257 0.52136 0.51989 0.51801 0.51599 0.51358 0.51076 0.50795

0.99973 0.99893 0.99760 0.99574 0.99335 0.99040 0.98700 0.98317 0.97870 0.97380

0.96835 0.96260 0.95630 0.94950 0.94200 0.93440 0.92650 0.91800 0.90920 0.90030

0.89080 0.88100 0.87060 0.86000 0.84950 0.83860 0.82720 0.81570 0.80400 0.79220

0.65500 0.64080 0.62760 0.61410 0.60250 0.59000 0.57550 0.56500 0.55100 0.54000

0.52800 0.51700 0.50400 0.49100 0.48000 0.46820 0.45650 0.44530 0.43330 0.42250

0.96151 1.91184 2.86589 3.82740 4.77028 5.72284 6.67004 7.61008 8.56787 9.50329

10.4528711.3666812.3132813.2673414.2288415.1456416.0475016.9941117.9034418.82924

19.7519220.6314421.5407722.4501023.3370924.2762525.1632126.0501926.9520727.80022

28.6589229.5459030.4105231.2475032.0950232.9447233.8019034.6516035.4739436.20883

37.0516437.9396138.7659539.6082240.3982841.1436542.0306142.7088943.4169344.16234

44.9822245.6903146.3611347.1735747.8518448.5072049.3425650.0282750.7438251.39220

400. 600. 800. 1000. 1200. 1400. 1600. 1800. 2000. 2200. 2400.

100. 135. 166. 192. 218. 242. 264. 284. 302. 320. 338.

13673 HP SINGLE STAGE AIRCRAFT GAS TURBINE ENGINE

10 2 60 111.333.274 13673. 100. .2 .02 .015 2960. 38.2

2.8 1.034 2.0 0.0 13856. 50 2.289 20000000. 45000. 93000.

0.9 .1 6. 130000. -.5 -.25 0 2 1.25 .1

/\*EOJ \*\*\*\*\*



\$\*  
\$JOB  
\$\*

VICTOR ENDO MAY 9, 1977  
SIPP, TIME=30  
VICOP ENDO MAY 9, 1977

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COMMON PI, GC, PJ, HJ, GAMMA, VX, SCOMEGA, DHOS, CP, GAMMA12, GAMMA1, KK
COMMON/APSA1/TET, TC, TIPCLA
DIMENSION SPEEDS(3,3,10), RADIUS(3,3,10), ALPHA(3,3,10), BETA(3,3,10)
DIMENSION DEN(3), PAD(3), TTEMP(3,10), TPRES(3,10), STEMP(3,3,10)
DIMENSION PEACI(3,10), PHO(3,3,10), REALMS(3,2,10), RELMP(3,2,10), TTE
1MPR(3,2,12), VAXIAL(3,12)
DIMENSION RBLADE(3,2,10), SBLADE(3,2,10), CHOFR(3,2,10), SIGMAB(2,10
1), SIGMAC(12), BLADEH(2,12), PTOCHO(2,12), ASPECR(2,12), TEM(50), V(50)
DIMENSION SPEED(5), WDOTP(7), OMEGAB(5), POIS(7), RUOC(3,10), BETAC(3,1
10), ALPHAC(3,10), TOTC(3,10), TSC(3,10), POC(3,10), OSC(3,10), PPOC(3,10
3), VEL(3,10), PVEL(3,10), EMAC(100), WTAP(100), PSPT(100), VELTOT(100), P
101PO2(7), ENTMJ(7), ENSMJ(7), PTOC(3,10), A(6), B(6)
DIMENSION WTAP11(70), WTAP12(70), MAC11(70), RMAC12(70), VROT11(70), V
1EOT12(70), PSPT11(70), PSPT12(70)
DATA SPEED/.4,.6,.8,1.,1.1/
DATA WDOTP/.6,.8,.9,.95,.99,1.,1.01/
KK=6
LL=5
IKL=0
PEAD(LL,779) CP, GO, HJ, RJ, VX, TOI, POI, E, W, KKK
FORMAT(F5.3, F10.4, F8.3, F10.4, F10.1, F10.1, F10.3, F5.3, F10.3, I2)
PEAD(LL,780) GAMMA1, PI, SCOMEGA, TET, TC, TIPCLA, GAMMA, KLM1, KLM, KCTM
FORMAT(F10.5, F10.8, F10.3, 4F5.3, I2, I3, I2)
READ(LL,782) ((RADIUS(I,J,K), I=1,3), J=1,3), K=1, KKK)
FORMAT(8F10.4)
READ(LL,781) ((TTEMP(J,K), J=1,3), K=1, KKK)
FORMAT(8F10.1)
READ(LL,781) ((STEMP(2,J,K), J=1,3), K=1, KKK)
READ(LL,781) ((TPRES(J,K), J=1,3), K=1, KKK)
READ(LL,781) ((SPEED(2,J,K), J=1,3), K=1, KKK)
READ(LL,782) ((PHO(2,J,K), J=1,3), K=1, KKK)
READ(LL,782) ((ALPHA(2,J,K), J=1,3), K=1, KKK)
READ(LL,782) ((BETA(2,J,K), J=1,3), K=1, KKK)

```

779

780

782

781

## COMPUTER PROGRAM #2

PGM200001  
PGM200002  
PGM200003  
PGM200004  
PGM200005  
PGM200006  
PGM200007  
PGM200008  
PGM200009  
PGM200010  
PGM200011  
PGM200012  
PGM200013  
PGM200014  
PGM200015  
PGM200016  
PGM200017  
PGM200018  
PGM200019  
PGM200020  
PGM200021  
PGM200022  
PGM200023  
PGM200024  
PGM200025  
PGM200026  
PGM200027  
PGM200028  
PGM200029  
PGM200030  
PGM200031  
PGM200032  
PGM200033  
PGM200034  
PGM200035  
PGM200036





```

398 READ(LL,782) ((PEALMS(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((RELMR(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((SBLADE(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((RBLADE(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((CHORDP(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((ASPECE(J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((PPOCHC(J,K),J=1,2),K=1,KKK)
    LKL=LKL+1
    IF(LKL.GT.1)GO TO 299
    READ(LL,200) (RMAC(I),I=1,60)
    READ(LL,201) (WTAP(I),I=1,60)
    READ(LL,201) (PSPT(I),I=1,60)
    READ(LL,201) (VELTOT(I),I=1,60)
    FORMAT(10F8.5)
    FORMAT(20F4.2)
    READ(LL,227) (TFM(I),I=1,KTM)
    READ(LL,227) (V(I),I=1,KTM)
    FORMAT(20F5.C)
    KLM=60
    KLM2=KLM-KLM1
    DO 230 I=1,KLM1
    II=I+KLM1-1
    RMAC11(I)=RMAC(I)
    WTAP11(I)=WTAP(I)
    PSPT11(I)=PSPT(I)
    VEOT11(I)=VELTOT(I)
    IF(I.GT.KLM2)GO TO 230
    RMAC12(I)=RMAC(II)
    WTAP12(I)=WTAP(II)
    PSPT12(I)=PSPT(II)
    VEOT12(I)=VELTOT(II)
    CONTINUE
    DO 350 MK=1,5
    OMEGAB(MK)=SOMEGA*SPEED(MK)
    DO 350 MJ=1,7
    K=1

```

```

PGM200037
PGM200038
PGM200039
PGM200040
PGM200041
PGM200042
PGM200043
PGM200044
PGM200045
PGM200046
PGM200047
PGM200048
PGM200049
PGM200050
PGM200051
PGM200052
PGM200053
PGM200054
PGM200055
PGM200056
PGM200057
PGM200058
PGM200059
PGM200060
PGM200061
PGM200062
PGM200063
PGM200064
PGM200065
PGM200066
PGM200067
PGM200068
PGM200069
PGM200070
PGM200071
PGM200072

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```

BETAC(1,K)=0.0
ALPHAC(1,K)=0.0
TOTC(1,K)=TOI
TSC(1,K)=TOI-VX**2/(2.*CP*HJ*GO)
POC(1,K)=POI
PSC(1,K)=POC(1,K)/(TOTC(1,K)/TSC(1,K))*GAMMA1
VAX=AL(1,K)=VX
RHOC(1,K)=PSC(1,K)/(TSC(1,K)*RJ)
POI=PCC(1,K)
TOI=TOI
GANG1=0.0

```

308  
306

```

J=1
L=J+1
JJ=0
U=OMEGAB(MK)*RADIUS(2,L,K)
PHO2=PHO(2,L,K)
RMAC2=.2*FEA1MS(2,2,K)
IF(J.EQ.2)PMAC2=.2*FEELMR(2,2,K)
BLADE2=SBLADE(2,2,K)
IF(J.EQ.2)BLADE2=RPLADE(2,2,K)
ANNUA=PI*(RADIUS(3,L,K)**2-RADIUS(1,L,K)**2)
JJ=JJ+1

```

300

```

IF(J.EQ.2)BLADE2=-BLADE2
CALL BLADE(GANG2,RMAC2,BLADE2,2)
IF(J.EQ.2)GANG2=-GANG2
IF(J.EQ.2)BLADE2=-BLADE2
CALL FIG(KLM,PMAC,VFI TOI,PMAC2,VFI TOT2)
V2=SQRT(TOI)*VF TOT2
CALL FIG(KLM,RMAC,PSIT,PMAC2,PSPTOT)
T2=TOI*PSPI2**2*(1./GAMMA1)
BLADE1=SBLADE(2,1,K)
IF(J.EQ.2)BLADE1=RPLADE(2,1,K)
IF(J.EQ.1)KLEAR=1
IF(J.EQ.2)KLEFAP=1
CALL PLOSS(GANG1,GANG2,PHO2,T2,BLADE1,PTCCHC(J,K),ASPECB(J,K),BMAC
12,V2,CHORDR(2,J,K),J,K,KLEAF,YT,KLM,TFM,V)

```



```

PGM20109
PGM20110
PGM20111
PGM20112
PGM20113
PGM20114
PGM20115
PGM20116
PGM20117
PGM20118
PGM20119
PGM20120
PGM20121
PGM20122
PGM20123
PGM20124
PGM20125
PGM20126
PGM20127
PGM20128
PGM20129
PGM20130
PGM20131
PGM20132
PGM20133
PGM20134
PGM20135
PGM20136
PGM20137
PGM20138
PGM20139
PGM20140
PGM20141
PGM20142
PGM20143
PGM20144
135

PO2=PO1/(YT*(1.-PSP*2)+1.)
O2=WDOT2(MJ)*W*SORT(TO1)/(ANNULA*COS(GANG2*PI/180.)*PC2)
IF(O2.GT.WTAP(KLM1))O2=WTAP(KLM1)*.99
IF(RMAC2.GT.PMAC(KLM1))GO TO 316
CALL FIG(KLM1,WTAP11,RMAC11,O2,RMAC2P)
CALL FIG(KLM1,WTAP11,PSP11,O2,PSP*2)
CALL FIG(KLM1,WTAP11,VEOT11,O2,VEOT*2)
GO TO 317
316 CONTINUE
CALL FIG(KLM2,WTAP12,RMAC12,O2,RMAC2P)
CALL FIG(KLM2,WTAP12,PSP12,O2,PSP*2)
CALL FIG(KLM2,WTAP12,VEOT12,O2,VEOT*2)
317 PO2=PO1/(YT*(1.-PSP*2)+1.)
P2=PO2*PSP*2
IF(PSP*2.LT.0.0)GO TO 399
T2=TO1*PSP*2*(1./GAMMA1)
RH02=P2/(T2*RJ)
V2=SORT(TO1)*VEOT*2
IF(ABS(RMAC2P/RMAC2-1.).LT.5)GO TO 302
IF((RMAC2P.GT.0.9).AND.(JJ.GT.3))GO TO 304
IF(JJ.EQ.10)GO TO 302
IF(ABS(RMAC2P/RMAC2-1.).LT.0.03)GO TO 318
RMAC2=PMAC2P
GO TO 300
318 RMAC2=PMAC2+(RMAC2P-PMAC2)/2.
GO TO 300
304 Q2PRIM=1.0001*Q2
IF((RMAC2P.GT.1.0))GO TO 320
CALL FIG(KLM1,WTAP11,PSP11,O2PRIM,PSP*2)
GO TO 331
330 CALL FIG(KLM2,WTAP12,PSP12,O2PRIM,PSP*2)
331 PO2PO1=1./(YT*(1.-PSP*2)+1.)
WPRIME=O2PRIM*PO2PO1*ANNULA*COS(GANG2*PI/180.)*PC1/SORT(TO1)
IF(WPRIME.LT.W*WDO*1(MJ))GO TO 305
GO TO 315
305 WRITE(KK,310)J,K,STEEP(MK),POI/144.

```



```

310  FORMAT(' ',RON,I2,' IN STAGE',I3,' FOR N/SORT(TO1) = ',P4.1,' ZO
      1R PO1 = ',P8.1,' CHECKED')
      CALL FIG(KLM2,WTA,P12,VEOT12,Q2,VZIO22)
      CALL FIG(KLM2,WTA,P12,PSP12,Q2,PSP12)
      V2=SQRT(TO1)*VFO12
      PO2=PO1/(VT*(1.-PSP12)+1.)
      T2=TO1*PSP12*(1./GAMMA1)
      P2=PO2*PSP12
      RH02=P2/(T2*PJ)
      GANG2=ARCCOS(W*WDOCP(MJ)*SQRT(TO1)/(PO2*Q2*ANNULA))*180./PI
      IF(J.EQ.2) GO TO 307
      J=J+1
      ALPHAC(2,K)=GANG2
      BETAC(2,K)=(ATAN(TAN(GANG2*PI/180.))-U/(V2*CCS(GANG2*PI/180.)))*18
      10./PI
      RHOC(2,K)=PHC2
      VWSQ=(V2*CCS(GANG2*PI/180.))*2*(1.+(TAN(BETAC(2,K)*PI/180.))*2)
      VAXIAL(2,K)=V2*CCS(GANG2*PI/180.)
      TOTC(2,K)=TO1
      TSC(2,K)=TC1-V2**2/(2.*CP*HJ*GO)
      TO1=T2+VWSQ/(2.*CP*HJ*GO)
      PSC(2,K)=P2
      POC(2,K)=PO2
      PVTOTP=SQRT(VWSQ)/SQRT(TO1)
      CALL FIG(KLM,VOT10,PSP1,VVOTR,PSP12)
      PO1=P2/PSP12
      GANG1=BETAC(2,K)
      GO TO 306
      TSC(3,K)=TC1-V2**2/(2.*GO*HJ*CP)
      PSC(3,K)=PO2*PSP12
      RHOC(3,K)=PHC2
      BETAC(3,K)=GANG2
      U=OMEGAB(MK)*RADIIUS(2,3,K)
      VAXIAL(3,K)=V2*CCS(GANG2*PI/180.)
      ALPHAC(3,K)=(ATAN(TAN(GANG2*PI/180.))+U/(V2*CCS(GANG2*PI/180.)))*1
      180./PI
302
307

```



```

VBLISO=(V2*COS(GANG2*PI/180.))*2*(1.+TAN(ALPHAC(S,K)*PI/180.))*2)
TOTC(3,K)=TSC(3,K)+VBLISO/(2.*GO*HJ*CP)
POC(3,K)=PSC(3,K)*(TOTC(3,K)/TSC(3,K))*GAMMA1
I2(K,EQ,KKK)GO EC 309
L=K
K=K+1
ALPHAC(1,K)=ALPHAC(3,L)
BETAC(1,K)=BETAC(3,L)
TOTC(1,K)=TOTC(3,L)
TSC(1,K)=TSC(3,L)
POC(1,K)=POC(3,L)
PSC(1,K)=PSC(3,L)
VAXIAL(1,K)=VAXIAL(3,L)
RHOC(1,K)=RHOC(3,L)
GANG1=ALPHAC(1,K)
TO1=TOTC(1,K)
PO1=POC(1,K)
GO TO 308

309 PO1PO2(MJ)=PO1/POC(3,KKK)
ENMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*(1.-(POC(3,KKK)/POC(1,1)))+(1./GAMMA1))
ENSMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*(1.-(PSC(3,KKK)/POC(1,1)))+(1./GAMMA1))
WRITE(KK,311)SPEED(MK),PO1PO2(MJ)
FORMAT('0','FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF
1 CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE','CALCUL
2ATIONS OF IMPROVED WINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CAL
3CULATIONS AT DESIGN POINT','POP N/SORT(TO1)=' ,F5.2,' PO1/PO2='
3,F5.2)
WRITE(KK,345)
FORMAT(' ','STAGE VX STATION 1 VX STATION 2 VX STATION 3')
WRITE(KK,344)(K1,(VAXIAL(J1,K1),J1=1,3),K1=1,KKK)
FORMAT(' ','I4,3F13.0)
WRITE(KK,312)
FORMAT('0','STATION STAGE TOTAL TEMPERATURE STATIC TEMP(R) TOP
1AL PRES(PSI) STATIC PRES(PSI) DENSITY(IB/FT**3) ALPHA RETA')

```

PGM20181  
 PGM20182  
 PGM20183  
 PGM20184  
 PGM20185  
 PGM20186  
 PGM20187  
 PGM20188  
 PGM20189  
 PGM20190  
 PGM20191  
 PGM20192  
 PGM20193  
 PGM20194  
 PGM20195  
 PGM20196  
 PGM20197  
 PGM20198  
 PGM20199  
 PGM20200  
 PGM20201  
 PGM20202  
 PGM20203  
 PGM20204  
 PGM20205  
 PGM20206  
 PGM20207  
 PGM20208  
 PGM20209  
 PGM20210  
 PGM20211  
 PGM20212  
 PGM20213  
 PGM20214  
 PGM20215  
 PGM20216







```

FORMAT(' ',I4,I8,F11.1,E8.1,F10.1,F8.1,F9.1,F7.1,F6.1,F5.1,F4.1,F3.1,F2.1,F1.1)
13,476.1)
DO 320 K1=1,KKK
DO 320 J1=1,3
TPRES2=TPRES(J1,K1)/144.
TPRESC=POC(J1,K1)/144.
SPRES2=SPRES(2,J1,K1)/144.
SPRESC=PSC(J1,K1)/144.
IF((J1.EQ.3).AND.(K1.NF.KKK))GO TO 320
WRITE(KK,313)J1,K1,TEMP(J1,K1),TOTC(J1,K1),STEMP(2,J1,K1),TSC(J1,
1K1),TPRES2,TPRESC,SPRES2,SPRESC,RHO(2,J1,K1),CHOC(J1,K1),ALPHA(2,J
1,K1),ALPHAC(J1,K1),BETA(2,J1,K1),BETAC(J1,K1)
CONTINUE
320 IF(MJ.NE.7)GO TO 35C
WRITE(KK,326)SPEED(MK)
FORMAT('O','FOR N/SORT(TO1) EQUAL TO ',F5.2)
WRITE(KK,328)
FORMAT('O','FO1/EO2 TOTAL EFFICIENCY STATIC EFFICIENCY M*SORT(
1TO1)/PC1')
WRITE(KK,327)(FO1EO2(I),FNIMJ(I),ENSMJ(I),WDOMP(I),I=1,7)
FORMAT(' ',F7.2,F15.4,F18.4,F17.3)
CONTINUE
IF(LKL.LT.4)GO TO 398
STOP
END
SUBROUTINE PICSS(ANGIN,ANGOUT,RHC,TEMP,BLADIN,PTOCHO,ASPECR,PETHACH
1,VELOT,CHORD,J,K,KLFAR,YT,KTM,TEM,U)
COMMON PI,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
COMMON/AEEA1/TET,TC,TIECLA
DIMENSION B18C(9),B275(9),A3(9),B370(9),B365(9),B360(9),B350(9),B3
140(9),A5(5),B5(5),A15(5),B1540(5),B155C(5),B156C(5),B1570(5),A7(7)
1,B7(7),A811(11),B865(11),B860(11),B855(11),B850(11),B840(11),A8(9)
3,B830(9),PCA(1),PCB(7),YP2A(7),YP2B(7),TFX(7),TFY(7),ANG80(7),ANG7
40(6),ANG30(7),YP1A(6),YPNA(7),SI75(7),ANG47(4),SIDA(4),AI(9),TEM(K
5TM),U(KTM),BI70(9),BI65(9),BI60(9),BI55(9),BI50(9),BI40(9)
DATA A3/.3,.4,.5,.6,.7,.8,.9,1.,1.1/

```

PGM20217  
PGM20218  
PGM20219  
PGM20220  
PGM20221  
PGM20222  
PGM20223  
PGM20224  
PGM20225  
PGM20226  
PGM20227  
PGM20228  
PGM20229  
PGM20230  
PGM20231  
PGM20232  
PGM20233  
PGM20234  
PGM20235  
PGM20236  
PGM20237  
PGM20238  
PGM20239  
PGM20240  
PGM20241  
PGM20242  
PGM20243  
PGM20244  
PGM20245  
PGM20246  
PGM20247  
PGM20248  
PGM20249  
PGM20250  
PGM20251  
PGM20252



DATA B180/.07149,.06468,.0594,.05882,.06128,.06553,.07285,.0834,.0  
 19532/  
 DATA B275/.06979,.05872,.05021,.04766,.04817,.05140,.05651,.06383,  
 1.074/  
 DATA B370/.06894,.05532,.04511,.03966,.03830,.04,.0434,.04936,.057  
 187/  
 DATA B365/.06809,.05498,.04255,.03574,.03098,.02928,.03115,.03643,  
 1.04494/  
 DATA B360/.06639,.05362,.04068,.03149,.02689,.02417,.02451,.02809,  
 1.03404/  
 DATA B350/.06519,.05055,.03949,.0303,.02502,.02264,.02128,.02077,.  
 102247/  
 DATA B340/.06468,.0497,.03745,.02911,.02383,.02043,.01855,.01804,.  
 101872/  
 DATA A1/.3,.4,.5,.6,.7,.8,.9,1,1.1/  
 DATA B170/.16113,.14186,.13640,.13981,.14868,.16190,.17596,.19096,  
 1.2077/  
 DATA B165/.15613,.13026,.11967,.11628,.12089,.12907,.13981,.15192,  
 1.1668/  
 DATA B160/.14334,.11793,.10571,.10196,.10401,.10827,.11475,.12361,  
 1.1328/  
 DATA B155/.14749,.11509,.09787,.08747,.08491,.08730,.09378,.10315,  
 1.1166/  
 DATA B150/.14408,.11168,.09122,.07843,.0731,.07502,.0798,.08791,.0  
 1987/  
 DATA B140/.13981,.10656,.0861,.07315,.06581,.0653,.0682,.07398,.08  
 1/  
 DATA A5/.4,.5,.6,.7,.8/  
 DATA B5/8.0576,6.8921,4.6043,1.4532,-2.3022/  
 DATA A15/.8,.85,.9,.95,1.1/  
 DATA B1540/-2.3022,-3.3813,-4.6763,-5.5396,-6.4029/  
 DATA B1550/-2.3044,-4.4604,-6.6187,-8.9928,-11.3669/  
 DATA B1560/-2.3066,-4.7626,-8.0432,-11.3669,-14.777/  
 DATA B1570/-2.3088,-5.3237,-8.6403,-14.1726,-19.1367/  
 DATA A7/-.9,-.6,-.4,-.2,0,.2,.4/  
 DATA B7/11.218,25.671,33.437,40.988,44.223,43.576,40.34/

PGM20253  
 PGM20254  
 PGM20255  
 PGM20256  
 PGM20257  
 PGM20258  
 PGM20259  
 PGM20260  
 PGM20261  
 PGM20262  
 PGM20263  
 PGM20264  
 PGM20265  
 PGM20266  
 PGM20267  
 PGM20268  
 PGM20269  
 PGM20270  
 PGM20271  
 PGM20272  
 PGM20273  
 PGM20274  
 PGM20275  
 PGM20276  
 PGM20277  
 PGM20278  
 PGM20279  
 PGM20280  
 PGM20281  
 PGM20282  
 PGM20283  
 PGM20284  
 PGM20285  
 PGM20286  
 PGM20287  
 PGM20288









```

IF (ABS(ANGOUT).GT.80.).OF. (ABS(ANGOUT).LT.30.)) GO TO 25
IF (ES(BLADIN).LT.0.5) GO TO 6
IF (ABS(ANGOUT).GT.70.) GO TO 2
IF (ABS(ANGOUT).LT.40.) GO TO 4
CALL FIG(9,AI,BI40,PTOCHO,YPIA(1))
CALL FIG(9,AI,BI50,PTOCHO,YPIA(2))
CALL FIG(9,AI,BI55,PTOCHO,YPIA(3))
CALL FIG(9,AI,BI60,PTOCHO,YPIA(4))
CALL FIG(9,AI,BI65,PTOCHO,YPIA(5))
CALL FIG(9,AI,BI70,PTOCHO,YPIA(6))
CALL FIG(6,ANG70,YPIA,A2,YPI)
GO TO 6

```

```

2 WRITE(KK,3)J,K
3 FORMAT(' ','THE ANGLE OUT WAS GREATER THAN 70 DEG AND THE DATA WAS
1 FROM 70 DEG FOR POW',I3,'STAGE',I3)
GO TO 6

```

```

4 WRITE(KK,5)J,K
5 FORMAT(' ','THE ANGLE OUT WAS LFSS
1 FROM 40 DEG FOR POW',I3,'STAGE',I3)
CALL FIG(9,AI,BI40,PTOCHO,YPI)
IF (ABS(ANGOUT).LT.40.) GO TO 7
CALL FIG(9,A3,B340,PTOCHO,YPNA(1))
CALL FIG(9,A3,B350,PTOCHO,YPNA(2))
CALL FIG(9,A3,B360,PTOCHO,YPNA(3))
CALL FIG(9,A3,B365,PTOCHO,YPNA(4))
CALL FIG(9,A3,B370,PTOCHO,YPNA(5))
CALL FIG(9,A3,B375,PTOCHO,YPNA(6))
CALL FIG(9,A3,B380,PTOCHO,YPNA(7))
CALL FIG(7,ANG80,YPNA,A2,YPN)
GO TO 8

```

```

7 WRITE(KK,5)J,K
8 CALL FIG(9,A3,B340,PTOCHO,YPN)
YP=(YPN+(BLADIN/ANGOUT)*2*(YPI-YPN))* (TC/.2) ** (-BLADIN/ANGOUT)
IF (ABS(CINCT).LT.0.5) GO TO 13
IF (PTOCHO.GT.1.) GO TO 35
CALL FIG(7,PCA,PCB,PTOCHO,C75)

```

PGM20325  
 PGM20326  
 PGM20327  
 PGM20328  
 PGM20329  
 PGM20330  
 PGM20331  
 PGM20332  
 PGM20333  
 PGM20334  
 PGM20335  
 PGM20336  
 PGM20337  
 PGM20338  
 PGM20339  
 PGM20340  
 PGM20341  
 PGM20342  
 PGM20343  
 PGM20344  
 PGM20345  
 PGM20346  
 PGM20347  
 PGM20348  
 PGM20349  
 PGM20350  
 PGM20351  
 PGM20352  
 PGM20353  
 PGM20354  
 PGM20355  
 PGM20356  
 PGM20357  
 PGM20358  
 PGM20359  
 PGM20360





```

35 GO TO 34
34 C75=PCB(7)-.4*(PTOCHO-1.)
A275=A2/C75
AB=BLADIN/A275
IF(J.EQ.2)AB=-BLADIN/A275
IF((AB.LT.-.9).OR.(AB.GT.1))GO TO 25
IF((A275.GT.65.)AND.(AB.GT.0.4))GO TO 9
IF((A275.LT.40.)AND.(AB.GT.0.6))GO TO 11
CALL FIG(9,A8,B830,AB,SI75(1))
CALL FIG(11,A811,B840,AB,SI75(2))
CALL FIG(11,A811,B850,AB,SI75(3))
CALL FIG(11,A811,B855,AB,SI75(4))
CALL FIG(11,A811,B860,AB,SI75(5))
CALL FIG(11,A811,B865,AB,SI75(6))
CALL FIG(7,A7,B7,AB,SI75(7))
CALL FIG(7,ANG30,SI75,A275,SI751)
GO TO 12
9 WRITE(KK,10)J,K
10 FORMAT(' ','THE ANGLE OUT WAS GREATER THAN 65 DEG AND THE DATA WAS
1 FROM 65 DEG FOR ROW',I3,'STAGE',I3)
CALL FIG(11,A811,B865,AB,SI751)
GO TO 12
11 WRITE(KK,5)J,K
CALL FIG(9,A8,B830,AB,SI751)
12 IF(PTCCHO.GT.0.8)GO TO 13
CALL FIG(5,A5,B5,PTCCHO,DELTST)
GO TO 16
13 IF(PTOCHO.GT.1.)GO TO 14
IF((A2.LT.40.).OR.(A2.GT.70.))GO TO 14
CALL FIG(5,A15,B1540,PTOCHO,SIDA(1))
CALL FIG(5,A15,B1550,PTOCHO,SIDA(2))
CALL FIG(5,A15,B1560,PTOCHO,SIDA(3))
CALL FIG(5,A15,B1570,PTCCHO,SIDA(4))
CALL FIG(4,ANG47,SIDA,A2,DELTST)
GO TO 16
14 WRITE(KK,15)J,K
PGM20361
PGM20362
PGM20363
PGM20364
PGM20365
PGM20366
PGM20367
PGM20368
PGM20369
PGM20370
PGM20371
PGM20372
PGM20373
PGM20374
PGM20375
PGM20376
PGM20377
PGM20378
PGM20379
PGM20380
PGM20381
PGM20382
PGM20383
PGM20384
PGM20385
PGM20386
PGM20387
PGM20388
PGM20389
PGM20390
PGM20391
PGM20392
PGM20393
PGM20394
PGM20395
PGM20396
142

```



```

15  FORMAT(' ', THE P/C RATIO WAS GREATER THAN DATA FOR OFF INCIDENCE
CALCULATIONS, A VALUE FOR DELTA INCIDENCE IS TAKEN FOR P/C=1./' ',
21  FOR ROW, I3, ' STAGE', I3)
IF(A2.GT.70.)DELTSI=B1570(5)
IF(A2.LT.70.)DELTSI=B1570(5)
IF(A2.LT.60.)DELTSI=B1560(5)
IF(A2.LT.50.)DELTSI=B1550(5)
IF(A2.LT.40.)DELTSI=B1540(5)
SI=DELTSI+SI751
SIR=CINCI/SI
IF((SIR.GT.1.5).OR.(SIR.LT.-4.))GO TO 17
CALL FIG(7, YP2A, YP2B, SIR, YPC)
YP=YPC*YPC
GO TO 18
YP=YPC*10.
WRITE(KK, 29) J, K
29  FORMAT(' ', THE DATA LIMITS WERE EXCEEDED IN ROW, I3, ' STAGE', I3)
18  A6=ANGIN*PI/180.
IF(J.EQ.1)A6=-A6
A4=ANGOUT*PI/180.
IF(J.EQ.2)A4=-A4
ANGMEN=ATAN(.5*(TAN(A4)-TAN(A6)))
CLSC=2.*(TAN(A6)+TAN(A4))*COS(ANGMEN)
ZETA=CLSC**2*(COS(A4)**2/COS(ANGMEN)**2)
YS=.0334*(COS(A4)/COS(B1))*ZETA/ASPEC
IF(KLEAP.EQ.1)B=.0
IF(KLEAP.EQ.2)B=.37
IF(KLEAP.EQ.3)B=.47
YK=B*(IIPCL/CHOPD)*.78*ZETA/ASPEC
IF(REMACH.GT.1.)YP=YPC*(1.+60.*(REMACH-1.))**2)
YPS=(YP+YS)*(PE/2.E5)**(-.2)
YT=YPS+YK
CALL FIG(7, TEL, TEY, TET, YPEC)
YT=YT*YDPC
RETURN
24  YT=0.5
25

```



```

26      WRITE(KK,26)J,K
      FORMAT(' ','THE INPUT DATA WAS GREATER THAN LIMITS OF PROGRAM & A
1       VALUE OF Y=.5 WAS ASSIGNED FOR POW',I2,' SIGGE',I3)
      GO TO 24
      END
      SUBROUTINE BLADE(A,B,C,J)
      C CALCULATES BLADE OF GAS ANGLES FOR ROTOR AND STATORS
      DIMENSION D(6),P(6)
      DATA D/24.34,40.,50.,60.,70.,80./
      DATA P/30.,47.717,53.962,62.453,70.943,78.868/
      IF(J.EQ.2)GO TO 3
      IF(B.LI.1.)GO TO 1
      C=A
      GO TO 2
1      CALL FIG(6,D,P,A,C)
      IF(B.LI.0.5)GO TO 2
      C=C-((B-.5)/.5)* (C-A)
      RETURN
2      IF(B.LI.1.)GO TO 5
      A=C
      GO TO 2
5      CALL FIG(6,P,D,C,A)
      IF(B.LI.0.5)GO TO 2
      A=A+((B-.5)/.5)* (C-A)
      GO TO 2
      END
      SUBROUTINE SIG(I,D,P,X,Y)
      DIMENSION D(I),P(I),DD(100),FF(100),A(4),B(4)
      IF((X.GT.D(I)).AND.(D(1).LT.D(I))).OR.((X.LT.D(1)).AND.(D(I).GT.D
1      (1))).OR.((X.LT.D(I)).AND.(D(1).GT.D(I))).OR.((
2      X.GT.D(1)).AND.(D(1).GT.D(I)))GO TO 11
      IF(D(1).GT.D(I))GO TO 9
      IF(J.EQ.4)GO TO 2
      N=I-1
      J=2
      IF(X.GE.D(N))GO TO 1

```

PGM20433  
 PGM20434  
 PGM20435  
 PGM20436  
 PGM20437  
 PGM20438  
 PGM20439  
 PGM20440  
 PGM20441  
 PGM20442  
 PGM20443  
 PGM20444  
 PGM20445  
 PGM20446  
 PGM20447  
 PGM20448  
 PGM20449  
 PGM20450  
 PGM20451  
 PGM20452  
 PGM20453  
 PGM20454  
 PGM20455  
 PGM20456  
 PGM20457  
 PGM20458  
 PGM20459  
 PGM20460  
 PGM20461  
 PGM20462  
 PGM20463  
 PGM20464  
 PGM20465  
 PGM20466  
 PGM20467  
 PGM20468



```

4      IF(X.LE.D(J))GO TO 2
        L=3
        IF(X.LT.D(L))GO TO 3
        L=L+1
        GO TO 4
3      L=L-2
        DO 8 K=1,4
          A(K)=D(L)
          B(K)=F(L)
          L=L+1
        GO TO 7
1      IJ=I-4
        DO 5 II=1,4
          A(II)=D(IJ)
          B(II)=F(IJ)
          IJ=IJ+1
        GO TO 7
5      GO TO 7
2      DO 6 II=1,4
          A(II)=D(II)
          B(II)=F(II)
        CALL BK(A,B,X,Y)
32     RETURN
9      DD(1)=D(I)
        FF(1)=F(I)
        J=I
        DO 15 M=2,I
          J=J-1
          DD(M)=D(J)
          FF(M)=F(J)
          DO 16 M=1,I
            D(M)=DD(M)
            F(M)=FF(M)
          GO TO 17
15     IF((X.GT.D(I)).AND.(D(1).LT.D(I)).OR.((X.LE.D(I)).AND.(D(1).GT.D
1(I))))Y=F(I)
16     IF(Y.NE.F(I))Y=F(1)
31

```

```

PGM20469
PGM20470
PGM20471
PGM20472
PGM20473
PGM20474
PGM20475
PGM20476
PGM20477
PGM20478
PGM20479
PGM20480
PGM20481
PGM20482
PGM20483
PGM20484
PGM20485
PGM20486
PGM20487
PGM20488
PGM20489
PGM20490
PGM20491
PGM20492
PGM20493
PGM20494
PGM20495
PGM20496
PGM20497
PGM20498
PGM20499
PGM20500
PGM20501
PGM20502
PGM20503
PGM20504

```





```

GO TO 22
END
SUBROUTINE BK(Y,Y,XARG,YOUT)
  DIMENSION X(4),Y(4)
  YOUT=0.0
  XSO=XARG**2
  XCU=XSC*XARG
  DO 10 K=1,4
    PI1=1.
    PI2=1.
    SUM1=0.0
    SUM2=0.0
    DO 20 J=1,4
      IF(J.EQ.K) GO TO 20
      PI1=PI1*(X(K)-X(J))
      SUM1=SUM1+X(J)
      PI2=PI2*X(J)
      DO 99 I=2,4
        IF(I.EQ.K) GO TO 99
        IF(I.LE.J) GO TO 99
        SUM2=SUM2+X(J)*X(I)
      CONTINUE
      YOUT=YOUT+1./PI1*(XCU-SUM1*XSO+SUM2*XARG-PI2)*Y(K)
      IF((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
1LE.X(2)).AND.((YOUT.GT.Y(1)).OR.(YOUT.LE.Y(2)))(YOUT=Y(1)-(Y(1)-Y(
12))*XARG-X(1))/(X(2)-X(1))
      IF((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
1LE.X(3)).AND.(XARG.GT.X(2)).AND.((YOUT.GT.Y(2)).OR.(YOUT.LE.Y(3)))(
2)YOUT=Y(2)-(Y(2)-Y(3))*XARG-X(2))/(X(3)-X(2))
      IF((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
1LE.X(4)).AND.(XARG.GT.X(3)).AND.((YOUT.GT.Y(3)).OR.(YOUT.LE.Y(4)))(
2)YOUT=Y(3)-(Y(3)-Y(4))*XARG-X(3))/(X(4)-X(3))
      IF((Y(1).LE.Y(2)).AND.(Y(2).LE.Y(3)).AND.(Y(3).LE.Y(4)).AND.(XARG.
1LE.X(2)).AND.((YOUT.LE.Y(1)).OR.(YOUT.GT.Y(2)))(YOUT=Y(1)-(Y(1)-Y(
12))*XARG-X(1))/(X(2)-X(1))

```

```

99
20
10

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PGM20505
PGM20506
PGM20507
PGM20508
PGM20509
PGM20510
PGM20511
PGM20512
PGM20513
PGM20514
PGM20515
PGM20516
PGM20517
PGM20518
PGM20519
PGM20520
PGM20521
PGM20522
PGM20523
PGM20524
PGM20525
PGM20526
PGM20527
PGM20528
PGM20529
PGM20530
PGM20531
PGM20532
PGM20533
PGM20534
PGM20535
PGM20536
PGM20537
PGM20538
PGM20539
PGM20540

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| PGM  | 20541 | 20542 | 20543 | 20544 | 20545 | 20546 | 20547 | 20548 | 20549 | 20550 | 20551 | 20552 | 20553 | 20554 | 20555 | 20556 | 20557 | 20558 | 20559 | 20560 | 20561 | 20562 | 20563 | 20564 | 20565 | 20566 | 20567 | 20568 | 20569 | 20570 | 20571 | 20572 | 20573 | 20574 | 20575 | 20576 |
|--|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|
| IF((Y(1).LE.Y(2)).AND.(Y(2).LE.Y(3)).AND.(Y(3).LE.Y(4)).AND.(Y(4).LE.X(3)).AND.(Y(5).LE.X(2)).AND.(Y(6).LE.X(1)).AND.(Y(7).LE.Y(2)).AND.(Y(8).LE.Y(3)).AND.(Y(9).LE.Y(4)).AND.(Y(10).LE.Y(5)).AND.(Y(11).LE.Y(6)).AND.(Y(12).LE.Y(7)).AND.(Y(13).LE.Y(8)).AND.(Y(14).LE.Y(9)).AND.(Y(15).LE.Y(10)).AND.(Y(16).LE.Y(11)).AND.(Y(17).LE.Y(12)).AND.(Y(18).LE.Y(13)).AND.(Y(19).LE.Y(14)).AND.(Y(20).LE.Y(15)).AND.(Y(21).LE.Y(16)).AND.(Y(22).LE.Y(17)).AND.(Y(23).LE.Y(18)).AND.(Y(24).LE.Y(19)).AND.(Y(25).LE.Y(20)).AND.(Y(26).LE.Y(21)).AND.(Y(27).LE.Y(22)).AND.(Y(28).LE.Y(23)).AND.(Y(29).LE.Y(24)).AND.(Y(30).LE.Y(25)).AND.(Y(31).LE.Y(26)).AND.(Y(32).LE.Y(27)).AND.(Y(33).LE.Y(28)).AND.(Y(34).LE.Y(29)).AND.(Y(35).LE.Y(30)).AND.(Y(36).LE.Y(31)).AND.(Y(37).LE.Y(32)).AND.(Y(38).LE.Y(33)).AND.(Y(39).LE.Y(34)).AND.(Y(40).LE.Y(35)).AND.(Y(41).LE.Y(36)).AND.(Y(42).LE.Y(37)).AND.(Y(43).LE.Y(38)).AND.(Y(44).LE.Y(39)).AND.(Y(45).LE.Y(40)).AND.(Y(46).LE.Y(41)).AND.(Y(47).LE.Y(42)).AND.(Y(48).LE.Y(43)).AND.(Y(49).LE.Y(44)).AND.(Y(50).LE.Y(45)).AND.(Y(51).LE.Y(46)).AND.(Y(52).LE.Y(47)).AND.(Y(53).LE.Y(48)).AND.(Y(54).LE.Y(49)).AND.(Y(55).LE.Y(50)).AND.(Y(56).LE.Y(51)).AND.(Y(57).LE.Y(52)).AND.(Y(58).LE.Y(53)).AND.(Y(59).LE.Y(54)).AND.(Y(60).LE.Y(55)).AND.(Y(61).LE.Y(56)).AND.(Y(62).LE.Y(57)).AND.(Y(63).LE.Y(58)).AND.(Y(64).LE.Y(59)).AND.(Y(65).LE.Y(60)).AND.(Y(66).LE.Y(61)).AND.(Y(67).LE.Y(62)).AND.(Y(68).LE.Y(63)).AND.(Y(69).LE.Y(64)).AND.(Y(70).LE.Y(65)).AND.(Y(71).LE.Y(66)).AND.(Y(72).LE.Y(67)).AND.(Y(73).LE.Y(68)).AND.(Y(74).LE.Y(69)).AND.(Y(75).LE.Y(70)).AND.(Y(76).LE.Y(71)).AND.(Y(77).LE.Y(72)).AND.(Y(78).LE.Y(73)).AND.(Y(79).LE.Y(74)).AND.(Y(80).LE.Y(75)).AND.(Y(81).LE.Y(76)).AND.(Y(82).LE.Y(77)).AND.(Y(83).LE.Y(78)).AND.(Y(84).LE.Y(79)).AND.(Y(85).LE.Y(80)).AND.(Y(86).LE.Y(81)).AND.(Y(87).LE.Y(82)).AND.(Y(88).LE.Y(83)).AND.(Y(89).LE.Y(84)).AND.(Y(90).LE.Y(85)).AND.(Y(91).LE.Y(86)).AND.(Y(92).LE.Y(87)).AND.(Y(93).LE.Y(88)).AND.(Y(94).LE.Y(89)).AND.(Y(95).LE.Y(90)).AND.(Y(96).LE.Y(91)).AND.(Y(97).LE.Y(92)).AND.(Y(98).LE.Y(93)).AND.(Y(99).LE.Y(94)).AND.(Y(100).LE.Y(95)).AND.(Y(101).LE.Y(96)).AND.(Y(102).LE.Y(97)).AND.(Y(103).LE.Y(98)).AND.(Y(104).LE.Y(99)).AND.(Y(105).LE.Y(100)).AND.(Y(106).LE.Y(101)).AND.(Y(107).LE.Y(102)).AND.(Y(108).LE.Y(103)).AND.(Y(109).LE.Y(104)).AND.(Y(110).LE.Y(105)).AND.(Y(111).LE.Y(106)).AND.(Y(112).LE.Y(107)).AND.(Y(113).LE.Y(108)).AND.(Y(114).LE.Y(109)).AND.(Y(115).LE.Y(110)).AND.(Y(116).LE.Y(111)).AND.(Y(117).LE.Y(112)).AND.(Y(118).LE.Y(113)).AND.(Y(119).LE.Y(114)).AND.(Y(120).LE.Y(115)).AND.(Y(121).LE.Y(116)).AND.(Y(122).LE.Y(117)).AND.(Y(123).LE.Y(118)).AND.(Y(124).LE.Y(119)).AND.(Y(125).LE.Y(120)).AND.(Y(126).LE.Y(121)).AND.(Y(127).LE.Y(122)).AND.(Y(128).LE.Y(123)).AND.(Y(129).LE.Y(124)).AND.(Y(130).LE.Y(125)).AND.(Y(131).LE.Y(126)).AND.(Y(132).LE.Y(127)).AND.(Y(133).LE.Y(128)).AND.(Y(134).LE.Y(129)).AND.(Y(135).LE.Y(130)).AND.(Y(136).LE.Y(131)).AND.(Y(137).LE.Y(132)).AND.(Y(138).LE.Y(133)).AND.(Y(139).LE.Y(134)).AND.(Y(140).LE.Y(135)).AND.(Y(141).LE.Y(136)).AND.(Y(142).LE.Y(137)).AND.(Y(143).LE.Y(138)).AND.(Y(144).LE.Y(139)).AND.(Y(145).LE.Y(140)).AND.(Y(146).LE.Y(141)).AND.(Y(147).LE.Y(142)).AND.(Y(148).LE.Y(143)).AND.(Y(149).LE.Y(144)).AND.(Y(150).LE.Y(145)).AND.(Y(151).LE.Y(146)).AND.(Y(152).LE.Y(147)).AND.(Y(153).LE.Y(148)).AND.(Y(154).LE.Y(149)).AND.(Y(155).LE.Y(150)).AND.(Y(156).LE.Y(151)).AND.(Y(157).LE.Y(152)).AND.(Y(158).LE.Y(153)).AND.(Y(159).LE.Y(154)).AND.(Y(160).LE.Y(155)).AND.(Y(161).LE.Y(156)).AND.(Y(162).LE.Y(157)).AND.(Y(163).LE.Y(158)).AND.(Y(164).LE.Y(159)).AND.(Y(165).LE.Y(160)).AND.(Y(166).LE.Y(161)).AND.(Y(167).LE.Y(162)).AND.(Y(168).LE.Y(163)).AND.(Y(169).LE.Y(164)).AND.(Y(170).LE.Y(165)).AND.(Y(171).LE.Y(166)).AND.(Y(172).LE.Y(167)).AND.(Y(173).LE.Y(168)).AND.(Y(174).LE.Y(169)).AND.(Y(175).LE.Y(170)).AND.(Y(176).LE.Y(171)).AND.(Y(177).LE.Y(172)).AND.(Y(178).LE.Y(173)).AND.(Y(179).LE.Y(174)).AND.(Y(180).LE.Y(175)).AND.(Y(181).LE.Y(176)).AND.(Y(182).LE.Y(177)).AND.(Y(183).LE.Y(178)).AND.(Y(184).LE.Y(179)).AND.(Y(185).LE.Y(180)).AND.(Y(186).LE.Y(181)).AND.(Y(187).LE.Y(182)).AND.(Y(188).LE.Y(183)).AND.(Y(189).LE.Y(184)).AND.(Y(190).LE.Y(185)).AND.(Y(191).LE.Y(186)).AND.(Y(192).LE.Y(187)).AND.(Y(193).LE.Y(188)).AND.(Y(194).LE.Y(189)).AND.(Y(195).LE.Y(190)).AND.(Y(196).LE.Y(191)).AND.(Y(197).LE.Y(192)).AND.(Y(198).LE.Y(193)).AND.(Y(199).LE.Y(194)).AND.(Y(200).LE.Y(195)).AND.(Y(201).LE.Y(196)).AND.(Y(202).LE.Y(197)).AND.(Y(203).LE.Y(198)).AND.(Y(204).LE.Y(199)).AND.(Y(205).LE.Y(200)).AND.(Y(206).LE.Y(201)).AND.(Y(207).LE.Y(202)).AND.(Y(208).LE.Y(203)).AND.(Y(209).LE.Y(204)).AND.(Y(210).LE.Y(205)).AND.(Y(211).LE.Y(206)).AND.(Y(212).LE.Y(207)).AND.(Y(213).LE.Y(208)).AND.(Y(214).LE.Y(209)).AND.(Y(215).LE.Y(210)).AND.(Y(216).LE.Y(211)).AND.(Y(217).LE.Y(212)).AND.(Y(218).LE.Y(213)).AND.(Y(219).LE.Y(214)).AND.(Y(220).LE.Y(215)).AND.(Y(221).LE.Y(216)).AND.(Y(222).LE.Y(217)).AND.(Y(223).LE.Y(218)).AND.(Y(224).LE.Y(219)).AND.(Y(225).LE.Y(220)).AND.(Y(226).LE.Y(221)).AND.(Y(227).LE.Y(222)).AND.(Y(228).LE.Y(223)).AND.(Y(229).LE.Y(224)).AND.(Y(230).LE.Y(225)).AND.(Y(231).LE.Y(226)).AND.(Y(232).LE.Y(227)).AND.(Y(233).LE.Y(228)).AND.(Y(234).LE.Y(229)).AND.(Y(235).LE.Y(230)).AND.(Y(236).LE.Y(231)).AND.(Y(237).LE.Y(232)).AND.(Y(238).LE.Y(233)).AND.(Y(239).LE.Y(234)).AND.(Y(240).LE.Y(235)).AND.(Y(241).LE.Y(236)).AND.(Y(242).LE.Y(237)).AND.(Y(243).LE.Y(238)).AND.(Y(244).LE.Y(239)).AND.(Y(245).LE.Y(240)).AND.(Y(246).LE.Y(241)).AND.(Y(247).LE.Y(242)).AND.(Y(248).LE.Y(243)).AND.(Y(249).LE.Y(244)).AND.(Y(250).LE.Y(245)).AND.(Y(251).LE.Y(246)).AND.(Y(252).LE.Y(247)).AND.(Y(253).LE.Y(248)).AND.(Y(254).LE.Y(249)).AND.(Y(255).LE.Y(250)).AND.(Y(256).LE.Y(251)).AND.(Y(257).LE.Y(252)).AND.(Y(258).LE.Y(253)).AND.(Y(259).LE.Y(254)).AND.(Y(2 |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |       |



|          |          |          |          |          |          |          |          |          |          |          |
|----------|----------|----------|----------|----------|----------|----------|----------|----------|----------|----------|
| 0.99973  | 0.99892  | 0.99769  | 0.99574  | 0.99335  | 0.99040  | 0.98700  | 0.98317  | 0.97879  | 0.97380  | PGM20577 |
| 0.96835  | 0.96260  | 0.95630  | 0.94950  | 0.94200  | 0.93440  | 0.92650  | 0.91800  | 0.90920  | 0.90030  | PGM20578 |
| 0.89080  | 0.88100  | 0.87060  | 0.86000  | 0.84950  | 0.83860  | 0.82720  | 0.81570  | 0.80400  | 0.79220  | PGM20579 |
| 0.78020  | 0.76750  | 0.75550  | 0.74270  | 0.73070  | 0.71800  | 0.70520  | 0.69140  | 0.67910  | 0.66650  | PGM20580 |
| 0.65500  | 0.64080  | 0.62760  | 0.61410  | 0.60250  | 0.58900  | 0.57550  | 0.56500  | 0.55100  | 0.54000  | PGM20581 |
| 0.52800  | 0.51700  | 0.50400  | 0.49100  | 0.48000  | 0.46820  | 0.45650  | 0.44500  | 0.43330  | 0.42250  | PGM20582 |
| 0.96151  | 1.91194  | 2.86589  | 3.82740  | 4.77028  | 5.73284  | 6.67004  | 7.61008  | 8.56787  | 9.50320  | PGM20583 |
| 10.45287 | 11.36668 | 12.31328 | 13.26734 | 14.22884 | 15.14564 | 16.04750 | 16.99411 | 17.90344 | 18.82024 | PGM20584 |
| 19.75192 | 20.63144 | 21.54077 | 22.45010 | 23.37082 | 24.27625 | 25.16321 | 26.05019 | 26.95207 | 27.80922 | PGM20585 |
| 28.65892 | 29.54590 | 30.41052 | 31.20750 | 32.09502 | 32.94472 | 33.80190 | 34.65160 | 35.47834 | 36.29383 | PGM20586 |
| 37.05164 | 37.93861 | 38.76595 | 39.60822 | 40.39828 | 41.14365 | 42.03061 | 42.70889 | 43.41698 | 44.16234 | PGM20587 |
| 44.98222 | 45.69031 | 46.36113 | 47.17357 | 47.85184 | 48.50720 | 49.34256 | 50.02827 | 50.74382 | 51.39220 | PGM20588 |
| 400.600  | 800.100  | 1200.140 | 1600.160 | 1800.200 | 2000.220 | 2400.240 |          |          |          | PGM20589 |
| 100.135  | 166.192  | 218.218  | 242.264  | 284.302  | 320.320  | 338.     |          |          |          | PGM20590 |
|          |          |          |          |          |          |          |          |          |          | PGM20591 |

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Preliminary design of  
a free vortex axial  
flow turbine.

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